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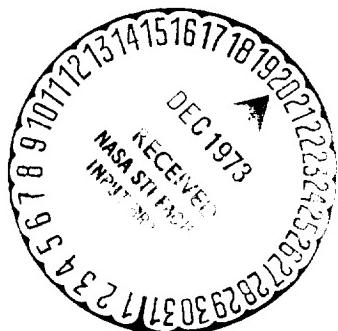
# ADHESIVE-BONDED SCARF AND STEPPED-LAP JOINTS

## TECHNICAL REPORT

by

L. J. HART-SMITH

Prepared under Contract NAS1-11234  
Douglas Aircraft Company  
McDonnell Douglas Corporation  
3855 Lakewood Blvd  
Long Beach, California 90846



January 1973

for

Langley Research Center  
Hampton, Virginia 23366

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

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## ABSTRACT

Continuum mechanics solutions are derived for the static load-carrying capacity of scarf and stepped-lap adhesive-bonded joints. The analyses account for adhesive plasticity and adherend stiffness imbalance and thermal mismatch. The scarf joint solutions include a simple algebraic formula which serves as a close lower bound, within a small fraction of a per cent of the true answer for most practical geometries and materials. The scarf joint solutions are believed to be the first such results ever obtained for dissimilar adherends. Digital computer programs have been developed and, for the stepped-lap joints, the critical adherend and adhesive stresses are computed for each step. The scarf joint solutions exhibit grossly different behavior from that for double-lap joints for long overlaps inasmuch as that the potential bond shear strength continues to increase with indefinitely long overlaps on the scarf joints. The stepped-lap joint solutions exhibit some characteristics of both the scarf and double-lap joints. The stepped-lap computer program handles arbitrary (different) step lengths and thicknesses and the solutions obtained have clarified potentially weak design details and the remedies. Indeed, the program has been used effectively to optimize the joint proportions.

## KEYWORD DESCRIPTORS

Bonded Joints

Adhesive Stresses and Strains

Adherend Stiffness Imbalance

Adherend Thermal Mismatch

Computer Analysis Programs

Scarf Joints

Stepped-Lap Joints

Static Strength

Elastic-Plastic Formulation

Advanced Composite Joints

## FOREWORD

This report was prepared by the Douglas Aircraft Company, McDonnell Douglas Corporation, Long Beach, California under the terms of Contract NAS1-11234. One summary report (NASA CR 2218) and four technical reports (NASA CR 112235, -6, -7, and -8) cover the work, which was performed between November 1971 and January 1973. The program was sponsored by the National Aeronautics and Space Administration's Langley Research Center, Hampton, Virginia. Dr. M. F. Card and Mr. H. G. Bush were the Contracting Agency's Technical Monitors.

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## SYMBOLS

$A_0,..A_n$	=	Coefficients of power series for shear stress distribution in adhesive layer
$a,c$	=	Extents of plastic stress state in adhesive at ends of bonded joint (in.)
$b$	=	Extent of elastic trough in adhesive (in.)
$C,D$	=	Integration constants
$\Delta T_{\text{HERM}}$	=	Non-dimensionalized adherend thermal mismatch coefficient
$d$	=	Length of elastic zone in adhesive bond (in.)
$E$	=	Young's modulus (longitudinal) for adherend (psi)
$ETR$	=	Adherend extensional stiffness ratio
$F_{..}$	=	Adherend allowable (or ultimate) stress (psi)
$G$	=	Adhesive shear modulus for elastic-plastic representation (psi)
$l$	=	Overlap (length of bond) (in.)
$P$	=	Applied direct load on entire joint (lb in. / in.)
$SGNLD$	=	Distinguisher between tensile and compressive shear loads
$T$	=	Direct stress resultants in adherends (lb / in.)
$\Delta T$	=	Temperature change ( $T_{\text{operating}} - T_{\text{cure}}$ )
$t$	=	Thickness of adherend (in.)
$x$	=	Axial (longitudinal) coordinate parallel to direction of load
$\alpha$	=	Coefficient of thermal expansion ( $^{\circ}\text{F}$ )
$\gamma$	=	Adhesive shear strain
$\gamma_e$	=	Elastic adhesive shear strain
$\gamma_p$	=	Plastic adhesive shear strain
$\delta$	=	Axial (longitudinal) displacement of adherend (in.)
$\xi, \xi, x$	=	Non-dimensionalized axial coordinates (different origin and/or sense from $x$ )

$n$  = Thickness of adhesive layer (in.)  
 $\theta$  = Scarf angle (small) ( $^{\circ}$ )  
 $\lambda$  = Exponent of elastic shear stress distribution (in. $^{-1}$ )  
 $\nu$  = Poisson's ratio for adherend(s)  
 $\tau$  = Adhesive shear stress (psi)  
 $\tau_{av}$  = Average adhesive shear stress (psi)  
 $\tau_p$  = Plastic (maximum) adhesive shear stress (psi)  
 $\phi$  =  $x/\ell$  = Non-dimensionalized coordinate

#### SUBSCRIPTS

$a,c$  = Adhesive (cement)  
 $e,p$  = Elastic and plastic values  
 $i,o$  = Inner and outer adherends of symmetric bonded joint  
 $1,2$  = Different adherends at each end of joint  
 $1,2,\dots,n$  = Power series counter

## SUMMARY

It has long been known that bonded scarf joints have a higher efficiency than uniform lap joints and that the latter are limited in strength and unsuitable for joining thicker sections. What has not been well understood until recently is that, in the bonding together of dissimilar adherends in a scarf joint, any adherend stiffness imbalance or thermal mismatch imposes a limitation on the joint efficiency. As a consequence the adhesive layer is not (essentially) uniformly stressed along its length as it is for a scarf joint between identical adherends. One objective of this report is to analyze and quantify these limitations on efficiency of unbalanced scarf joints. In doing so, adhesive plasticity is accounted for by the Douglas elastic-plastic model which has been demonstrated to be effective for uniform lap joints. One dominant characteristic deduced for scarf joints is that for long overlaps, regardless of any adhesive ductility and/or adherend thermal mismatch, the ratio of the average adhesive shear stress to the peak adhesive shear stress is equal to the lower ratio ( $<1$ ) of the adherend extensional stiffnesses. The governing differential equations do not possess an explicit solution in terms of standard functions, so a series solution was employed. Even so, an algebraic expression was deduced for a lower bound which proved to be so close to the more precise solutions that it could be employed directly for practically all realistic joint proportions. Severe adverse effects of adherend thermal mismatch are confined to a specific overlap range. The effects decrease asymptotically to zero for very short or very long overlaps.

Stepped-lap joints represent a cross between scarf joints and uniform lap joints. The stepped-lap joint overcomes the upper limit on joint strength of uniform lap joints but retains the severe adhesive strain concentration at the end of each step. One advantage of stepped-lap joints over scarf joints is that the alignment and fit is far less critical when there are joints on more than a single interface. Another is that it is more suitable for boron-epoxy laminates than is a scarf joint because of the thick brittle filaments. This is particularly important for the titanium edge members frequently used in conjunction with boron-epoxy panels. Because the graphite fibers are so much thinner and more flexible than boron filaments, the former can take advantage of the higher efficiency of the scarf joint.

Digital computer FORTRAN IV programs are included for the iterative solutions necessary for these problems. The scarf joint solutions are in terms of non-dimensionalized parameters. The stepped-lap joint program is dimensional and permits each step to be varied independently so as to be able to identify and improve the most critical detail(s) of the joint. One key factor in the design of stepped-lap joints is that the bond load transfer is concentrated at the end of the joint from which the softer (less stiff) adherend extends. Consequently, it is necessary to restrict the length of the end step of the stiffer adherend to prevent it from being overloaded. Another characteristic of stepped-lap joints identified by the analyses is that the end three steps of the more critical end dominate the internal load distribution and effectively determine the load capacity. The steps at the less critical end are found to have practically no effect on the load capacity.

## 1. INTRODUCTION

It is generally recognized that, in the bonding together of thick sections, the use of either scarf or stepped-lap joints is mandatory if an acceptable structural efficiency is to be realized. References (1) and (2) explain how, for uniform lap joints, the maximum possible joint efficiency decreases with increasing thickness (extensional stiffness) of the members being bonded together. The objective of this report is to apply the elastic-plastic adhesive analysis techniques developed in References (1) and (2) to the scarf and stepped-lap joints. The approach used remains that of continuum mechanics rather than finite elements. The governing differential equations were relatively straightforward to set up but, in most cases, specific closed-form solutions could not be derived. Severe numerical accuracy problems had to be overcome in developing the FORTRAN IV digital computer programs employed and this phase of the work represented by far the bulk of the investigation. The computer programs are listed in the Appendices and representative non-dimensionalized solutions are illustrated to show the effect of the governing scarf joint parameters. Specific solutions are presented for stepped-lap joints.

This scarf joint analysis is concerned with the non-uniform adhesive shear stresses necessarily associated with the bonding together of dissimilar adherends. It is well-known that the stresses are uniform if the adherends are identical. It has only recently begun to be appreciated that the adhesive shear stresses are markedly non-uniform if the adherends are dissimilar. Indeed, the literature contains very few references to this problem. The mechanism whereby these non-uniform stresses are developed is illustrated in Figure 1 for the case of thermal mismatch between stiffness-balanced adherends. The first publication on scarf joints between dissimilar adherends appears to be that of Lubkin [Reference (3)] who, in 1957 sought the particular scarf angle associated with uniform adhesive stress for a particular ratio of adherend elastic moduli. He omitted consideration of any adherend thermal dissimilarity. Unfortunately the predictions of his equation [10] are such as to indicate the appropriate scarf angle  $\theta$  is so great (typically in excess of 45 degrees) as to be of no practical interest for bonding aerospace materials together. For realistic adhesives and adherend materials, the scarf angle should be restricted to less than 4 degrees in order for the potential bond

strength to exceed the adherend strength(s). Working independently, in 1971, the present author [Reference (4)] and Erdogan and Ratwani [Reference (5)] demonstrated by calculation the non-uniform adhesive shear stress associated with scarf joints between dissimilar adherends. The former work was based on a perfectly-plastic adhesive analysis, while the latter derived from a linearly-elastic formulation. Consequently neither afforded a complete solution but both demonstrated clearly that the adhesive load transfer is concentrated at that end of the joint from which the softer adherend extends. The present solution utilizes an elastic-plastic adhesive model with linearly elastic adherends and accounts for adherend stiffness and thermal imbalances. Eccentricities in the load path are excluded and, in keeping with common design practice, the scarf angle is considered to be so small that adhesive tension (or compression) stresses may be neglected in comparison with the shear stresses.

In 1968, an elastic finite-element analysis of scarf joints was performed by Richards [see Reference (6)]. Boron/epoxy-to-boron/epoxy and boron/epoxy-to-aluminum joints were analyzed. Thermal effects were neglected. In the former case, relatively small (<4%) stress concentrations were identified in the vicinity of the ends of the scarf. Their existence had not been demonstrated prior to that investigation. In the latter case a markedly non-uniform stress distribution was deduced, with significantly more load being transferred to and from the 0° plies in the laminate than occurred with the ± 45° plies. This is to be expected in view of the much lower modulus of the cross plies.

While the mathematical complexity of equations governing the scarf joint has restricted the number of solutions obtained, a number of investigations of the stepped-lap adhesive-bonded joint have been performed. Finite-element elastic solutions are reported in References (5) to (9) but none of these include any thermal mismatch effects. Reference (10) included adhesive and adherend non-linear behavior in the analysis but, for the stepped-lap joint, encountered convergence difficulties at high load levels. Grimes, Calcote, Wah, et al [Reference (10)] also performed non-linear iterative theoretical analyses of double-lap, single-lap and stepped-lap joints which they compared with their discrete element analyses, showing good agreement for the first two. They also formulated the scarf joint equations (see their Appendix A) in greater detail than is done here, but were unable to solve them. Corvelli and Saleme

[Reference (11)] developed analysis techniques for bonded joints which included analytical solutions for stepped-lap joints, but in a less comprehensive form than presented here.

Past attempts to include non-linear adhesive behavior in the analytical solutions have centered around the Ramberg-Osgood representation which has a smooth continuous characteristic. This has precluded the derivation of any explicit closed-form solutions. The present author had earlier derived such solutions for double- and single-lap adhesive-bonded joints using an elastic-plastic adhesive formulation [see References (12) and (13)]. These showed that the adhesive shear strain energy per unit bond area was the necessary and sufficient adhesive characteristic governing the potential bond shear strength. The precise shape of the stress-strain curve appeared to be unimportant. This belief was further reinforced in Reference (1) by the derivation of precisely the same potential bond shear-strength for any arbitrary bi-elastic adhesive characteristic having the same strain energy and failure stress and strain. In addition, the author's elastic-plastic solution was in good agreement with the discrete element solutions by Teodosiadis [Reference (14)], who represented the adhesive and interlaminar shear characteristics by six straight segments. The success of this elastic-plastic adhesive approach in these simpler problems led to the decision to apply the same techniques to the scarf and stepped-lap joints in this report.

The adhesive-bonded stepped-lap joint is of practical interest principally because of extensive use in the bonding of boron-epoxy to titanium edge members. The boron filaments are too thick (0.005 inch), and too hard to machine, to be as suitable for the more efficient scarf joints as the very thin graphite fibers are. In practice the stepped-lap joint contains a large number of small steps and closely approximates the behavior of the equivalent scarf joint. The only difference is marked for very brittle (high-temperature) adhesives and is the adhesive shear stress (and strain) concentrations at the ends of each step, particularly at the outermost steps. It transpired that peel stresses imposed more severe limitations for thick double- and single-lap joints than did the adhesive shear stresses [see References (1) and (2)]. In actual design practice for scarf and stepped-lap joints, the slope is small and the end step is

invariably thin so there is no way for severe peel stresses to develop. For any unusual stepped-lap joint, with a thick outer end step, the analysis in Reference (1) can be employed to assess any potential peel problem.

This report considers in turn elastic and elastic-plastic analyses of scarf and stepped-lap joints and discusses parametric effects and design procedures. The digital computer programs prepared from the analyses are recorded in the Appendices, along with brief instructions for their use.

## 2. ELASTIC ANALYSIS OF SCARF JOINTS

Figure 2 depicts the geometry and nomenclature for the analysis of a non-eccentric bonded scarf joint. The diagram serves for both the elastic and elastic-plastic solutions. In the former case, the plastic adhesive zones should be considered removed. That is, set  $a = c \equiv 0$  and  $b = l$ . The scarf angle  $\theta$  is considered so small that  $\cos\theta \approx 1$  and  $\theta \approx 0$ . In other words, the effect of adhesive peel stresses is omitted from consideration. This is quite legitimate for the small scarf angles associated with practical aerospace materials.

The conditions of horizontal equilibrium for a differential element  $dx$  within the joint are

$$\frac{dT_1}{dx} + \tau = 0 , \quad \frac{dT_2}{dx} - \tau = 0 . \quad (1)$$

The stress-strain relations for the adherend materials, accounting for thermo-elastic effects, yield

$$\frac{d\delta_1}{dx} = \frac{T_1}{(Et)_1} + \alpha_1 \Delta T , \quad \frac{d\delta_2}{dx} = \frac{T_2}{(Et)_2} + \alpha_2 \Delta T , \quad (2)$$

in which the adherend thicknesses, as a function of the axial coordinate  $x$  are

$$(Et)_1 = E_1 t_1 \left(1 - \frac{x}{l}\right) , \quad (Et)_2 = E_2 t_2 \left(\frac{x}{l}\right) . \quad (3)$$

The adhesive shear strain is taken to be uniform across the thickness of the bond. That is

$$\gamma = (\delta_2 - \delta_1)/n . \quad (4)$$

The elastic adhesive shear stress follows as

$$\tau = G\gamma = G(\delta_2 - \delta_1)/n . \quad (5)$$

In solving these equations it is desirable to non-dimensionalize the solution with respect to the peak adhesive shear stress  $\tau_p$  and the bond overlap. Thus, introducing the non-dimensionalized axial co-ordinate

$$\phi = x/l , \quad (6)$$

a series solution is sought, having the form

$$\frac{\tau}{\tau_p} = \sum_{n=1}^{\infty} A_n \phi^{n-1} . \quad (7)$$

We define the adherend 1 end of the joint as critical so that

$$A_1 \equiv 1 , \quad (8)$$

if necessary by interchange of the identifying subscripts 1 and 2. While a single non-linear differential equation has been derived from the equations above, it cannot be solved directly. This is why a series solution is employed here and, in this case, it is more straightforward to work in terms of the equations above than the derivative governing equation.

The solution proceeds from equation (7). Substitution into equation (1) yields, for the adherend forces per unit width,

$$T_1 = \tau_{av} \ell - \tau_p \ell \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^n , \quad T_2 = \tau_p \ell \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^n . \quad (9)$$

Now equation (5) is differentiated.

$$\frac{d(\tau / \tau_p)}{d\phi} = \frac{G}{\tau_p \eta} \left[ \frac{d\delta_2}{d\phi} - \frac{d\delta_1}{d\phi} \right] . \quad (10)$$

Substitution of the series (7) and (9), with the aid of equations (2), leads to the solution

$$\begin{aligned} \sum_{n=1}^{\infty} (n-1) A_n \phi^{(n-2)} &= \frac{G \ell}{\tau_p^n} \left\{ (\alpha_2 - \alpha_1) \Delta T + \frac{\tau_p \ell}{E_2 t_2} \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^{(n-1)} \right. \\ &\quad \left. - \frac{\tau_{av} \ell}{E_1 t_1 (1-\phi)} + \frac{\tau_p \ell}{E_1 t_1 (1-\phi)} \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^n \right\} . \quad (11) \end{aligned}$$

Multiplication throughout by  $(1 - \phi)$  converts the equation directly into a form suitable for solution by recurrence relations.

$$(1 - \phi) \sum_{n=1}^{\infty} (n-1) A_n \phi^{(n-2)} = \frac{G\ell}{\tau_p^n} (\alpha_2 - \alpha_1) \Delta T (1 - \phi) - \frac{G\ell^2 \tau_{av}}{n E_1 t_1 \tau_p} + \frac{G\ell^2}{n} \left[ \frac{(1 - \phi)}{E_2 t_2} \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^{(n-1)} + \frac{1}{E_1 t_1} \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^n \right] . \quad (12)$$

In order to give the solution the greatest coverage with the minimum number of independent variables, certain non-dimensional parameters are introduced. The non-dimensionalized overlap is given by the square root of

$$(\lambda\ell)^2 = \frac{G\ell^2}{n} \left[ \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right] = \frac{\tau_p \ell^2}{n \gamma_e} \left[ \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right] , \quad (13)$$

the non-dimensionalized thermal mismatch term is

$$CTHERM(1) = \frac{\lambda(\alpha_2 - \alpha_1) \Delta T}{\tau_p \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right)} , \quad CTHERM(2) = -CTHERM(1) , \quad (14)$$

and the adherend stiffness ratio is

$$ETR(1) = E_1 t_1 / E_2 t_2 , \quad ETR(2) = E_2 t_2 / E_1 t_1 . \quad (15)$$

It is interesting to note that precisely the same variables govern the double-lap joint [see Reference (1)]. Equation (12) then becomes

$$\sum_{n=1}^{\infty} [(n-1) A_n - (n-2) A_{n-1}] \phi^{(n-2)} = (\lambda\ell) \times CTHERM(1) \times (1 - \phi) - \frac{(\lambda\ell)^2}{[1 + ETR(1)]} \times \frac{\tau_{av}}{\tau_p} + \frac{(\lambda\ell)^2 ETR(1)}{[1 + ETR(1)]} \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^{(n-1)} + \frac{(\lambda\ell)^2 [1 - ETR(1)]}{[1 + ETR(1)]} \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^n . \quad (16)$$

By rearranging the limits of the series it follows that

$$\sum_{n=1}^{\infty} [(n+1)A_{n+2} - nA_{n+1}] \phi^n = (\lambda\ell) \times \text{CTHERM}(1) \times (1 - \phi) - \frac{(\lambda\ell)^2}{[1 + \text{ETR}(1)]} \times \frac{\tau_{av}}{\tau_p} \\ + \frac{(\lambda\ell)^2 \text{ETR}(1)}{[1 + \text{ETR}(1)]} \sum_{n=1}^{\infty} \frac{A_{n+1}}{n-1} \phi^n + \frac{(\lambda\ell)^2 [1 - \text{ETR}(1)]}{[1 + \text{ETR}(1)]} \sum_{n=1}^{\infty} \frac{A_n}{n} \phi^n . \quad (17)$$

For large values of  $n$ , on setting to zero the coefficient of the term  $\phi^{n-2}$ , the recurrence relation is deduced as

$$A_n = \left\{ (n-2)A_{n-1} + (\lambda\ell)^2 \left[ \left( \frac{\text{ETR}(1)}{1 + \text{ETR}(1)} \right) A_{n-1} + \left( \frac{1 - \text{ETR}(1)}{1 + \text{ETR}(1)} \right) A_{n-2} \right] \right\} / (n-1) . \quad (18)$$

It remains now to establish the initial conditions by examining the coefficients of the  $\phi^0$  and  $\phi^1$  terms. From the coefficient of  $\phi^0$ ,

$$A_2 = (\lambda\ell) \text{ CTHERM}(1) - (\lambda\ell)^2 \left[ \frac{1}{1 + \text{ETR}(1)} \right] \tau_{av} + (\lambda\ell)^2 \left[ \frac{\text{ETR}(1)}{1 + \text{ETR}(1)} \right] A_1 \quad (19)$$

while, from the coefficient of  $\phi^1$ ,

$$2A_3 - A_2 = -(\lambda\ell) \text{ CTHERM}(1) + \frac{(\lambda\ell)^2}{[1 + \text{ETR}(1)]} \left\{ [1 - \text{ETR}(1)] A_1 + \text{ETR}(1) \frac{A_2}{2} \right\} . \quad (20)$$

It follows from equation (19) that, quite generally, for long overlaps (large values of  $\lambda\ell$ ),

$$\frac{\tau_{av}}{\tau_p} \rightarrow \text{ETR}(1) \leq 1 . \quad (\text{Interchange 1 and 2 if necessary.}) \quad (21)$$

This surprisingly simple result proves to dominate the entire behavior of bonded scarf joints, even for elastic-plastic adhesives. This equation demonstrates conclusively the importance of maintaining adherend stiffness balance whenever possible. When this is maintained, in the absence of any thermal mismatch, the adhesive is essentially uniformly stressed throughout the entire overlap of any length. The only minor exception is the local end effect identified by Richards in Reference (6).

Returning now to the solution, in terms of equations (18) to (20), it follows by integrating equation (7) that

$$\frac{\tau_{av}}{\tau_p} = \sum_{n=1}^{\infty} \frac{A_n}{n} . \quad (22)$$

In using this series it is necessary to employ two arbitrary constants to satisfy the boundary conditions. The first two are chosen. That is

$$\frac{\tau_{av}}{\tau_p} = A_1 \times \text{SIG}(3) + A_2 \times \text{SIG}(4) \quad (23)$$

and, because of equation (8),

$$\frac{\tau_{av}}{\tau_p} = \text{SIG}(3) + A_2 \times \text{SIG}(4) . \quad (24)$$

The summations  $\text{SIG}(3)$  and  $\text{SIG}(4)$  are the quantities formed by evaluating the coefficients in equation (22) by means of equations (20) and (18) after setting, in turn,

$$A_1 = 1 , \quad A_2 = 0 \quad \text{for } \text{SIG}(3) \quad (25)$$

and

$$A_1 = 0 , \quad A_2 = 1 \quad \text{for } \text{SIG}(4) . \quad (26)$$

The solution procedure employed in the FORTRAN IV digital computer program listed in Appendix A1 is as follows. The coefficient  $A_3$  for each set of initial values (25) and (26) is evaluated in terms of equation (20). Then a number of higher order coefficients are evaluated in turn through the recurrence relation (18), the same number being evaluated for  $\text{SIG}(3)$  as for  $\text{SIG}(4)$ . The results of these summations are then substituted into equation (19) which takes on the form

$$A_2 \left\{ 1 + \frac{(\lambda \ell)^2}{[1 + \text{ETR}(1)]} \text{SIG}(4) \right\} = (\lambda \ell) \text{CTHERM}(1) + \frac{(\lambda \ell)^2 \text{ETR}(1)}{[1 + \text{ETR}(1)]} - \frac{(\lambda \ell)^2}{[1 + \text{ETR}(1)]} \text{SIG}(3) . \quad (27)$$

The unknown  $A_2$  is then to be evaluated and substituted into equation (19) re-arranged in the form

$$\frac{\tau_{av}}{\tau_p} = ETR(1) + \frac{[1 + ETR(1)] CTHERM(1)}{(\lambda\ell)} - \frac{[1 + ETR(1)]}{(\lambda\ell)^2} A_2 . \quad (28)$$

This equation establishes the potential bond shear strength.

The detailed discussion of parametric effects is presented in Section 5 but certain features of the mathematics of the numerical solution merit elaboration at this stage. The most important feature is the decision to evaluate the terms  $A_n/n$  of the average stress series (22) directly rather than the quantities  $A_n$  of the series (7). To do so, equation (18) is re-organized to the form

$$\left(\frac{A_n}{n}\right) = \frac{(n-1)(n-2)\left(\frac{A_{n-1}}{n-1}\right) + (\lambda\ell)^2 \left[ \frac{ETR(1)}{1 + ETR(1)} \left(\frac{A_{n-1}}{n-1}\right) + \frac{1 - ETR(1)}{1 + ETR(1)} \left(\frac{A_{n-2}}{n-2}\right) \right]}{[n(n-1)]} . \quad (29)$$

The reason for this is the factor  $(\lambda\ell)^2$  in equations (29) and (18). Because of this, for long overlaps, a much higher value of  $n$  is needed to reach negligible values of  $A_n$  from equation (18) than to reach negligible values of  $A_n/n$  from equation (29). Indeed, even with the use of equation (29) rather than equation (18) it remained impossible to compute reliable internal stress distributions for long overlap joints, even with as many as 50 terms of the shear stress series because of overflow in the computer. Such a computation is of little importance, however, since the critical location must be at one end or other of the joint. In spite of this problem, however, equation (27) converges rapidly, usually within the first five successive evaluations (for progressively increasing  $n$ ) of SIG(3) and SIG(4). The program in Appendix A1 used 20 terms. In addition to this, because  $A_2$  is divided by  $(\lambda\ell)^2$  in equation (28), an extremely reliable value of  $\tau_{av}/\tau_p$  can be computed readily. The program identifies the more critical end by the simple expedient of estimating the strength starting from each end of the joint and selecting the lower value. It is obvious that a computation of  $\tau_{av}/\tau_p > 1$  signifies simply that condition (8) was violated. A negative value of  $\tau_{av}/\tau_p$  indicates such severe thermal mismatch between adherends that the joint will break apart prior to application of any mechanical loads.

The computation of joint strength proceeding from the other end of the joint is effected by simply interchanging the subscripts 1 and 2 on all affected quantities. With regard to adherend stiffness imbalance alone, it is always possible to identify from equation (28) that the more critical end (1) is that for which  $ETR(1) \leq 1$ . The possible ambiguity arises as the result of the thermal mismatch terms. Since  $CTHERM(1)$  may be either negative or positive independently of whether  $ETR(1)$  is less than or greater than unity, severe thermal mismatch may nullify or even overpower any stiffness imbalance effects. This possibility is evidently greatest for short overlaps because of the factor ( $\lambda\ell$ ) in the denominator of the thermal term in equation (28). It follows that the critical end of the joint between given adherends may well change as the overlap changes and, indeed, such behavior was predicted by the computer program output.

Equations (1) and (2) have been set up for applied tensile loads in the adherends. In the event that the applied load is compressive, it can be seen with reference to Figure 2 that all quantities except the thermal strain terms will change sign. This implies that, in the absence of any thermal mismatch effects, the same end of the joint is critical for both tensile and compressive adherend loads and that the joint strength is the same. Rather than change the sign of all quantities with the exception of the thermal terms, the program merely changes the signs of  $CTHERM(1)$  and  $CTHERM(2)$  to account for compressive loading rather than tensile loading. It should be noted that, as a consequence, the opposite end of the same joint may be critical for a reversed load and that the strength may not be the same if there is also stiffness imbalance between the adherends. Likewise, just as for double-lap joints, if the thermal mismatch terms nullify any stiffness imbalance effects for one load direction, they must aggravate the stress concentrations for a load in the reverse direction. By analogy with the double-lap joint analyses in Reference (1), the case of in-plane shear loading is covered by the analysis above replacing  $E_1$  and  $E_2$  in equation (2) and those equations based on it by the shear moduli  $G_1$  and  $G_2$  and neglecting the thermal affects which induce bond stresses at right angles to those of concern for mechanical in-plane shear loads except at the sides of the joint. The direct adherend forces  $T_1$  and  $T_2$  are replaced by shear forces  $S_1$  and  $S_2$  per unit length. A more precise representation of thermal effects for in-plane shear loading would necessarily require a two-dimensional

analysis rather than the one-dimensional solution above and the justification for doing so is minimized by the small amount of adhesive plasticity that even the real brittle adhesives exhibit.

### 3. ELASTIC-PLASTIC ANALYSIS OF SCARF JOINTS

The preceding elastic analysis covers essentially the most difficult formula-tive portions of the elastic-plastic scarf joint analysis. New numerical difficulties of major proportions were encountered in the generation of specific answers by the computer program, but the plastic part of the analysis is straightforward. The necessary additional geometry and nomenclature are identified in Figure 2. Equations (1), (2) and (4) continue to apply, with the substitutions

$$dx = \ell d\xi = \ell dx = \ell d\zeta \quad (30)$$

as appropriate. Equation (5) is supplemented by the relation

$$\tau = \tau_p \quad \text{for } 0 \leq \xi \leq a \quad \text{and} \quad 0 \leq \zeta \leq c . \quad (31)$$

The relations (4) for the adherend stiffnesses are replaced by

$$(Et)_1 = E_1 t_1 (1 - \xi) = E_1 t_1 (1 - \frac{a}{\ell} - x) = E_1 t_1 (1 - \frac{a}{\ell} - \frac{b}{\ell} - \zeta) \quad (32)$$

and

$$(Et)_2 = E_2 t_2 \xi = E_2 t_2 \left( \frac{a}{\ell} + x \right) = E_2 t_2 \left( \frac{a}{\ell} + \frac{b}{\ell} + \zeta \right) . \quad (33)$$

In the elastic zone, the location of which has yet to be determined, the same power series solution is sought:

$$\frac{\tau}{\tau_p} = \sum_{n=1}^{\infty} A_n x^{(n-1)} \quad \text{or} \quad \frac{\gamma}{\gamma_e} = \sum_{n=1}^{\infty} A_n x^{(n-1)} , \quad (34)$$

again with  $A_1 = 1$  by definition of adherend 1 as the more highly loaded end of the joint.

In the left adhesive plastic zone of the joint illustrated in Figure 2, the adherend forces per unit width follow from equations (1) and (31) as

$$T_1 = \tau_{av} \ell - \tau_p \ell \xi \quad \text{and} \quad T_2 = \tau_p \ell \xi . \quad (35)$$

Substitution into equation (4), making use of equations (2), yields

$$\gamma = \frac{\delta_2 - \delta_1}{\eta} = \frac{1}{\eta} \left[ (\alpha_2 - \alpha_1) \Delta T \ell \xi + \int_0^\xi \frac{T_2 \ell d\xi}{(Et)_2} - \int_0^\xi \frac{T_1 \ell d\xi}{(Et)_1} \right] , \quad (36)$$

$$\gamma = \frac{1}{\eta} \left[ (\alpha_2 - \alpha_1) \Delta T \ell \xi + \frac{\tau_p \ell^2}{E_2 t_2} \xi + \int_0^\xi \frac{(\tau_p - \tau_{av}) \ell^2}{(E_1 t_1)(1-\xi)} d\xi - \frac{\tau_p \ell^2}{E_1 t_1} \xi \right] + C , \quad (37)$$

$$\gamma = \frac{1}{\eta} (\alpha_2 - \alpha_1) \Delta T \ell \xi + \tau_p \ell^2 \left[ \frac{1}{E_2 t_2} - \frac{1}{E_1 t_1} \right] \xi - \frac{(\tau_p - \tau_{av}) \ell^2}{E_1 t_1} \ln(1-\xi) + C . \quad (38)$$

The appropriate boundary conditions are that

$$\gamma = \gamma_e + \gamma_p \quad \text{at } \xi = 0 \quad (39)$$

and

$$\gamma = \gamma_e \quad \text{at } \xi = a/\ell . \quad (40)$$

Consequently, from equations (39) and (38),

$$C = (\gamma_e + \gamma_p) \quad (41)$$

so that, from equations (40) and (38),

$$\gamma_p = - \frac{1}{\eta} \left[ (\alpha_2 - \alpha_1) \Delta T \ell \left( \frac{a}{\ell} \right) + \tau_p \ell^2 \left( \frac{1}{E_2 t_2} - \frac{1}{E_1 t_1} \right) \left( \frac{a}{\ell} \right) - \frac{(\tau_p - \tau_{av}) \ell^2}{E_1 t_1} \ln \left( 1 - \frac{a}{\ell} \right) \right] . \quad (42)$$

Equation (42) may be non-dimensionalized by use of the quantities in equations (13) to (15). It then adopts the form

$$\left( \frac{\gamma_p}{\gamma_e} \right) = - (\lambda \ell) \text{CTHERM}(1) \left( \frac{a}{\ell} \right) + (\lambda \ell)^2 \left( \frac{a}{\ell} \right) \frac{1 - \text{ETR}(1)}{1 + \text{ETR}(1)} + (\lambda \ell)^2 \frac{[1 - (\tau_{av}/\tau_p)]}{[1 + \text{ETR}(1)]} \ln \left( 1 - \frac{a}{\ell} \right) . \quad (43)$$

In solving for the joint strength it is necessary to maintain continuity at the transition ( $\xi = a/\ell$ ) from plastic to elastic adhesive behavior. The continuity of adherend stresses requires that there be no change in  $d\gamma/dx$ .

From equations (4) and (2)

$$\frac{d\gamma}{dx} = \frac{1}{\eta} \left[ (\alpha_2 - \alpha_1) \Delta T + \frac{T_2}{(Et)_2} - \frac{T_1}{(Et)_1} \right] \quad (44)$$

or, in non-dimensionalized form, for the plastic side of the transition

$$\frac{d(\gamma/\gamma_e)}{d\xi} \Big|_{\xi=a/\ell} = (\lambda\ell) CTHERM(1) - (\lambda\ell)^2 \frac{1 - ETR(1)}{1 + ETR(1)} + \frac{(\lambda\ell)^2 [1 - (\tau_{av}/\tau_p)]}{[1 + ETR(1)][1 - (a/\ell)]} . \quad (45)$$

For the elastic side, equation (34) requires that

$$\frac{d(\gamma/\gamma_e)}{dx} \Big|_{x=0} = A_2 . \quad (46)$$

Since  $A_1 \equiv 1$ , the elastic stress distribution can now be evaluated by a recurrence formula, just as in Section 2.

Under certain combinations of stiffness and thermal mismatch between adherends there will be no second plastic adhesive shear stress zone at the far end of the joint while under others there will be. In the former case, the evaluation of the elastic adhesive shear stress at  $x = 1 - (a/\ell)$  by means of the series (34) will lead to a result  $\tau_{end}/\tau_p \leq 1$ . A value of this ratio greater than unity indicates a need for evaluating the affects of the presence of a second plastic adhesive zone, at the far end of the joint. Referring again to Figure 2, the adherend forces per unit width are evaluated through equations (1) and (31) as

$$T_1 = \tau_p \ell \left( \frac{c}{\ell} - \zeta \right) , \quad T_2 = \tau_{av} \ell - \tau_p \ell \left( \frac{c}{\ell} - \zeta \right) . \quad (47)$$

Substitution of equations (47) and (33) into equation (44) leads to the expression

$$\frac{d(\gamma/\gamma_e)}{d\xi} = \frac{G}{\tau_p^{\eta}} (\alpha_2 - \alpha_1) \Delta T \ell - \frac{\tau_p \ell^2}{E_1 t_1} + \frac{\tau_p \ell^2}{E_2 t_2} - \frac{(\tau_p - \tau_{av}) \ell^2}{(E_2 t_2)[1 - \frac{c}{\ell} + \zeta]} . \quad (48)$$

The transition relation at  $\zeta = 0$  follows as

$$\frac{d(\gamma/\gamma_e)}{d\gamma} \Big|_{\zeta=0} = (\lambda\ell) CTHERM(1) - (\lambda\ell)^2 \left[ \frac{1 - ETR(1)}{1 + ETR(1)} \right] - (\lambda\ell)^2 \frac{[1 - (\tau_{av}/\tau_p)] ETR(1)}{[1 + ETR(1)][1 - \frac{c}{\ell}]} . \quad (49)$$

Equation (48) may be integrated once, yielding

$$\left(\frac{\gamma}{\gamma_e}\right) = (\lambda \ell) C\text{THERM}(1) \zeta - (\lambda \ell)^2 \frac{1 - \text{ETR}(1)}{1 + \text{ETR}(1)} \zeta - \\ - (\lambda \ell)^2 \frac{[1 - (\tau_{av}/\tau_p)]\text{ETR}(1)}{[1 + \text{ETR}(1)]} \ln(1 - \frac{c}{\ell} + \zeta) + C , \quad (50)$$

in which, since

$$\gamma = \gamma_e \quad \text{at } \zeta = 0 , \quad (51)$$

$$1 = -(\lambda \ell)^2 \frac{[1 - (\tau_{av}/\tau_p)]\text{ETR}(1)}{[1 + \text{ETR}(1)]} \ln(1 - \frac{c}{\ell}) + C . \quad (52)$$

Signifying by  $\gamma_{max}$  the peak adhesive shear strain at the less critical (by definition) right hand end of the joint,

$$\left(\frac{\gamma_p}{\gamma_e}\right) > \frac{\gamma_{max}}{\gamma_e} = (\lambda \ell) C\text{THERM}(1) \left(\frac{c}{\ell}\right) - (\lambda \ell)^2 \left[\frac{1 - \text{ETR}(1)}{1 + \text{ETR}(1)}\right] \left(\frac{c}{\ell}\right) \\ + (\lambda \ell)^2 \frac{[1 - (\tau_{av}/\tau_p)]\text{ETR}(1)}{[1 + \text{ETR}(1)]} \ln\left(1 - \frac{c}{\ell}\right) . \quad (53)$$

A comparison of equations (43) and (53) shows complete consistency upon interchanging subscripts 1 and 2. While equation (53) could be employed to identify whether the left or right hand end of the joint in Figure 2 is more critical once the extent of the second plastic zone ( $c/\ell$ ) had been established, there is an inherent numerical difficulty in the step by step computation of the strength by the procedure outlined above. It was explained in Section 2 that, for the perfectly-elastic adhesive, only the average adhesive shear stress could be computed and not the stress distribution as a function of position along the joint. In the computer program in Appendix A3, the only reason why it proved possible to evaluate the extent of the elastic trough, for long overlaps, was the factor  $[1 - (a/\ell) - (c/\ell)]^{n-1} < 1$  in equation (34). At high values of  $n$ , this very small term was able to overpower the influence of the  $(\lambda \ell)^2$  factor in the numerator of the recurrence formula (18). This numerical accuracy problem prevented the reliable evaluation of  $d(\gamma/\gamma_e)/dx$  at  $x = (b/\ell)$  to match boundary conditions at the transition of the second plastic adhesive zone. Consequently an iterative solution had to be employed to evaluate the maximum possible

extent of the elastic trough.

Referring to equations (45) and (46), it can be seen that, in the iterative solution process, the second term of the elastic adhesive shear stress series  $A_2$  depends on the preceeding estimates of both  $(a/\ell)$  and  $(\tau_{av}/\tau_p)$ . In the early development of the digital computer program for elastic-plastic scarf joints insurmountable convergence difficulties were encountered if the initial estimates for  $(\tau_{av}/\tau_p)$  and  $(a/\ell)$  were not sufficiently close to the true values. This difficulty was eventually overcome by the following technique. Equation (43) was re-arranged to read

$$\left(\frac{\tau_{av}}{\tau_p}\right) = 1 - \frac{\left\{ \frac{[1 + ETR(1)]}{(\lambda\ell)^2} \left(\frac{\gamma_p}{\gamma_e}\right) + \left[ \frac{[1 + ETR(1)]CTHERM(1)}{(\lambda\ell)} - [1 - ETR(1)] \right] \left(\frac{a}{\ell}\right) \right\}}{\ln[1 - (a/\ell)]}. \quad (54)$$

This can be differentiated with respect to  $(a/\ell)$  so that

$$d(\tau_{av}/\tau_p) / d(a/\ell) = 0 \quad \text{when}$$

$$\left(1 - \frac{a}{\ell}\right) \ln\left(1 - \frac{a}{\ell}\right) = \frac{\left\{ \frac{[1 + ETR(1)]}{(\lambda\ell)^2} \left(\frac{\gamma_p}{\gamma_e}\right) + \left[ \frac{[1 + ETR(1)]CTHERM(1)}{(\lambda\ell)} - [1 - ETR(1)] \right] \left(\frac{a}{\ell}\right) \right\}}{[1 - ETR(1)] - \frac{[1 + ETR(1)]CTHERM(1)}{(\lambda\ell)}}. \quad (55)$$

Substitution of equation (55) into equation (54) yields, for the minimum (stationary) value of  $(\tau_{av}/\tau_p)$

$$\begin{aligned} \frac{\tau_{av}}{\tau_p} &= ETR(1) + \frac{[1 + ETR(1)]CTHERM(1)}{(\lambda\ell)} \\ &+ \frac{a}{\ell} \left\{ [1 - ETR(1)] - \frac{[1 + ETR(1)]CTHERM(1)}{(\lambda\ell)} \right\}. \end{aligned} \quad (56)$$

This is evidently consistent with the elastic solution  $(a/\ell) = 0$  for large overlaps and, upon subsequent comparison with the more precisely estimated joint strengths, proved to be an extremely close lower bound for all cases of practical interest. It is significantly conservative only for very short overlaps [small values of  $(\lambda\ell)$ ] or very brittle adhesives [very small values of  $(\gamma_p/\gamma_e)$ ]. The adhesive shear strain capacity  $\gamma_p$  is involved in equation (56) implicitly through the extent  $(a/\ell)$  of the plastic zone. Equation (55)

is solved by iteration to evaluate  $(a/\ell)$  and the result substituted into equation (56) or (54). Appendix A2 contains a listing of the FORTRAN IV digital computer program employed to solve equations (55) and (54), together with sample outputs and brief user instructions. The iteration technique eventually adopted proved to be quite convergent, after other re-arrangements of equation (55) demonstrated strongly divergent characteristics.

This program in Appendix A2 served to provide the initial estimates of  $(a/\ell)$  and  $(\tau_{av}/\tau_p)$  in the more precise solution listed in Appendix A3. The sequence of variables used in the solution is  $(a/\ell)$ ,  $(\tau_{av}/\tau_p)$  and  $(c/\ell)$  after which  $(\tau_{av}/\tau_p)$  is recomputed and the estimate of  $(a/\ell)$  adjusted until convergence is attained. In those cases in which the critical end is not evident by inspection, the potential bond shear strength is computed from each end of the joint and the lower value adopted. Brief user instructions and sample outputs are included in Appendix A3.

The analyses above for scarf joints pertain to adhesive shear stresses and it is demonstrated that a small enough scarf angle can always be found to transfer the full adherend strength through the bond with an adequate margin. There is, of course, a potential problem with the adherend strength(s) if the scarf angle is too small. Specifically, one adherend will fail if the scarf angle  $\theta$  is so small that

$$\theta < \tau_p/F_u , \quad (57)$$

(where  $F_u$  is the ultimate adherend stress in tension, compression, or shear, as appropriate) at the more critical end of the joint (identified by the adhesive shear stress analysis). Should this situation arise, the solution is to decrease the adherend stiffness imbalance across the joint by local reinforcement of the softer adherend. It is evident from equation (17) that this potential problem of breaking off the tip of (usually) the stiffer adherend is more likely to arise with the brittle adhesives (higher values of peak adhesive shear stress  $\tau_p$ ) than with ductile adhesives. This is one important reason for preferring to effect the load transfer with a shorter overlap of ductile adhesive than with a longer overlap of brittle adhesive. The extreme case of making the overlap so extremely long that the peak adhesive shear

stress actually developed is restricted to a small fraction of its capacity when adherend failure occurs outside the joint has theoretical appeal only, frequently being quite impractical.

#### 4. DISCUSSION OF PARAMETRIC EFFECTS

Representative solutions from Sections 2 and 3 for unbalanced bonded scarf joints are illustrated in Figures 3 through 7. Figures 3 and 4 show the separate effects of adherend stiffness and thermal mismatch, respectively, on the elastic joint strength. The deviations from unity in the  $(\tau_{av}/\tau_p)$  ratio, for a given overlap ( $\lambda l$ ), are proportional to the individual imbalances. The effect of stiffness imbalance is a smooth decrease from a fully-efficient bond ( $\tau_{av} = \tau_p$ ) to a less efficient bond ( $\tau_{av} < \tau_p$ ) asymptotizing towards the solution given in equation (21). This diagram, more than any other, characterizes the dominant feature of the scarf joint behavior. This is that the potential bond strength continues to increase indefinitely with increasing overlap. This is in marked contrast to the behavior of uniform lap joints [References (1) and (2)], which develop maximum strengths which remain effectively constant beyond intermediate overlaps. The effect of this characteristic on the potential bond strength of scarf joints is that, by making the scarf angle sufficiently small, one can always design a joint in which the potential bond strength exceeds the adherend strength by any specified factor. This is amply demonstrated by curve D in Figure 4. While adherend stiffness and thermal mismatch combine to decrease the bond efficiency below the unit value of curve A, the bond strength for long overlaps ends up being proportional to the overlap. As a consequence of this characteristic, the elastic adhesive shear stresses play a far more important role in the strength of scarf joints than they do in the case of uniform lap joints. Nevertheless, it would be erroneous to conclude that one could always design an unbalanced scarf joint within the capabilities of an elastic adhesive. The limiting problem is that, as the scarf angle becomes very small, there is a strong probability of breaking off the tip of the stiffer adherend. While not as acute a design detail problem as its counterpart for stepped-lap joints, this feature restricts the scarf angle to exceed the value

$$\theta = \text{ARCTAN}(\tau_p/F_u) \quad (58)$$

in which  $F_u$  is the adherend ultimate strength (in tension, compression, or shear, as appropriate for the applied load).

The effect of adherend thermal mismatch on the potential bond strength of scarf joints is shown in Figure 4. It is clear that the effects are insignificant for very short and very long overlaps, being significant only for those

overlaps of practical interest. The effects are maximum at  $(\lambda l) = 2$  for all values of the thermal mismatch coefficient CTERM.

Figure 5 shows the interaction between adherend stiffness and thermal mismatch. Curves B, D and E represent one set of solutions, with curve B showing the effect of stiffness imbalance alone. Curve D adds the influence of compounding thermal mismatch as well. Curve E demonstrates the behavior of self-cancelling adherend imbalances at  $(\lambda l) = 3$ . For values of  $(\lambda l)$  less than 3, the thermal mismatch effects dominate over those arising from stiffness mismatch and the more critical end of the joint is reversed. Curves A, C and F form another set showing how, for severe adherend thermal mismatch, there is a range of overlaps for which the residual thermal stresses are so severe that the joint will split apart without the application of any mechanical loads. Quite unlike the behavior of uniform lap joints [References (1) and (2)], this problem can be eliminated completely by sufficient extension of the overlap.

Just as is the case for uniform lap joints adhesive plasticity can increase the potential bond shear strength. The extent of this strength increase is shown in Figures 6 and 7 for stiffness and thermal mismatch, respectively. For each amount of adhesive plastic shear strain, there is an associated overlap below which the bond can be uniformly stressed. For indefinitely large overlaps the asymptotic solution (21) again holds, masking completely the influence of any adhesive plasticity. In the overlaps of practical interest, the actual amount of adhesive plasticity available from real structural adhesives can improve the potential joint strength greatly. One benefit of using a ductile adhesive of moderately high peak shear stress rather than a brittle adhesive of very high peak shear stress is that the joint is better able to withstand the variation in joint load which inevitably occurs as the result of manufacturing imperfections and non-uniform load distribution. Another benefit is that the problem of breaking off the tip of the adherend at the more critically loaded end [see equation (58)] is greatly alleviated. If the tip of the stronger adherend were allowed to be broken off, this would impose an effective net area loss on the cross-section of the weaker adherend.

## 5. ELASTIC ANALYSIS OF STEPPED-LAP JOINTS

The analysis for the strength of stepped-lap adhesive-bonded joints contains features of both the uniform lap joints [References (1) and (2)] and the scarf joint above. Peel stress problems are ignored on the grounds that the outermost end steps are invariably thin enough (in good design practice) not to induce significant peel stresses in the adhesive. Likewise, the small eccentricity in the load path has been ignored in the interests of obtaining a useful uncomplicated design tool.

A representative idealized stepped-lap joint is shown in Figure 8, along with the sign convention and nomenclature necessary for the analysis. Just as for the scarf joint analysis, the same diagram serves also for the elastic-plastic analysis, so it contains information not necessary for the elastic analysis. This begins with the equilibrium equations for a differential element of one of the steps.

$$\frac{dT_o}{dx} + 2\tau = 0 , \quad \frac{dT_i}{dx} - 2\tau = 0 . \quad (59)$$

Here the subscripts  $o$  and  $i$  refer to the "outer" and "inner" adherends, respectively, and the factors 2 in equations (59) account for the two bond surfaces surrounding the inner adherend. Consequently the adherend thicknesses  $t_o$  and  $t_i$  refer to the total cross-section and the forces  $T_o$  and  $T_i$  do likewise. The nature of the solution is such that it is, on occasions, necessary to interchange the subscripts  $o$  and  $i$  mathematically. The thermo-elastic relations for the adherends are

$$\frac{d\delta_o}{dx} = \frac{T_o}{E_o t_o} + \alpha_o \Delta T , \quad \frac{d\delta_i}{dx} = \frac{T_i}{E_i t_i} + \alpha_i \Delta T . \quad (60)$$

The adhesive shear strain, for tensile lap shear loading, is

$$\gamma = (\delta_i - \delta_o) / \eta . \quad (61)$$

while the elastic adhesive shear stress is related to the shear strain by the relation

$$\tau = G\gamma = G(\delta_i - \delta_o)/\eta . \quad (62)$$

The solution proceeds just as in Reference (1).

$$\frac{d\tau}{dx} = -\frac{G}{\eta} \left[ \frac{d\delta_i}{dx} - \frac{d\delta_o}{dx} \right] = -\frac{G}{\eta} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]. \quad (63)$$

$$\frac{d^2\tau}{dx^2} = -\frac{G}{\eta} \left[ \frac{2}{E_i t_i} + \frac{2}{E_o t_o} \right] \tau = \lambda^2 \tau. \quad (64)$$

The solution of equation (64) is

$$\tau = A \cosh(\lambda x) + B \sinh(\lambda x) \quad (65)$$

where the integration constants A and B are to be determined by boundary conditions for each step. Substitution of equation into equation (59) yields

$$T_o = T_{o_{ref}} - 2 \frac{A}{\lambda} \sinh(\lambda x) - 2 \frac{B}{\lambda} [\cosh(\lambda x) - 1] \quad (66)$$

and

$$T_i = T_{i_{ref}} + 2 \frac{A}{\lambda} \sinh(\lambda x) + 2 \frac{B}{\lambda} [\cosh(\lambda x) - 1]. \quad (67)$$

The values of  $T_{o_{ref}}$  and  $T_{i_{ref}}$  depend upon the origin of x adopted. In the solution it proves convenient to adopt the start of each step as the origin for that step. Integrating again, by means of equations (69),

$$\delta_o = \delta_{o_{ref}} + \alpha_o \Delta T x + \frac{1}{E_o t_o} \left[ T_{o_{ref}} x - 2 \frac{A}{\lambda^2} \cosh(\lambda x) - 2 \frac{B}{\lambda^2} [\sinh(\lambda x) - (\lambda x)] \right] \quad (68)$$

and

$$\delta_i = \delta_{i_{ref}} + \alpha_i \Delta T x + \frac{1}{E_i t_i} \left[ T_{i_{ref}} x + 2 \frac{A}{\lambda^2} \cosh(\lambda x) + 2 \frac{B}{\lambda^2} [\sinh(\lambda x) - (\lambda x)] \right]. \quad (69)$$

In the FORTRAN IV digital computer program, listed in Appendix A4, used to solve the equations above for the elastic stepped-lap joint, the technique of solution is as follows. The solution proceeds, one joint step at a time starting with assumed values of the load and initial adhesive shear strain (or stress). The latter is set at the maximum adhesive allowable and remains so unless it is computed that the peak adhesive shear strain is greater elsewhere (most probably at the other end of the joint) in which case the initial strain is reduced as much as necessary to avoid exceeding the allowable. The key

equation in the solution is equation (65). The integration constant A is evaluated as the specified (or subsequently computed) adhesive shear stress at the start of the step under consideration.

$$A = \tau_{x=0} . \quad (70)$$

The other constant B derives from equation (63), also evaluated at the start of that step. That is

$$\frac{d\tau}{dx} = A\lambda \sinh(\lambda x) + B\lambda \cosh(\lambda x) = -\frac{G}{n} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right] \quad (71)$$

so that at  $x = 0$

$$B = \frac{G}{n\lambda} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]_{x=0} . \quad (72)$$

The values of  $\tau$ ,  $T_o$ ,  $T_i$ ,  $\delta_o$  and  $\delta_i$  at the end of that step then follow from equations (65), (66), (67), (68), (69) and (62), respectively. If, after one complete set of computations, the load computed to be transferred out of the far end of the joint does not match that assumed to act at the near (starting) end, the initial estimate is adjusted until the two quantities do match. At that stage, a check is made throughout the joint, step by step, to identify the most critical adhesive and adherend locations. If any negative margins are identified, the load and peak adhesive shear stress are reduced as much as is necessary to eliminate them.

While the formulation of the equations and analysis scheme above is quite straightforward, the actual numerical solution of the problem proved to be quite difficult. Even with double precision it was almost invariably impossible to compute values for all steps of the joint in a single pass, even if the initial conditions (load and peak adhesive shear stress) were precisely correct to 16 significant figures. A change of 1 in the 16th significant digit of an initial condition would frequently effect a change by a factor of up to  $\pm 10^7$  in a quantity computed in the fourth or fifth step. This was not the result of a poorly conditioned mathematical formulation. It follows directly from strong physical characteristics of stepped-lap joints. It is the nature of stepped-lap joints, be they bonded or bolted, that any non-uniformities in the load transfer are dominated by the geometry and materials of the end three

steps. What happens in between has only negligible effect on the critical loads which almost invariably occur at one end or other of the joint. Likewise, in a uniform lap joint, practically all the load is transferred through the end three (rows of) bolts or through a narrow effective end zone of adhesive. Because of this characteristic the initial coding of the equations led to a highly accurate estimate of the load (assuming that the adhesive was critical at one end of the joint) but was unable to compute the internal loads and check on the adherend strength margin. The technique finally employed for dealing with this problem took advantage of the seemingly undesirable characteristics and is summarized as follows. By printing out intermediate computations it became clear that, if the initial load estimate on a given step was too high (even if only minutely), on the step just before computations for a subsequent step caused overflows and underflows in the computer the computations would diverge in a characteristic way, precisely the opposite of that for an initial underestimate of that load. Therefore upper and lower bounds were placed on the load estimate and the trial load was taken as the average of these. If the trial load was found to be too high, it served as the new upper bound and, were it too low, it was used to raise the lower bound. This technique was found to bring the upper and lower bounds into precise agreement rapidly. Once this had occurred the computations for the start of that step were frozen and the solution proceeded to perturb each successive step in turn, using the same convergence check above, until the load transferred out of the far end of the joint precisely equalled that input at the near end. Then a check is made, at the ends of each step, on the adhesive and adherend stresses to ensure that neither exceeds the allowable. Due allowance is made for the sign of the quantities involved. In the absence of any thermal mismatch this last operation of checking on the allowables can be performed by simple linear scaling. However, if there is any adherend thermal mismatch present, this adjustment must be performed by iteration since, as is evident from equation (62), the thermal stress terms do not scale in proportion to the adhesive and adherend stresses. A necessary check on the accuracy of the numerical processes has been accomplished by checking that precisely the same solution is obtained regardless of whether the computations commence at the more critically loaded end of the joint or at the other end.

In view of the numerical problems encountered with this analytical solution, it stands to reason that they will have their counterpart in any finite-element solution. Very fine grids would be needed in the high stress gradient areas.

## 6. ELASTIC-PLASTIC ANALYSIS OF STEPPED-LAP JOINTS

In addition to the equations of Section 5 for the perfectly elastic analysis of stepped-lap joints, the elastic-plastic analysis requires, instead of equation (62), that

$$\tau = \tau_p \quad \text{for } \gamma \geq \gamma_e , \quad (73)$$

and

$$\tau = G\gamma \quad \text{for } \gamma \leq \gamma_e . \quad (74)$$

The elastic-plastic solution is best carried out in terms of the adhesive shear strains rather than the shear stresses. In the plastic adhesive zones, from equations (61) and (60),

$$\frac{d\gamma}{dx} = \frac{1}{n} \left[ \frac{d\delta_i}{dx} - \frac{d\delta_o}{dx} \right] = \frac{1}{n} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right] \quad (75)$$

whence, from equations (59)

$$\frac{d^2\gamma}{dx^2} = \frac{2}{n} \left[ \frac{1}{E_i t_i} + \frac{1}{E_o t_o} \right] \tau_p = \frac{\lambda^2}{G} \tau_p = \text{constant} . \quad (76)$$

Therefore, in the plastic zone,

$$\gamma = \frac{\lambda^2}{2G} \tau_p x^2 + Cx + D \quad (77)$$

and

$$T_o = T_{o_{ref}} - 2\tau_p x , \quad T_i = T_{i_{ref}} + 2\tau_p x \quad (78)$$

while

$$\delta_o = \delta_{o_{ref}} + \alpha_o \Delta T x + \frac{1}{E_o t_o} \left[ T_{o_{ref}} x - \tau_p x^2 \right] \quad \left. \right\} . \quad (79)$$

and

$$\delta_i = \delta_{i_{ref}} + \alpha_i \Delta T x + \frac{1}{E_i t_i} \left[ T_{i_{ref}} x + \tau_p x^2 \right] \quad \left. \right\}$$

In equation (77), D is set equal to  $\gamma$  at the start of any step, since a new zero for  $x$  is chosen at that location for each step. The other constant C follows from equations (75) and (77). Thus

$$C = \frac{d\gamma}{dx} \Big|_{x=0} = \frac{1}{n} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]_{x=0} . \quad (80)$$

Very few individual steps of stepped-lap joints have fully-plastic adhesive throughout the entire joint. Any adhesive plasticity is frequently confined to the end(s) of the step(s). Therefore, in performing an elastic-plastic analysis of a stepped-lap joint, it is necessary to be able to compute the extent of the plastic zones. Therefore, beginning at the left hand end of the step element shown in Figure 8 and assuming a sufficiently high load intensity for the adhesive to be in the plastic state, the first computation is that of the maximum possible extent of the plastic zone. This is then compared with the actual extent of the step. If necessary, a second computation is performed of the maximum possible extent of the elastic trough in that same step. Starting from equation (77) with  $\gamma = \gamma_{ref}$  at  $x = 0$ ,

$$\gamma = \frac{\lambda^2}{2G} \tau_p x_p^2 + Cx + \gamma_{ref} \quad (81)$$

where the constant C is given by equation (80). It is necessary to find the lesser value of  $x$  for which  $\gamma = \gamma_e$ . Equation (81) is re-arranged to read

$$\frac{\lambda^2 \tau_p}{2G} x_p^2 + Cx_p + (\gamma_{ref} - \gamma_e) = 0 \quad (82)$$

so that the maximum extent of plastic adhesive zone is given by

$$x_p = -C \pm \sqrt{C^2 - 2\lambda^2 \tau_p (\gamma_{ref} - \gamma_e)} . \quad (83)$$

Now, since  $C = d\gamma/dx < 0$  at  $x = 0$  the minus sign in front of the radical holds. Once  $x_p$  has been computed, it is compared with the step length  $\ell_{step}$ . If  $x_p > \ell_{step}$ , that particular step is fully-plastic throughout and the values of the various quantities at the far end of the step are evaluated from equations (73) to (80). Should  $x_p$  be less than  $\ell_{step}$ , the difference is examined elastically, to see whether it remains elastic throughout or becomes plastic again at the far end. For  $x_p < \ell_{step}$ , the values of the various stresses, strains,

displacements and forces are evaluated in terms of equations (73) to (79) and the subscripts  $pe$  serve to identify the plastic-to-elastic transition. Likewise  $ep$  identifies the possible elastic-to-plastic transition at the far end of the joint. It is necessary that  $d\gamma/dx$  be maintained at these transitions, as is evident from equation (75). The maximum possible extent of elastic trough must be deduced from equation (65). In doing so, it is mathematically far simpler to shift the  $x$  origin to the middle of the elastic trough (of extent  $2x_e$ ) so that

$$\tau = \tau_p \cosh(\lambda x) / \cosh(\lambda x_e) \quad (84)$$

At the  $pe$  transition ( $x = -x_e$ ) equation (62) requires that

$$\frac{d\tau}{dx} = \frac{G}{\eta} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]_{pe} = -\tau_p \lambda \tanh(\lambda x_e) \quad (85)$$

so that the elastic trough could extend, if  $\lambda_{step}$  were great enough, a distance

$$2\lambda x_e = \tanh^{-1} \left\{ -\frac{1}{\lambda \eta \gamma_e} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]_{pe} \right\}. \quad (86)$$

By use of known formulas for hyperbolic functions in terms of exponentials and the interrelation between exponential and logarithmic functions, the solution (85) is more conveniently expressed as

$$2x_e = \frac{1}{\lambda} \ln \left\{ \frac{1 - \frac{1}{\lambda \eta \gamma_e} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]_{pe}}{1 + \frac{1}{\lambda \eta \gamma_e} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]_{pe}} \right\}. \quad (87)$$

In the event that  $x_e$  does not extend beyond the far (right hand) end of the step being analyzed, it is necessary to compute the load transferred between the adherends throughout the elastic trough. In doing so, it is quite simple to take the value of  $2x_e$  from equation (87) and substitute it back into equations (65) to (72) for the standard elastic analysis of the preceding section. Should the elastic trough not extend to the far end of the step under analysis, equations (73) to (80) are employed for the plastic zone to the end of the step.

Equation (77) now becomes

$$\gamma = \frac{\lambda^2 \tau_p}{2G} x^2 + Cx + \gamma_{ep} \quad (88)$$

with

$$\tau = \tau_p \quad \text{for } x > x_{ep} . \quad (89)$$

The constant  $C$  in equation (88) is evaluated in terms of equation (75)

$$C = \left. \frac{d\gamma}{dx} \right|_{ep} = \frac{1}{\eta} \left[ \frac{T_i}{E_i t_i} - \frac{T_o}{E_o t_o} + (\alpha_i - \alpha_o) \Delta T \right]_{ep} . \quad (90)$$

In the last steps of the joint at the far end, the adhesive may be fully plastic throughout in which case, in equation (87),  $\gamma_{ep}$  should be replaced by  $\gamma_{ref}$ . Likewise, in those steps, near the middle of the joint, in which the adhesive shear strains are so small as not to reach the plastic state at either end of the step, the step will be elastic throughout and equations (65) to (72) are employed in the analysis. Towards the far end of the joint there may be a step which starts elastically and becomes plastic. In this case the actual extent of elastic behavior is determined by iteration, using equations (65) to (72) with a cut off (either positive or negative) on the shear stress.

If it should transpire that, at the end of the step,  $\gamma$  exceeds  $(\gamma_e + \gamma_p)$  or  $T_i$  or  $T_o$  exceed their respective allowables, this does not cause any analytical difficulty. An iterative procedure is employed in the analysis to reduce the external load and initial adhesive strain whenever necessary. While this does not represent any analytical difficulty, one should recognize that exceeding the allowables on an inner step can occur only as the result of poor detail design. The improvement of such details can increase the potential joint strength.

No new numerical difficulties were encountered in the program listed in Appendix A5 for the elastic-plastic analysis of stepped-lap joints which did not have a direct counterpart in the perfectly elastic analysis. However, the logic associated with keeping track of the locations of the transitions between elastic and plastic adhesive behavior, and vice versa, as they moved with each successive iteration posed a formidable problem. One small computational

problem was that, if the load estimate at some early stage in the iteration sequence was too far removed from the correct value, the computer would predict physically unrealizable large negative shear strains in the adhesive. A special set of instructions was prepared for this quirk.

The computer program, as basically written, checks simultaneously for the allowable adherend and adhesive strengths at the most critical locations in each step. Since stepped-lap joints are frequently more critical in the adherend than in the adhesive, a special feature has been added to increase greatly all adherend strengths artificially in order to print out also the potential adhesive bond strength and confirm that it exceeds the adherend strength by an adequate margin.

The analysis above is presented for the case of tensile lap shear loads being positive and the sign convention is in accordance. The computer programs have been so coded that, by a single input for the variable SGNLD, the respective solutions for tensile shear loading ( $SGNLD = +1$ ) and compressive shear loading ( $SGNLD = -1$ ) can be printed out. In the event that there are simultaneous stiffness and thermal mismatches between the adherends, the joint strength will not be the same for each load sense. Such a situation is common in the bonding of titanium edge members to boron-epoxy panels.

## 7. DISCUSSION OF DESIGN OF STEPPED-LAP JOINTS

The digital computer programs developed above to analyze stepped-lap joints can serve also as a useful design tool. Three clear dominant joint characteristics have been confirmed by studies with this program. The first is that the joint load capacity is defined by the end three steps at the more critical end of the joint. If other steps have a significant influence it will be adverse and be due to poor design detailing. The second is that, once the joint is essentially well-designed, quite major changes can be made to other than the critical end three steps without any significant impact on the joint strength. Third is that, in a well-designed joint, it is the very end step that is likely to precipitate joint failure unless its length is restricted in the design process. The necessary restriction is that the product of maximum adhesive shear stress and total bond area on the end step must not exceed the product of adherend material allowable and, cross section of the end step. Consequently, a ductile adhesive with higher strain energy provides stronger joints than a brittle adhesive with higher peak stress but less strain energy. It should be noted also that minimizing adherend stiffness imbalance increases the potential bond shear strength.

Mathematically speaking, the stepped-lap family of joints represent perturbations about the scarf joint solution. These perturbations become progressively greater as the number of steps decreases until the stepped-lap solution reduces to a single-lap joint for one step. Stepped-lap joints with only two or three steps are usually confined to thin adherends for which the potential bond shear strength is far in excess of the adherend(s) strength. In such cases the added strain concentrations in the bond due to the step discontinuities are not very important. Most applications of stepped-lap joints contain a large number of steps and, with a ductile adhesive softening the most severe of the adhesive stress spikes, the behavior very closely approaches that of the scarf joint. For this reason, preliminary design of practical stepped-lap joints by means of the scarf joint solution appears to be quite realistic. In doing so, however, one should exercise caution with regard to the critical end step of the adherend. The stepped-lap joint analysis, and practical experience, have identified the end step of the stiffer adherend as a prime candidate for the most critical design detail. If the extensional modulus of a composite adherend is

significantly less than that of a metal adherend to which it is bonded, most of the shear load transfer will be concentrated at the composite end of the joint with the probable result that tip fracture of the stiffer adherend will occur. One simple remedy to this potential difficulty is to be found in the concept of the dual-slope scarf joint illustrated in the upper part of Figure 9. In this joint, in order to protect the tip of the adherend, the scarf angle  $\theta_1$  is set to exceed

$$\theta_{1\min} = \tau_p / F_u \quad (91)$$

in which  $\tau_p$  is the peak adhesive shear stress and  $F_u$  is the appropriate adherend allowable stress in tension, compression, or shear as dictated by the nature of the applied load. The next step in the preliminary design process is to estimate the total scarf length necessary to effect the transfer of the entire load  $P$ . A reasonable approximation to this is given by the approximation

$$P = \left( \frac{\tau_{av}}{\tau_p} \right) \tau_p l \approx \left[ \frac{E_1 t_1}{E_2 t_2} \right] \tau_p l \quad (92)$$

for the asymptotic scarf joint solution for very long overlaps, whence

$$l \approx \frac{P}{\tau_p} \left[ \frac{E_2 t_2}{E_1 t_1} \right]. \quad (93)$$

The optimum location of the transition from scarf angle  $\theta_1$  to  $\theta_2$  can then be determined by trial and error using the stepped-lap joint computer program developed in Section 6. As a preliminary guide, it is suggested that one third of the total thickness be tried. The conversion of this conceptual scarf joint design into a practical stepped-lap joint is illustrated schematically in the lower part of Figure 9. It should be noted that the steps are thinner in the more critical load transfer region, and at the extreme opposite end for a single step to minimize potential peel stress problems. Normally peel stresses will not be a problem with stepped-lap joints for practical design configurations but the double-lap joint analysis can serve as a check if appropriate. The larger step sizes in the lightly loaded area effect an economy of fabrication which offsets the greater expense of proper detailing in the more critical areas.

For reasons evident from the discussion above, the dual-slope scarf joint has merits in its own right as well as for a model for approximate stepped-lap joint analysis. The steepening of the scarf angle at one end is particularly important for the brittle adhesives for which  $\tau_p$  is much higher than for the ductile adhesives. This greater importance follows from equation (91).

One characteristic of the internal stress distribution within stepped-lap bonded joints is directly traceable to double-lap joint phenomena and has no counterpart in scarf joint behavior. This characteristic is that, once each or any step is sufficiently long to contain a fully-developed elastic trough in the adhesive shear-stress distribution, an increase in that step length does not alter the joint shear strength. Indeed, as confirmed by application of the computer programs A4EF and A4EG, the internal adherend and adhesive stresses at the ends of each and every step are invariant with respect to such step length increases, whether one, some, or all of the step lengths are increased. That this should be so follows directly from the governing equations for each step of the joint. These are precisely the same as for an unbalanced double-lap joint, the shear strength of which is independent of overlap beyond some value. The impact of this phenomenon on the design of stepped-lap bonded joints is that, if analysis indicates inadequate bond strength and the overlap is already reasonably great, no further increase in step lengths can accomplish an improvement in joint strength. It is necessary to increase the number of steps and decrease the incremental step thickness.

The technique of refining the preliminary analysis developed by the rules above is as follows. An analysis is performed, and the limiting (critical) detail identified. If this is the strength of the end step of the stiffer adherend, the appropriate procedure is to decrease this length and increase the length of the other steps. A halving of the step thickness increment and doubling of the number of steps at the more critical end of the joint will also help. This situation can be identified by a solution in which the maximum adhesive shear strain developed is less than the allowable. In rare instances it may not be the very end but one or two steps inside which are critical. The procedure for improving the joint strength is the same. Reduce the length of the critical steps and increase the others. In doing so, it should be remembered that any fully-elastic step will not transfer much more load even if its length is

increased. Furthermore, if the adhesive shear stresses at each end of the step are less than their plastic value, increasing the step length indefinitely will not introduce a plastic zone. If the adhesive shear strain is predicted to be the limiting feature rather than the adherend strength, the joint strength may be improved by increasing the number of joint steps. In doing so, steps at one end of the joint will tend to become critical and length increases in the remaining (elastic) steps will continue to increase the joint strength, but at a decreasing rate. The behavior of bonded scarf joints (Figure 6) serves to explain this approach. Since the average bond stress on a scarf joint approaches a fixed fraction of the maximum bond shear stress, an overlap sufficiently long can always be found to develop a potential strength 50 percent in excess of the adherend strength. The only inherent difficulty in this approach is the care needed not to exceed the adherend allowables near the more critical end of the joint. One may look upon an optimally designed stepped-lap joint as an approximation to a dual slope scarf joint with a small angle at the less critical end to build up the total load transferred and a steeper angle (still small) at the more critical end to prevent breaking off the tip of the adherend.

In the presence of adherend thermal mismatch (advanced composite-to-metal for example), a reversal of load direction can reverse the more critical and less critical ends of the joint. Therefore it is necessary in such cases to design for both the maximum tensile shear and compressive shear loads to be applied.

Figures 10 to 12 illustrate solutions obtained to stepped-lap bonded joint analyses using the computer programs above. The joint is drawn to scale in Figures 10 and 11 and the material properties can be found in the sample printout included in the Appendix. The brittle and ductile adhesives referred to are, respectively, Narmco Metlbond 329 and Hysol EA951 which have the shear characteristics illustrated in Figure 13. The elastic solutions in Figure 10 show dramatically the sharp spikes in the shear stress distribution at the ends of each step. These spikes, separated by relatively lightly-loaded troughs, represent the influence of the uniform thickness steps. It is evident also from Figure 10 that the ductile adhesive, with its lower modulus and higher elastic shear strain carries slightly more load elastically than does the brittle adhesive. Figures 10 to 12 omit the influence of thermal mismatch between adherends and, had this been included, the elastic strength disparity in

Figure 10 would have been very pronounced in favor of the ductile adhesive for a tensile shear loading. Figure 11 shows the computed bond shear stress distributions, corresponding with Figure 10, when the adhesive properties are modified to account for plasticity. As is to be expected from the adhesive characteristics in Figure 13, this modification does not increase the joint strength of the brittle adhesive sufficiently for the bond to be stronger than the weaker adherend. The ductile adhesive, on the other hand, is computed to have a potential bond strength nearly as great as the strength of the titanium outside the joint. Actually, by the time the adhesive has used up only 15 percent of its total shear strain capacity, the load level is so high as to cause the end (thin) titanium step to yield, as shown in the middle illustration of Figure 11. The ductile adhesive has a considerable strength margin over the composite adherend. Figure 12 demonstrates how the theory identifies the end metal step as being prone to fatigue failure, even though the end step had been shortened to alleviate the problem. In the static load case the theory predicts that, once the titanium has yielded locally, as shown in the second illustration of Figure 12, the load level will increase until failure occurs in the composite at the end of the joint, as shown in the fourth illustration. Figures 11 and 12 depict only the most critical conditions within each step because the computer program does not normally output a continuous solution. The adhesive shear stress distribution throughout the lightly loaded regions is not crucial to the design/analysis cycle. For illustrative purposes one can easily artificially divide each step into a number of short segments in order to avoid adding another computation sequence to the programs. This has been demonstrated to be free from convergence problems (as confirmed by Figure 10) but, naturally, takes more computer time.

The following table enumerates a number of solutions obtained with the stepped-lap joint computer programs above. The effects of thermal stresses are included, as also is the influence of the direction of the applied load. Of interest is the way in which the adherend thermal and stiffness imbalances compound to decrease the joint strengths for tensile loading while they counteract each other for the compressive loading. The failure modes predicted are identified by the comment codes 1 through 5 which are explained at the foot of the table. All cases except those for optimized step lengths have the joint geometry shown in Figure 10. In optimizing the joint proportions, the computer program

STRENGTHS OF STEPPED-LAP ADHESIVE-BONDED JOINTS

JOINTS OF TITANIUM TO ISOTROPIC HTS GRAPHITE-EPOXY  
TITANIUM 0.25 IN. THICK    GRAPHITE-EPOXY 0.264 IN. THICK

FAILURE LOADS (LBS/INCH)

ADHESIVE	$\Delta T$	0° TENSION & COMPRESSION	-280°F TENSION	-280°F COMPRESSION	OPTIMIZED STEP LENGTHS		-400°F TENSION	-400°F COMPRESSION
					-280°F TENSION	-280°F COMPRESSION		
EPON 951 PURELY-ELASTIC ELASTIC-PLASTIC POTENTIAL BOND STRENGTH COMMENTS		7829 14430	4927 11866	10730 16997	4367 18180	9990 18182	3683 10769	9203 17821
METLBOND 329 PURELY-ELASTIC ELASTIC-PLASTIC COMMENTS		28362 1 , 2	26099 1 , 2	30569 1 , 4	23257 5	27299 5	25123 1 , 2	45000 1 , 2
METLBOND 329 PURELY-ELASTIC ELASTIC-PLASTIC COMMENTS		6764 13505 3	3812 10555 3	8552 16457 3			2547 9290 3	6152 17720 3

- COMMENT LEGEND :    1. TITANIUM YIELDS ON END (THIN) STEP  
                       2. FAILURE IN COMPOSITE OUTSIDE JOINT AT 18216 LB/IN.  
                       3. FAILURE IN ADHESIVE AT COMPOSITE END OF JOINT  
                       4. FAILURE IN COMPOSITE ONE STEP IN FROM TITANIUM END OF JOINT  
                       5. FAILURE IN COMPOSITE ONE STEP IN FROM COMPOSITE END

$\Delta T$  = OPERATING TEMPERATURE - CURE TEMPERATURE OF ADHESIVE

was used to identify the most critical location and the step lengths were modified by hand for re-analysis until the minimum tensile and compressive joint strengths matched the composite adherend strength. This took only two iterations to achieve the results shown and this feature is one of the more beneficial merits of the complete internal joint analysis.

Figure 14 illustrates the bond shear stress distributions for both ductile and brittle adhesives. A comparison is effected between a joint of uniform step lengths, at left, and that with optimized lengths, at right. A small loss in elastic joint strength is incurred by shortening the end steps (and some of this could be recovered by increasing the lengths of the other steps to compensate) but the problem of yielding the end titanium step has been eliminated for the ductile adhesive. It is interesting to observe that the brittle adhesive had insufficient strain energy in shear for the problem to arise. Another important phenomenon revealed is that the ductile adhesive uses up only about a third of its ultimate shear strain capacity in breaking the composite adherend just outside the joint. This leaves a generous margin for dealing with

stress concentration due to irregularities in load intensity or bond thickness across the width of the joint. Because of these ever-present considerations, the brittle adhesive should not be expected to develop the full predicted joint strength over each inch of a wide panel. Failure would be initiated by a local effect and then be propagated rapidly.

Figure 14 omits consideration of thermal effects in order not to complicate the comparisons made. Figure 15 includes these effects for both tensile and compressive shear loading with the ductile adhesive. This figure compares the performance of the preliminary design (Figures 10 and 11) with the optimized design. Improvements in ultimate compressive strength and tensile fatigue load capacity are demonstrated.

## 8. CONCLUSION

This report presents elastic and elastic-plastic analysis methods for adhesive-bonded scarf and stepped-lap joints. The solutions obtained are analytic in form and the necessary digital computer FORTRAN IV programs are listed in the Appendices. These solutions are believed to be the first for such joints which account for adhesive plasticity. They include also the effects of adherend stiffness- and thermal-mismatch. While the precise solutions require iterative numerical solutions, explicit algebraic formulas are derived for a close lower-bound on the strength of scarf joints. The dominant characteristic of scarf joints is that, for long overlaps, the average bond stress asymptotes towards a fixed fraction of the peak bond stress, that fraction equalling the lesser ratio of adherend extensional stiffnesses. Unlike uniform lap joints, which reach a definite strength limit which cannot be exceeded by using longer overlaps, the potential bond strength of scarf joints increases indefinitely with overlap so that a design can always be devised in which the failure is forced to occur outside the joint. In using this approach, however, it is necessary also to check on the adherend stresses at the tip of the stiffer adherend to ensure that the scarf angle is not too small. Stepped-lap joints exhibit some characteristics of both the scarf joint and uniform double-lap joints. Those steps near the middle of a stepped-lap joint carry significantly more load than that transferred in the corresponding area of a uniform lap joint but the load so transferred is usually not a major contribution. Most of the load transfer is effected through the end three steps at one or both ends of the joint, depending on the nature of the adherend imbalances and the direction of the load. Within each step, since the governing equations are precisely the same as for an unbalanced double-lap joint, it is found that no further load can be transferred once the overlap has exceeded a determinable value. In other words, unlike scarf joints, the potential shear strength of stepped-lap joints cannot be increased indefinitely by increasing the overlap(s). The appropriate procedure is to employ more steps of finer thickness increments in order to augment the load capacity. Because scarf and stepped-lap joints can efficiently transfer load between thicker adherends than is possible with uniform double-lap joints, the latter are restricted to thinner sections in practical applications.

The inclusion of adhesive plasticity in the analysis has a marked effect on the

strength predictions. On the other hand, the elastic adhesive stresses play a far more important role in the behavior of scarf and stepped-lap joints than they do for uniform lap joints. The inclusion in the analyses of thermal mismatch effects permits their application to the bonding of titanium to the advanced composite laminates and explains how the joint strength changes with the load direction in such a situation.

The elastic-plastic analysis of the internal stresses within stepped-lap bonded joints provides sufficient information for the joint proportions to be optimized. Analyses should be performed for each load direction and at the extremes of the environmental temperature range, taking due account of material property changes with temperature, in the optimization sequence.

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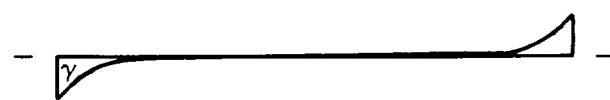
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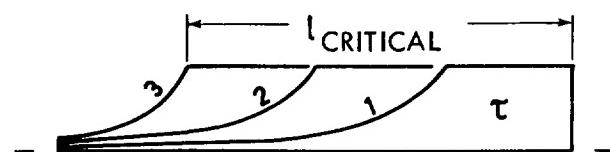
SCARF JOINT BETWEEN DISSIMILAR MATERIALS ( $\alpha_1 > \alpha_2$ ,  $E_1 t_1 = E_2 t_2$ )



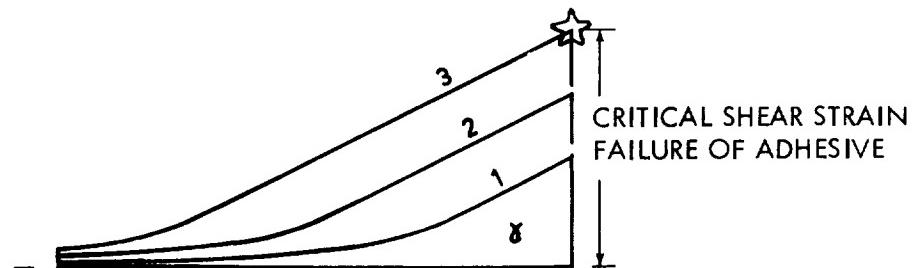
RESIDUAL ADHESIVE SHEAR STRESS



RESIDUAL ADHESIVE SHEAR STRAIN



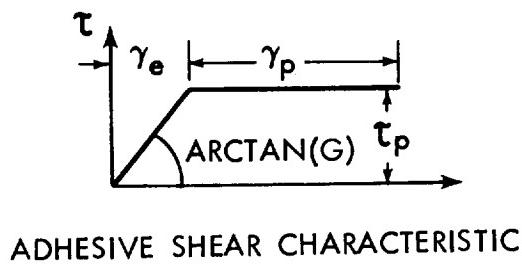
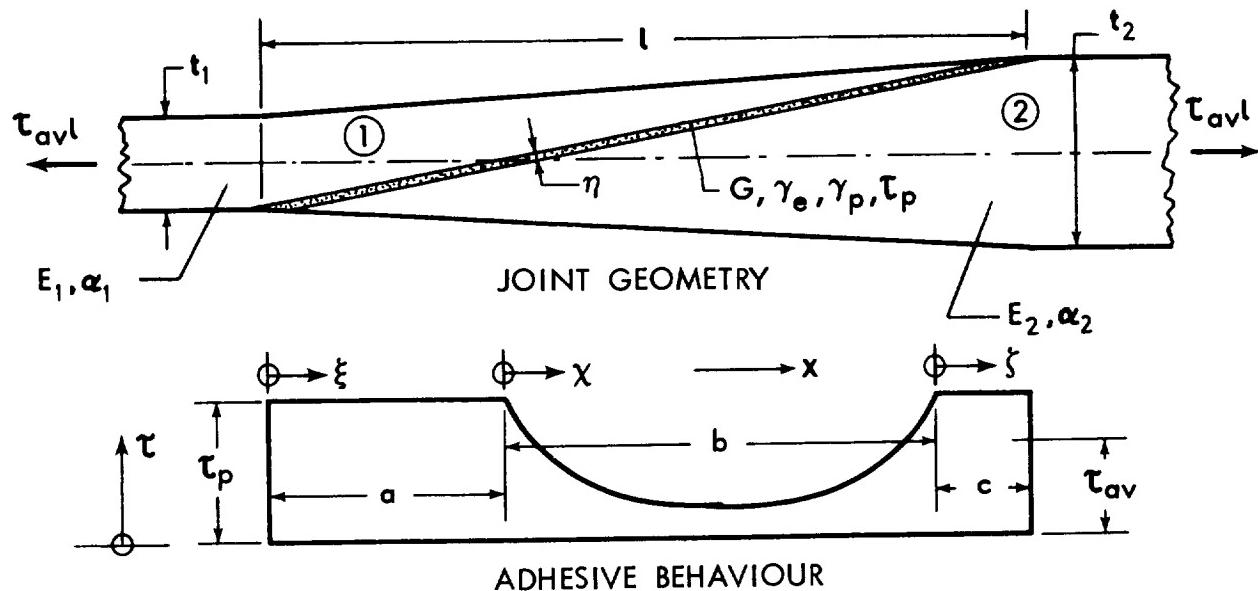
ADHESIVE SHEAR STRESSES  
UNDER PROGRESSIVELY INCREASING TENSILE LOADS



CORRESPONDING ADHESIVE SHEAR STRAINS

(NOTE THAT THE STRESSES AND STRAINS FOR PARTIAL LOADING, ABOVE, WOULD ALSO INDICATE THE BEHAVIOR UNDER STIFFNESS IMBALANCE IF  $\alpha_1 = \alpha_2$  AND  $E_1 t_1 > E_2 t_2$  BUT, WHEREAS THE CRITICAL END REVERSES WITH LOAD DIRECTION FOR THERMAL IMBALANCE, IT REMAINS THE SAME FOR STIFFNESS IMBALANCE.)

FIGURE I. EXPLANATION OF NON-UNIFORM ADHESIVE SHEAR STRESSES IN BONDED SCARF JOINTS BETWEEN DISSIMILAR ADHERENDS



$$(\lambda l)^2 = \frac{G}{\eta} \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right) l^2$$

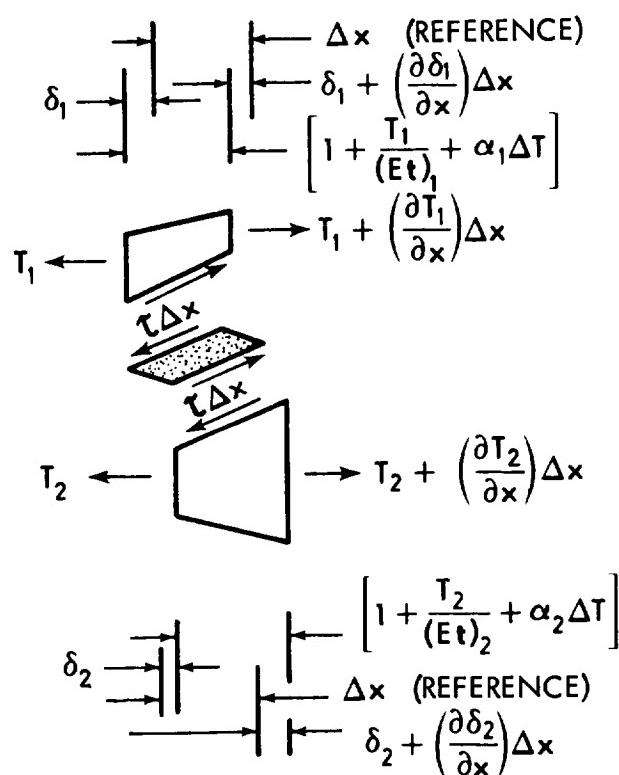
$$\text{ETR}(1) = E_1 t_1 / E_2 t_2$$

$$\text{ETR}(2) = E_2 t_2 / E_1 t_1$$

$$\text{CTHERM}(1) = \frac{(\alpha_2 - \alpha_1) \Delta T \lambda}{\tau_p \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right)}$$

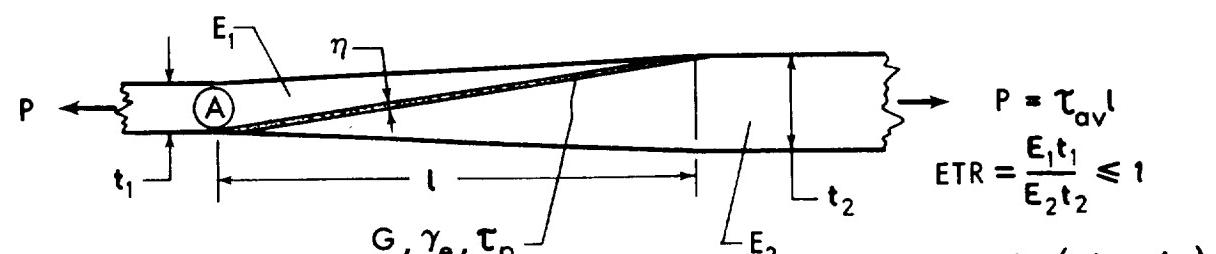
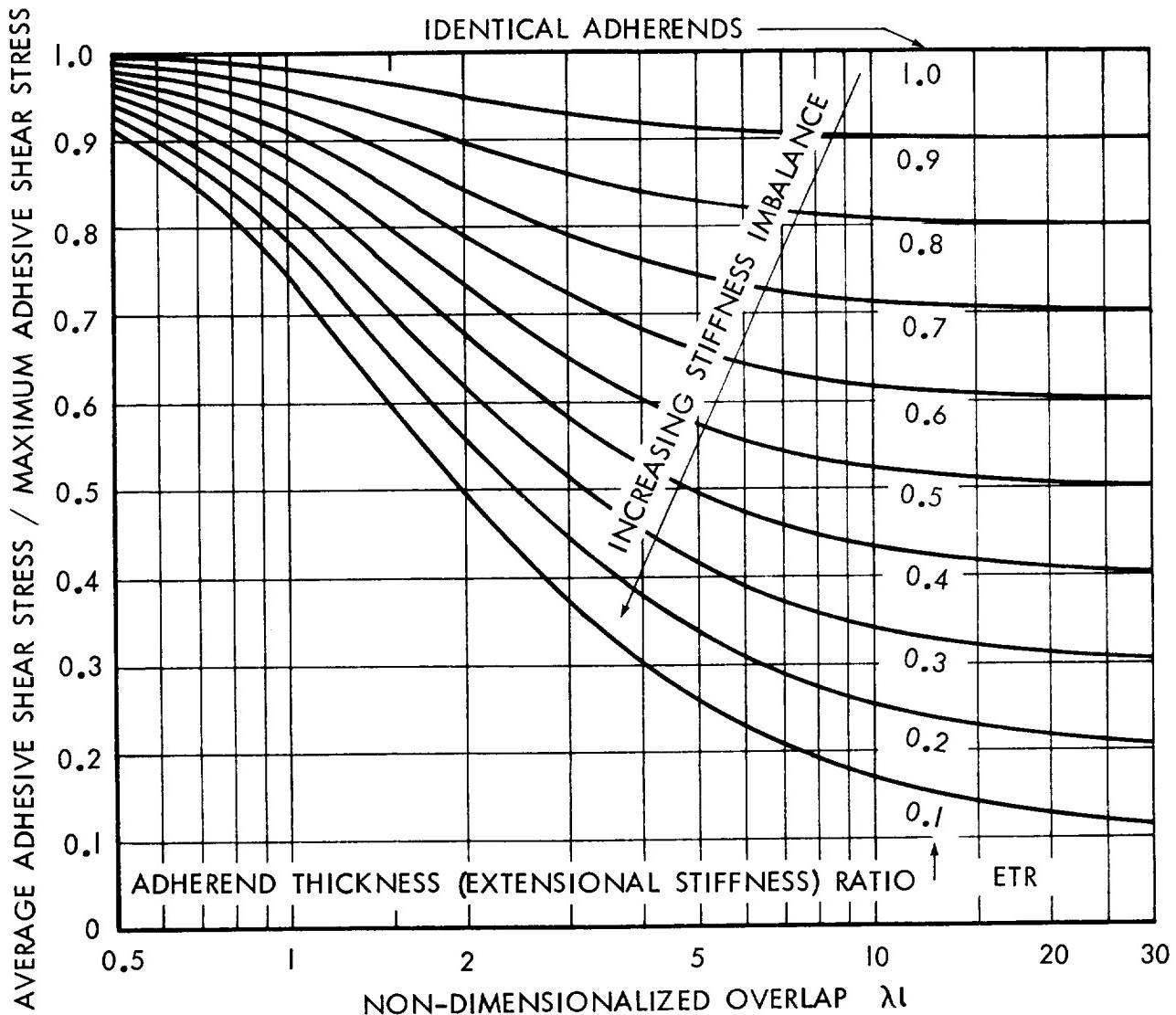
$$\text{CTHERM}(2) = \frac{(\alpha_1 - \alpha_2) \Delta T \lambda}{\tau_p \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right)}$$

NON-DIMENSIONALIZED  
JOINT PARAMETERS



DISPLACEMENTS AND ELEMENT LOADS

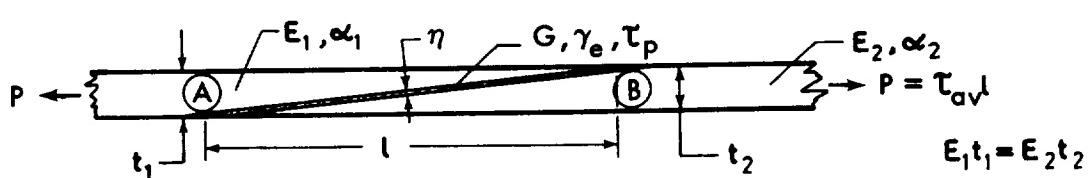
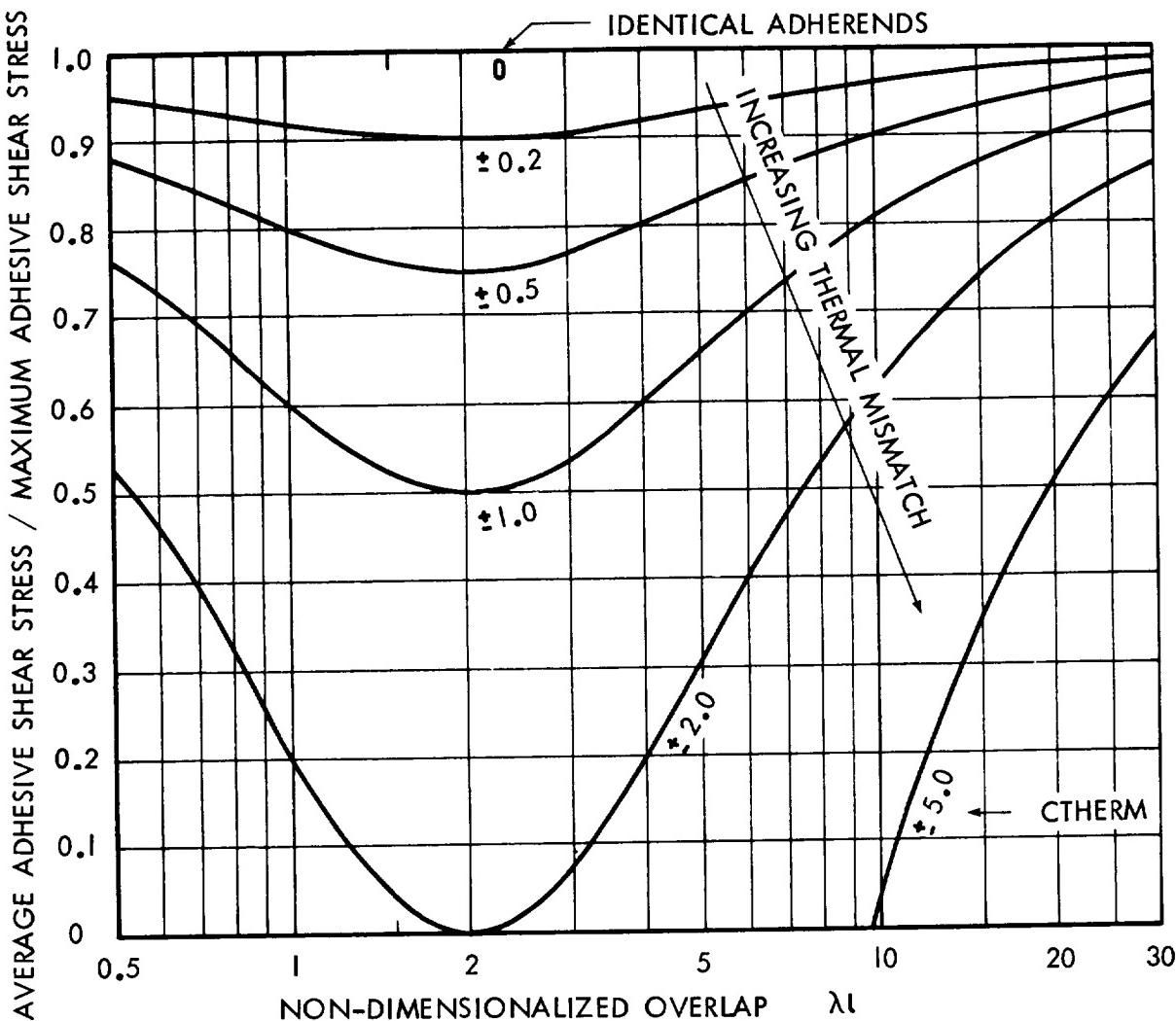
FIGURE 2. NOTATION AND GEOMETRY FOR ADHESIVE-BONDED SCARF JOINT ANALYSIS



$$\lambda^2 = \frac{G}{\eta} \left[ \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right]$$

LOCATION A IS CRITICAL FOR BOTH POSITIVE (TENSILE LAP-SHEAR)  
AND NEGATIVE (COMPRESSIVE LAP-SHEAR) VALUES OF LOAD P

FIGURE 3. EFFECT OF ADHEREND STIFFNESS IMBALANCE ON ELASTIC STRENGTH OF BONDED SCARF JOINTS



$$\lambda^2 = \frac{G}{\eta} \left[ \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right] \quad C_{THERM} = \frac{(\alpha_2 - \alpha_1) \Delta T \lambda}{\tau_p \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right)}, \quad \Delta T = T_{operating} - T_{stress\ free}$$

LOCATION A CRITICAL FOR  $C_{THERM} < 0$  AND  $P > 0$   
 LOCATION A CRITICAL FOR  $C_{THERM} > 0$  AND  $P < 0$   
 LOCATION B CRITICAL FOR  $C_{THERM} < 0$  AND  $P < 0$   
 LOCATION B CRITICAL FOR  $C_{THERM} > 0$  AND  $P > 0$

FIGURE 4. EFFECT OF ADHEREND THERMAL MISMATCH ON ELASTIC STRENGTH OF BONDED SCARF JOINTS

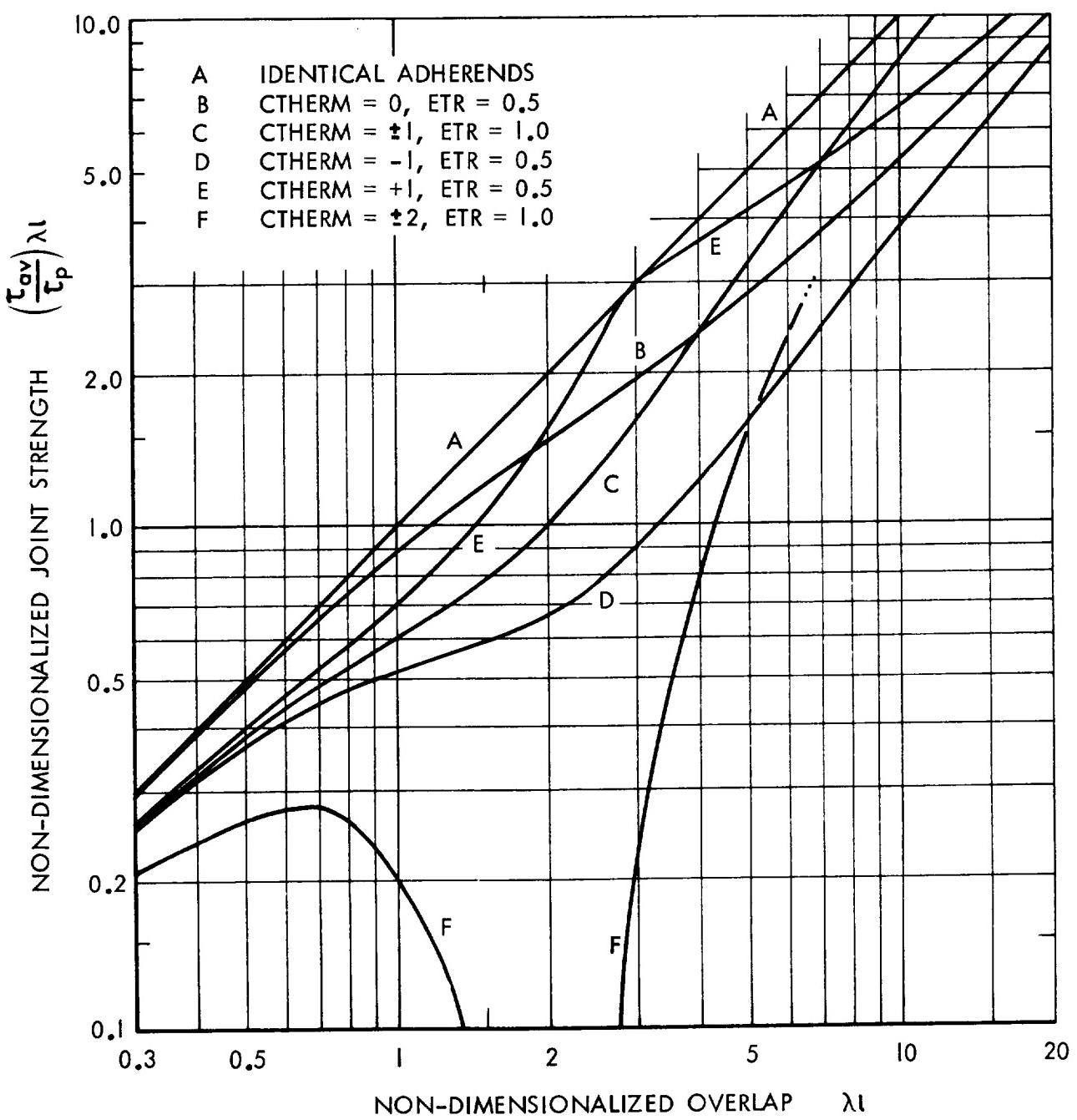


FIGURE 5. INTERACTION OF ADHEREND STIFFNESS AND THERMAL IMBALANCES FOR ELASTIC BONDED SCARF JOINTS

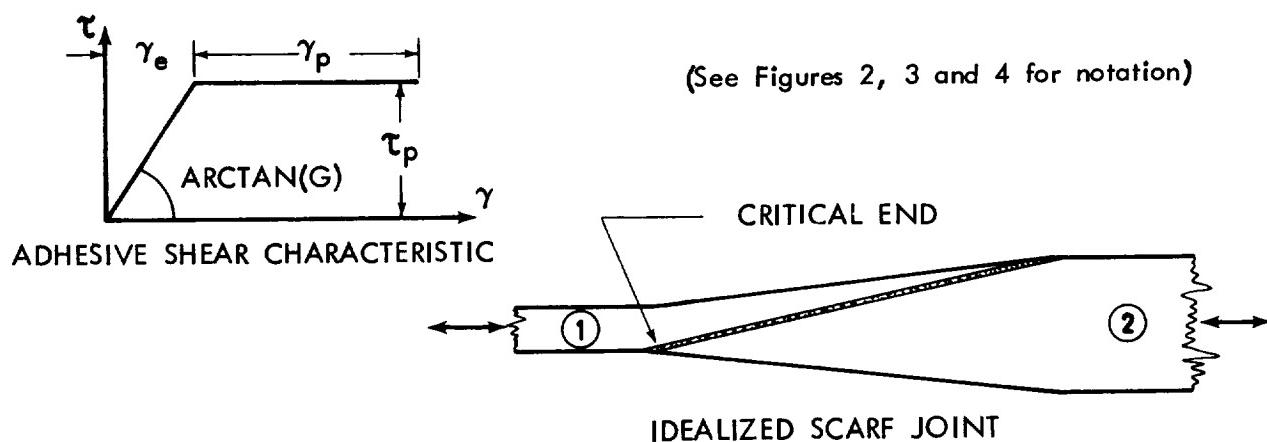
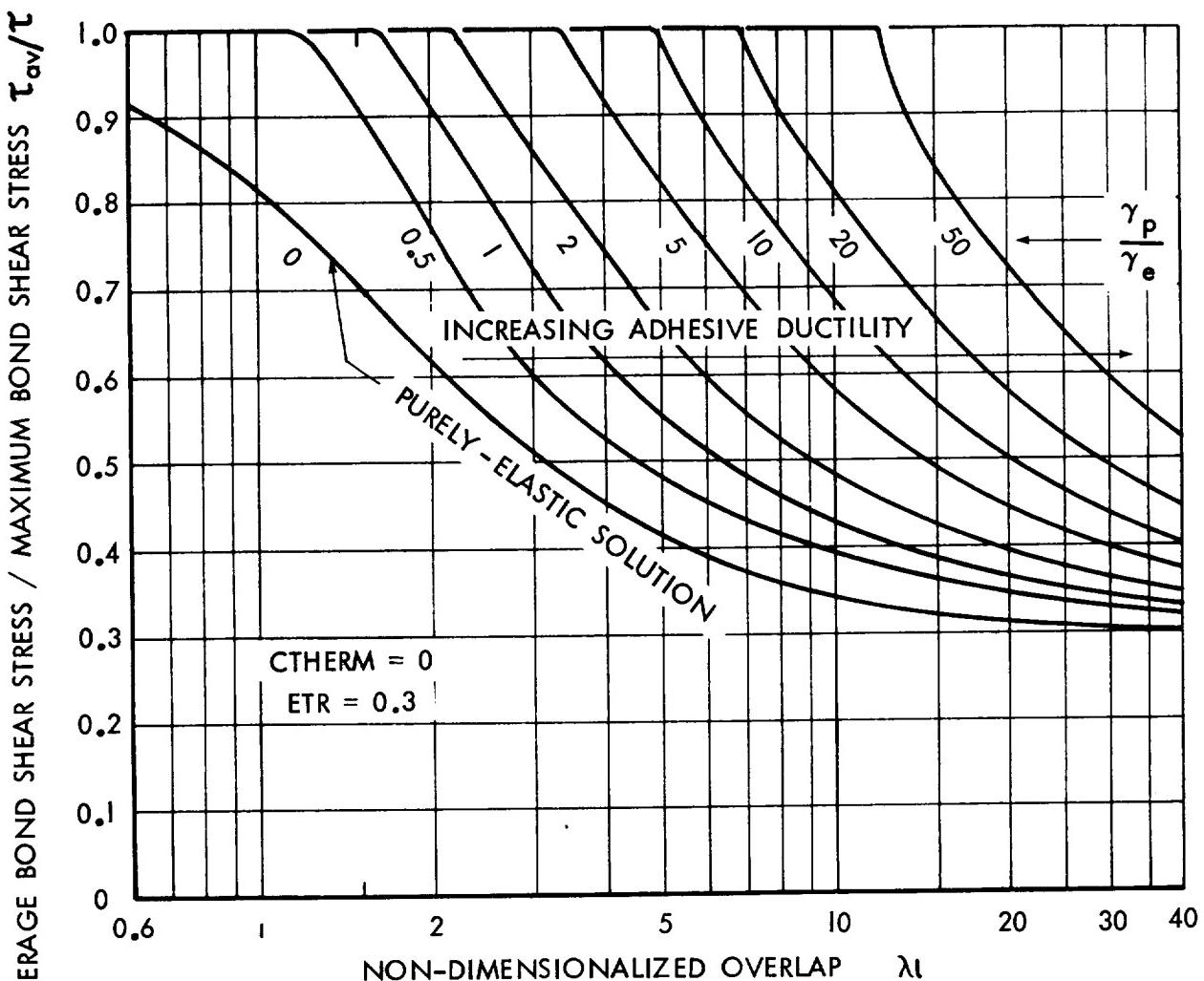
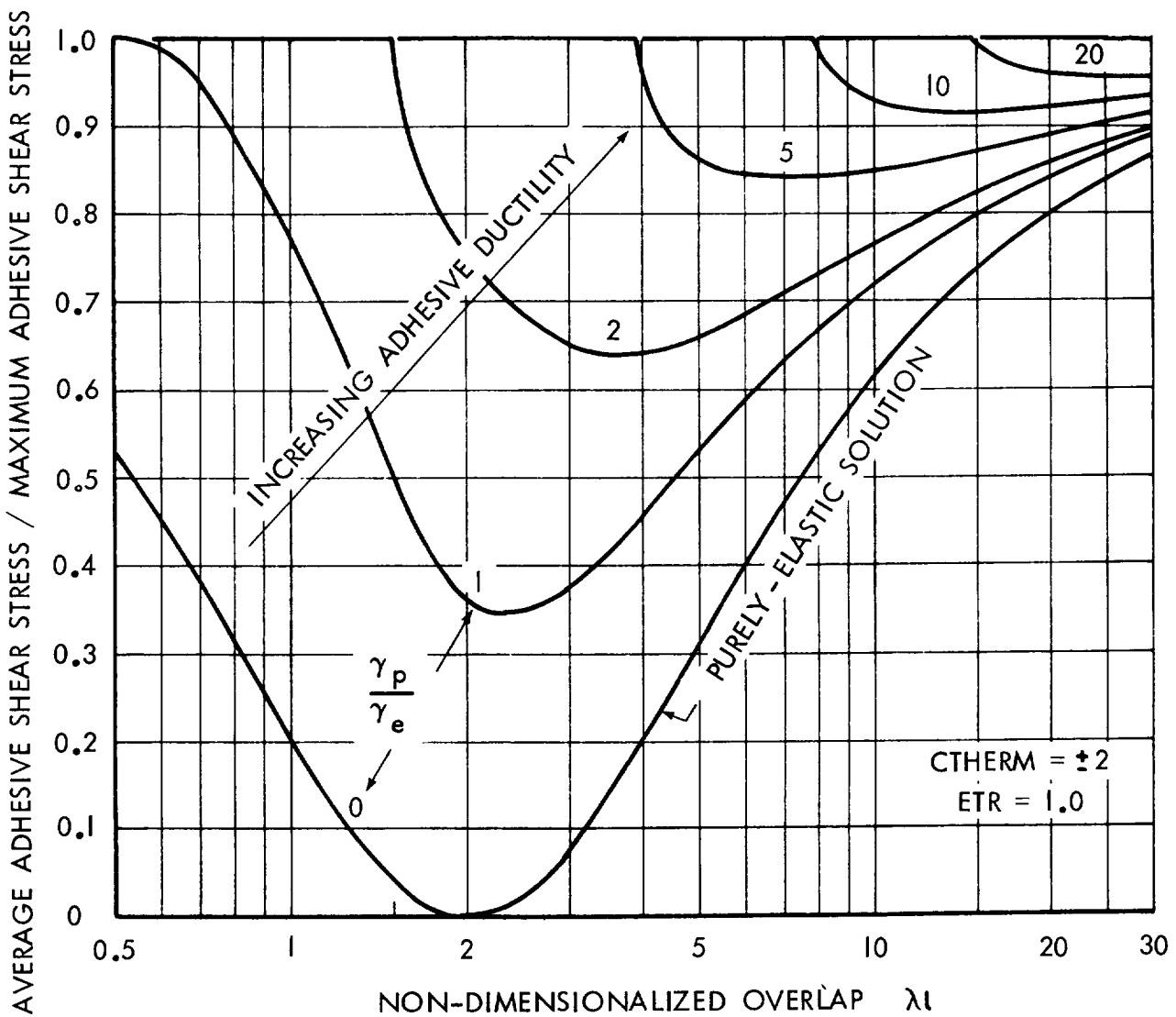
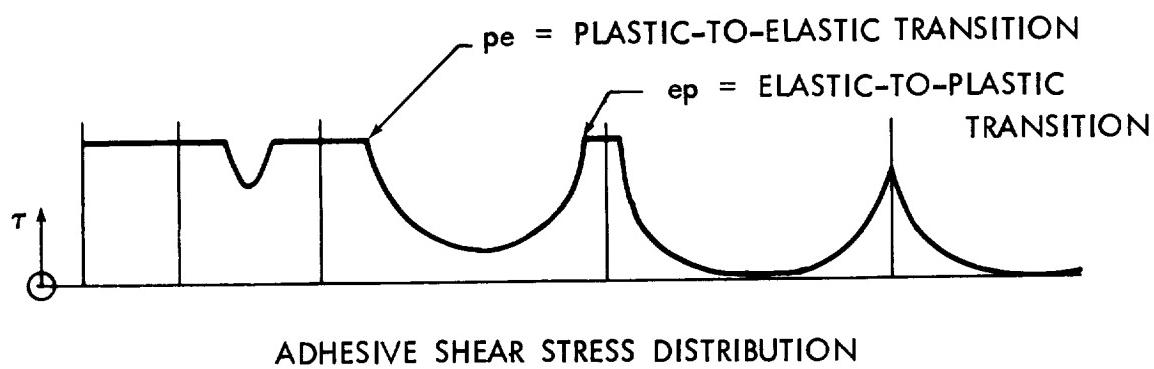
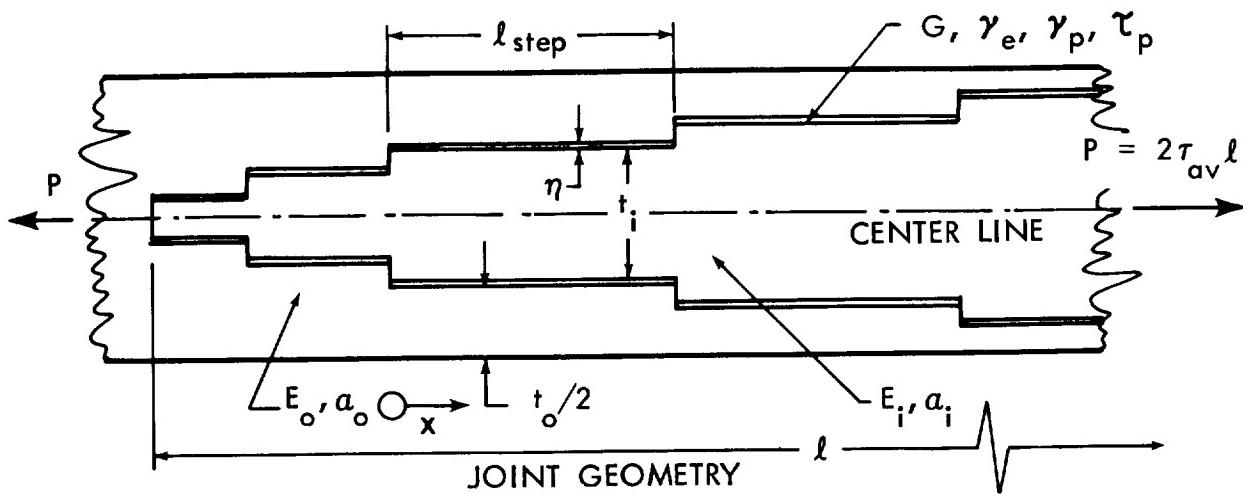


FIGURE 6. EFFECT OF ADHESIVE PLASTICITY IN REDUCING STRENGTH LOSS DUE TO ADHEREND STIFFNESS IMBALANCE FOR BONDED SCARF JOINTS

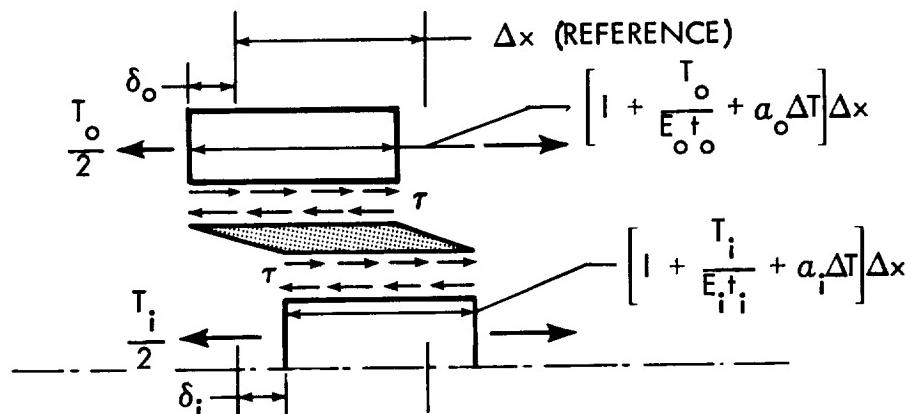


(See Figures 2, 3 and 4 for notation)

FIGURE 7. EFFECT OF ADHESIVE PLASTICITY IN REDUCING STRENGTH LOSS DUE TO ADHEREND THERMAL MISMATCH FOR BONDED SCARF JOINTS

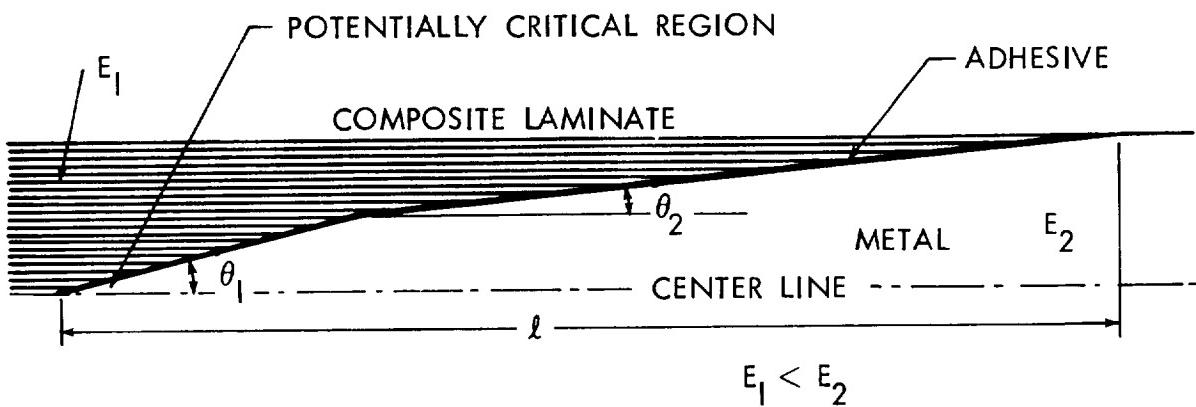


ADHESIVE SHEAR STRESS DISTRIBUTION

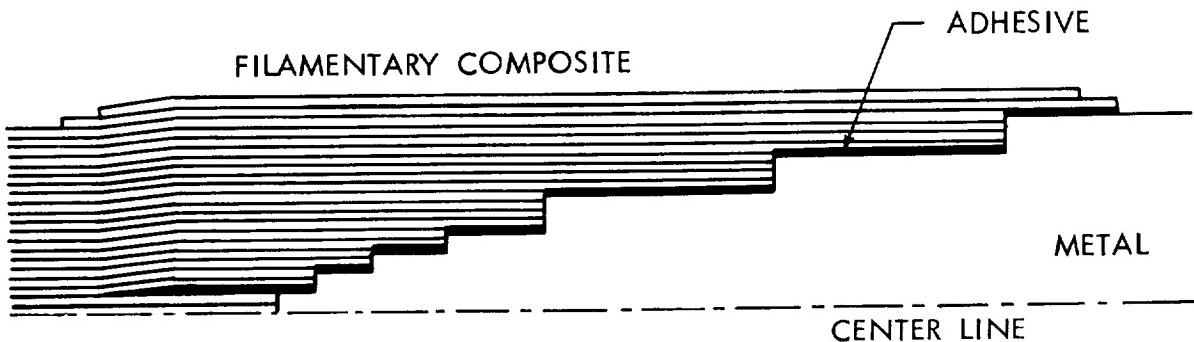


DISPLACEMENTS AND ELEMENT LOADS

FIGURE 8. NOTATION AND GEOMETRY FOR ADHESIVE-BONDED STEPPED-LAP JOINT ANALYSIS



(A) . IDEALIZED DUAL-SLOPE SCARF JOINT  
TO PROTECT TIP OF STIFFER ADHEREND



(B) . REPRESENTATION OF OPTIMIZED STEP PROPORTIONS  
FOR STEPPED-LAP JOINT WITH MODULUS OF COMPOSITE  
LAMINATE LESS THAN THAT OF METAL ADHEREND

FIGURE 9. PRACTICAL PROPORTIONING OF STEPPED-LAP JOINTS TO PROTECT  
AGAINST FATIGUE FAILURES AT TIP OF METAL ADHEREND

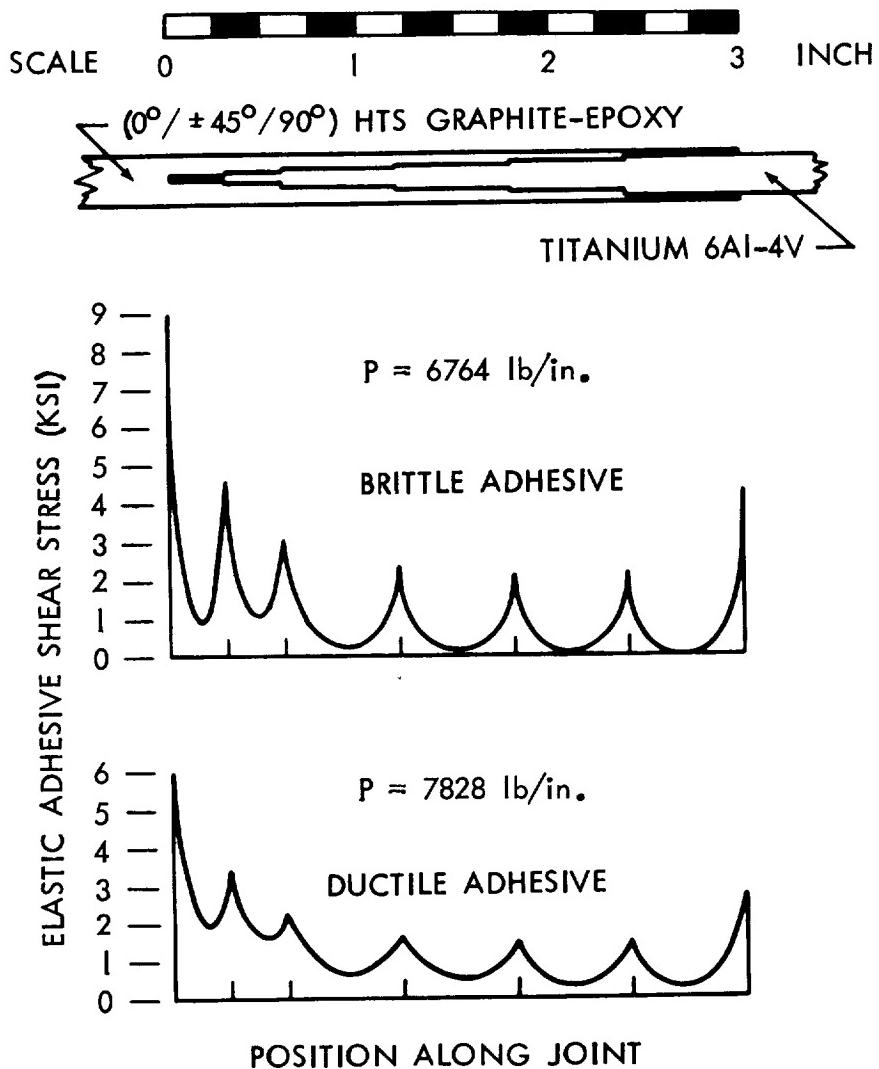


FIGURE 10. ELASTIC SHEAR STRESS DISTRIBUTIONS FOR BRITTLE AND DUCTILE ADHESIVES IN BONDED STEPPED-LAP JOINTS

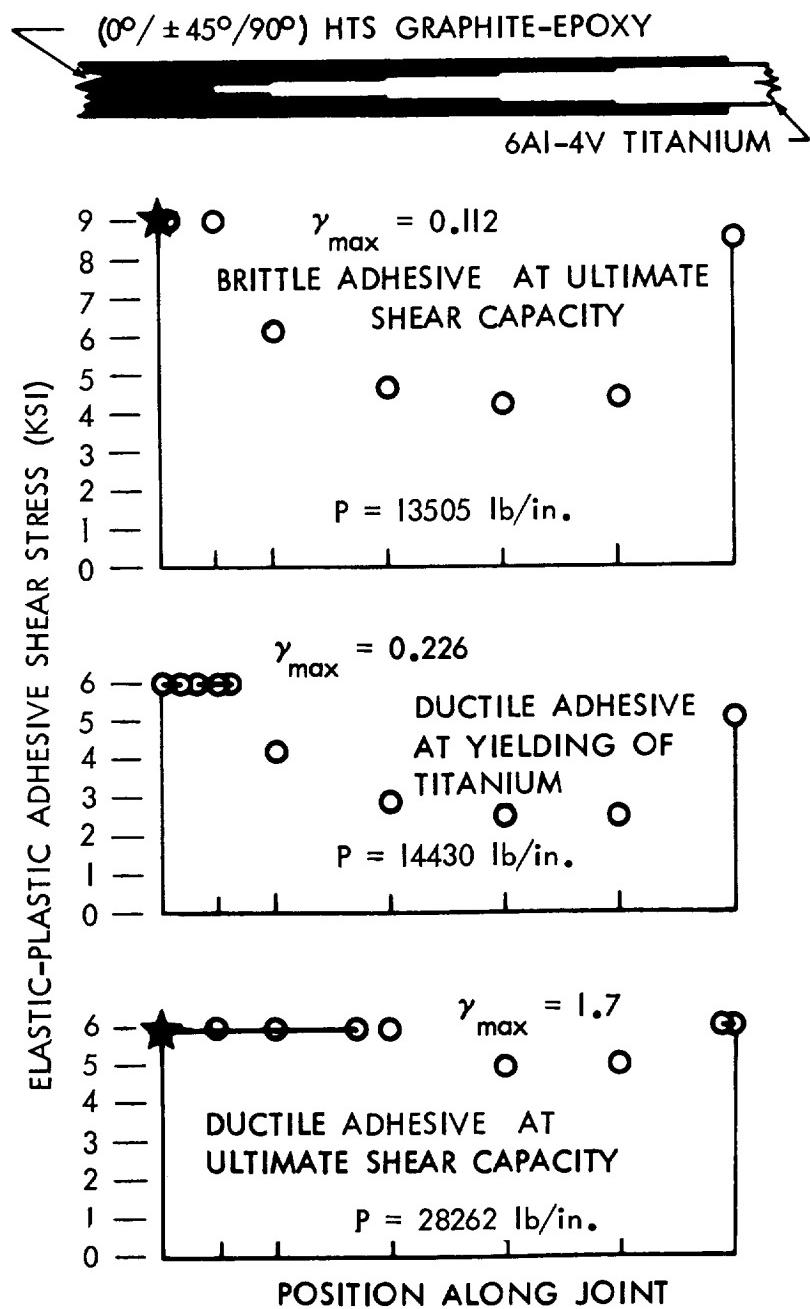


FIGURE II. ELASTIC-PLASTIC SHEAR STRESS DISTRIBUTIONS FOR BRITTLE AND DUCTILE ADHESIVES IN BONDED STEPPED-LAP JOINTS

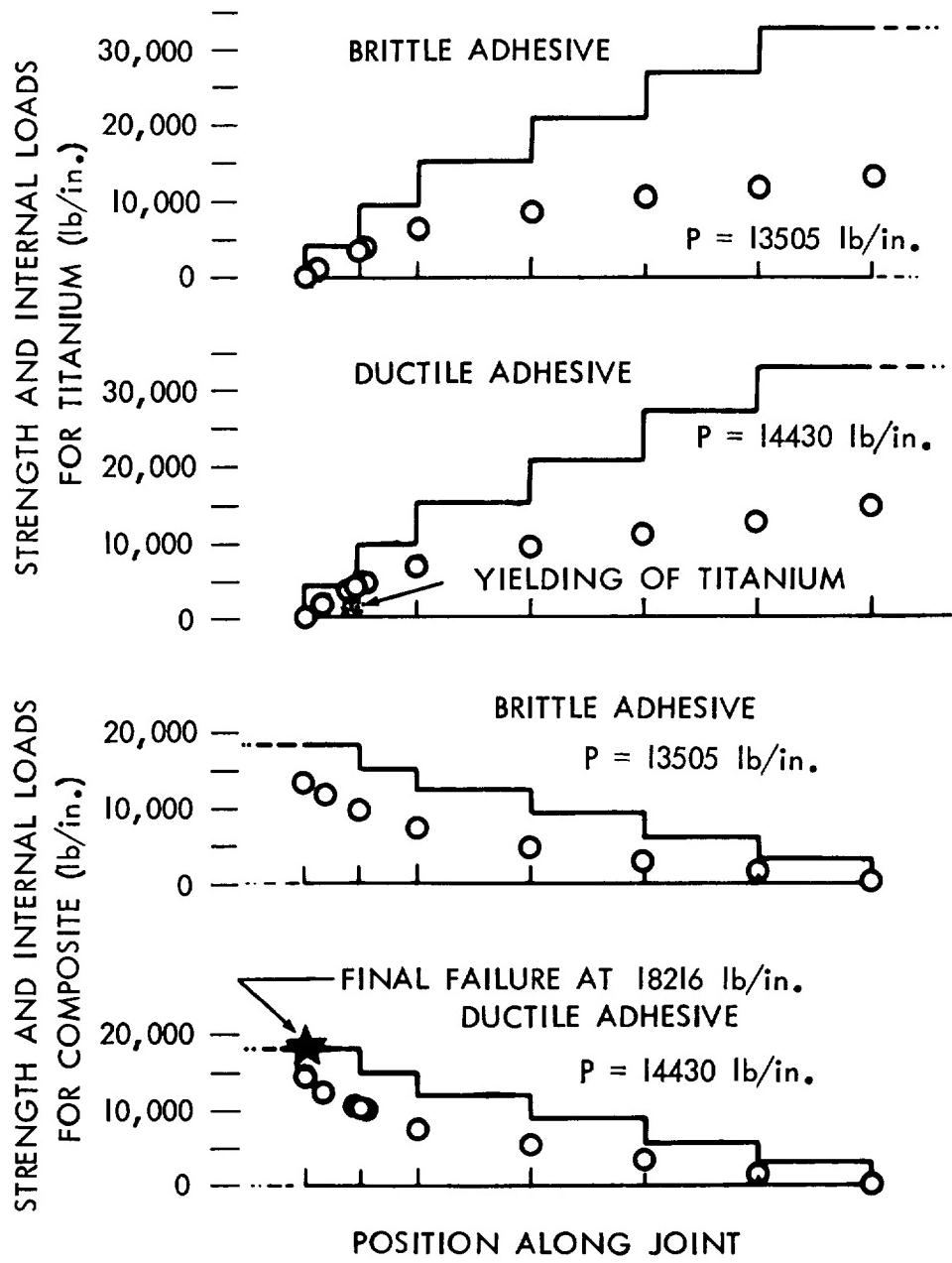


FIGURE 12. ADHEREND STRENGTHS AND INTERNAL LOADS FOR BONDED STEPPED-LAP JOINTS

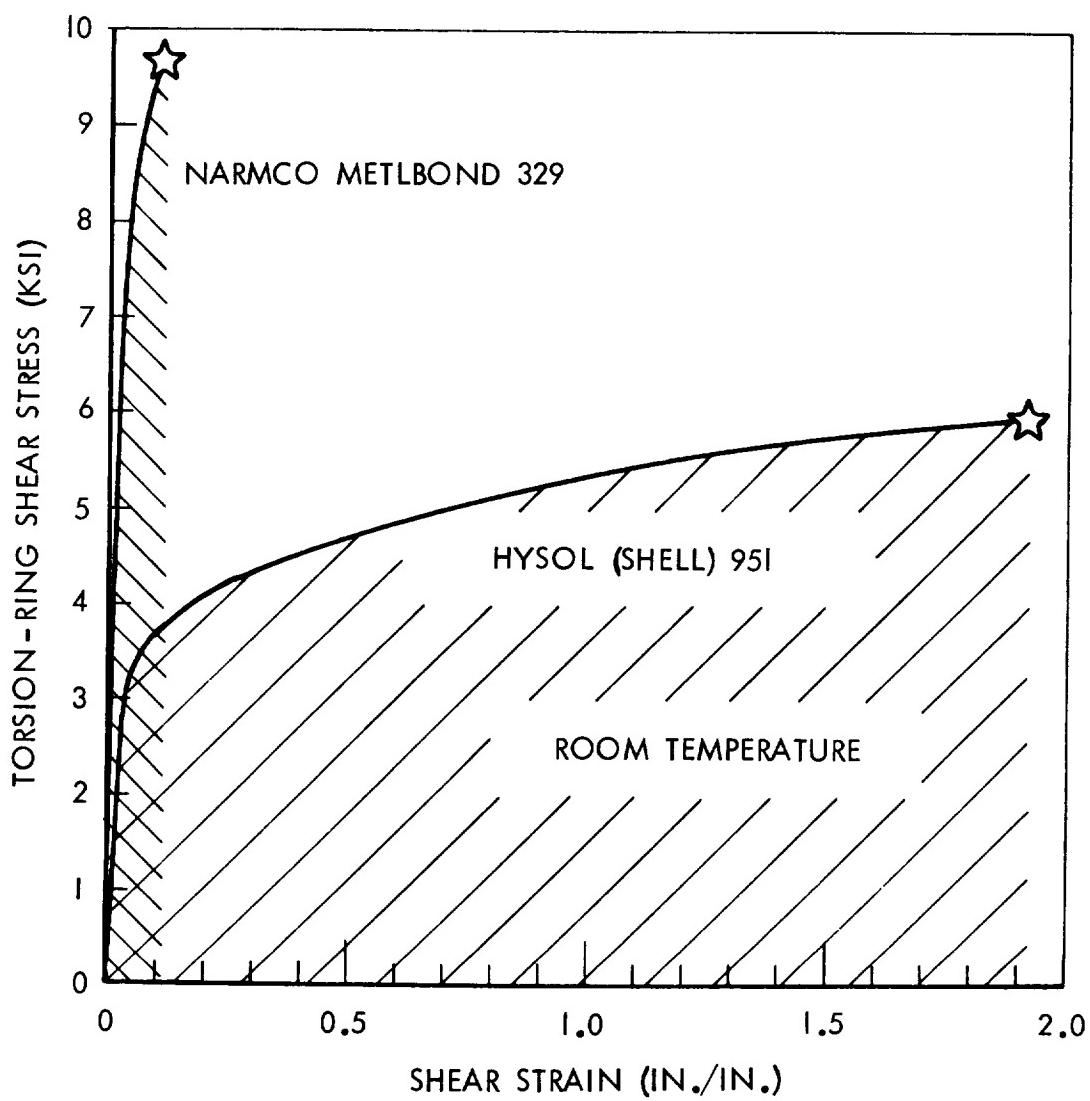


FIGURE I3. COMPARISON OF SHEAR STRESS-STRAIN CHARACTERISTICS FOR BRITTLE AND DUCTILE ADHESIVES

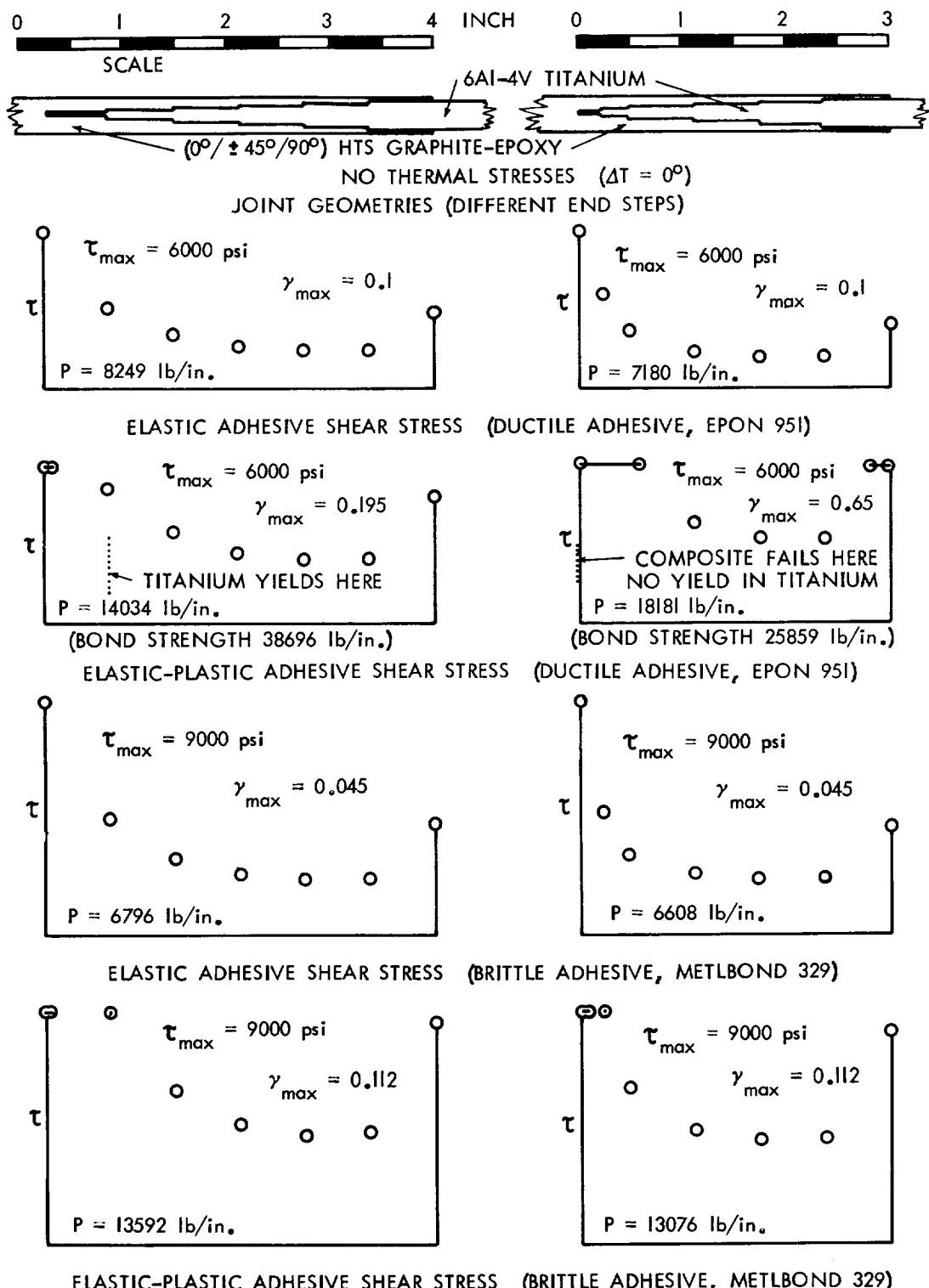
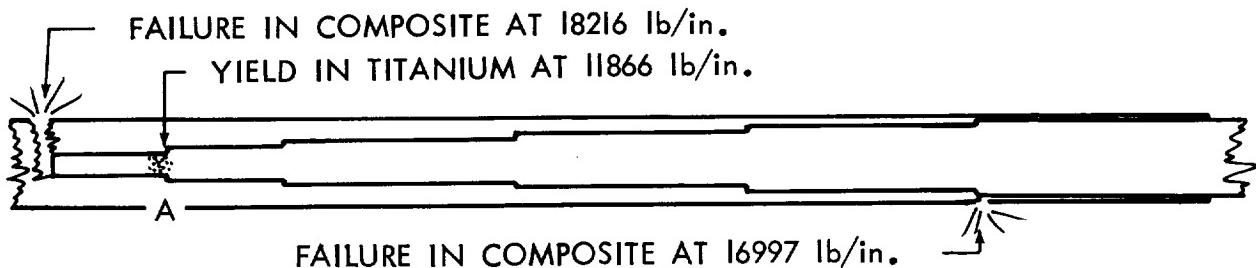


FIGURE 14. COMPARISON BETWEEN STEPPED-LAP JOINTS WITH UNIFORM STEP LENGTHS AND WITH OPTIMIZED STEP LENGTHS

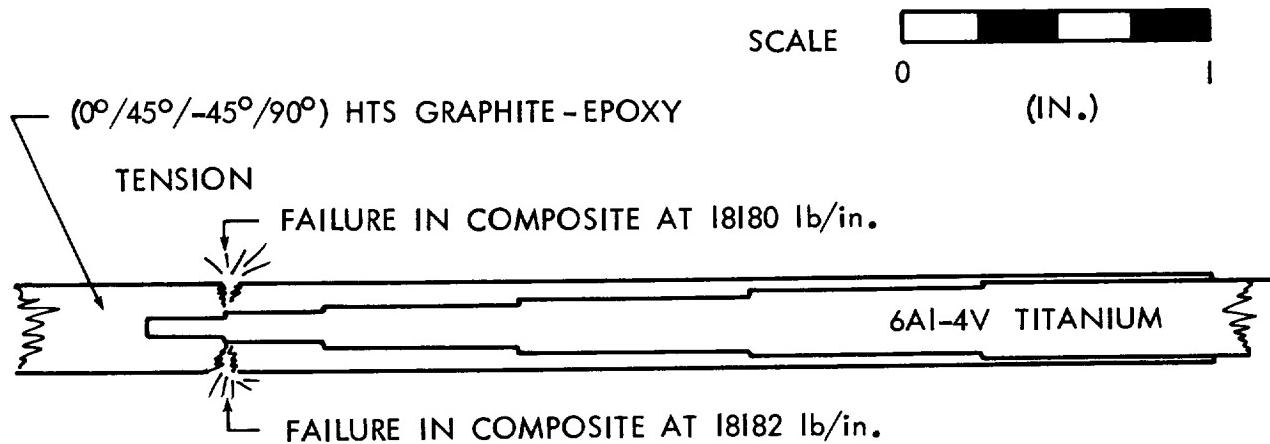
## TENSION



## COMPRESSION

NOTE THAT TITANIUM END STEPS WERE ALREADY SHORTENED DURING PRELIMINARY DESIGN. WITH UNIFORM STEPS 0.75 IN. LONG THROUGHOUT, PREMATURE FATIGUE FAILURE WOULD OCCUR AT A, FOLLOWED BY FAILURE OF COMPOSITE AT THE SAME (REDUCED) SECTION.

### (A) PRELIMINARY DESIGN



## COMPRESSION

NO YIELDING OF TITANIUM

### (B) OPTIMIZED DESIGN

DUCTILE ADHESIVE CURED AT 350°F.  
STRENGTHS CALCULATED AT ROOM TEMPERATURE.  
STRENGTH OF COMPOSITE ADHEREND OUTSIDE JOINT = 18216 lb/in.  
POTENTIAL BOND SHEAR STRENGTH WOULD EXCEED 23257 lb/in. IN EVERY CASE SHOWN IF ADHERENDS WERE SUFFICIENTLY STRONG.

FIGURE 15. OPTIMIZATION OF DETAILS IN STEPPED-LAP BONDED JOINTS

## APPENDICES

### A.1 Computer Program A4EC For Elastic Strength of Bonded Scarf Joints

The FORTRAN IV digital computer program associated with the analysis in Section 2 is listed below. This program has been checked out thoroughly and sample solutions are illustrated in Section 4. Only shear stresses are considered, with the peel stresses neglected in accordance with the very small scarf angles used in practice. As discussed in Section 2, there are severe convergence problems associated with the series solution to this problem. While the average shear stresses computed are considered very reliable, no computation sequence for the stress distribution was found which was considered sufficiently accurate over the far end of the joint ( $x/l \approx 1$ ). The peak shear stress is located correctly by program A4EC at one end or other of the overlap. The only real need for a shear stress distribution is as an intermediate step in the computation of the internal adherend stresses. Since the convergence of the series was enhanced greatly by prior integration into the contributions to the average shear stress, it is recommended that any attempt to pursue the adherend stress distribution should proceed along similar lines. The adhesive shear stress distribution series can be integrated mathematically so that a more tractable series solution is obtained for the adherend stresses. The first two terms follow from the average shear stress solution and the subsequent ones would derive from recurrence formulae. The condition under which a need for such information could arise is the possible breaking off of the thin tip of the stiffer adherend for a very small scarf angle. Such a situation is unlikely for perfectly elastic adhesives because the shear stress drops off very rapidly away from the ends. A simpler procedure is available for the elastic-plastic adhesive.

The format of the input data necessary to operate the A4EC computer program is as follows:

CARD 1:

FORMAT (415)

IMAX = Number of thermal mismatch coefficients. IMAX .LE. 20.

JMAX = Number of non-dimensionalized overlaps. JMAX .LE. 40.

(Note that this is one more than the number of overlaps to be read in. The limiting case of OL(1)=0 is set by the program.)

KMAX = Number of adherend stiffness imbalances. KMAX .LE. 10.

(Note that this controls the number of columns of answers printed across the page and cannot be increased indefinitely.)

NMAX = Number of terms in power series. 10 .LE. NMAX .LE. 50.

(Note NMAX = 20 is recommended.)

CARDS 2, 2A, 2B, etc.:

FORMAT (12F6.2)

OL(J)= Non-dimensionalized overlaps. Number restricted to 40 by dimension statement. (Note that OL(J) must be read in in ascending order and that OL(2), which is the first entry on card 2, must not exceed 0.5 because of internal computations. OL(1) = 0 is set by program as limiting case.) Values of OL(J) exceeding 50 are impractically large.

CARDS 3, 3A, 3B, etc.:

FORMAT (10F5.2)

ETR(K)=Adherend stiffness ratios  $(E_1 t_1)/(E_2 t_2)$ . Number restricted to 10 by dimension statement. (Subscripts 1 and 2 must be identified so that 0 .LT. ETR .LE. 1. Array should be read in in ascending or descending order.)

CARDS 4, 4A, 4B, etc.:

FORMAT (10F7.3)

CTHERM(I) = Adherend thermal mismatch coefficients in non-dimensionalized form. Number restricted to 20 by dimension statement. (Note that equal and opposite values must be read in consecutively to account for the difference between tensile and compressive application of the shear load. Values of up to  $\pm 5$  are sufficient for the available range of adhesives. Greater values are usually associated with failure of the joint under residual thermal stresses alone.)

The complete listing follows, along with sample output pages. The output tables come in pairs with the ratio of the average to maximum adhesive shear stress ( $\tau_{av}/\tau_p$ ) and the non-dimensionalized joint strength ( $\tau_{av}/\tau_p$ )( $\lambda l$ ) as functions of the adherend extensional stiffness ratio  $ETR = E_1 t_1 / E_2 t_2 \leq 1$  horizontally and the non-dimensionalized joint overlap  $\lambda l = \sqrt{\frac{G}{n} \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right) l^2}$  vertically.

Each table is prepared for a single value of thermal mismatch coefficient  $(\alpha_2 - \alpha_1) \Delta T \lambda$

$$CTHERM = \frac{(\alpha_2 - \alpha_1) \Delta T \lambda}{\tau_p \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right)}$$

and equal and opposite values are treated in

turn to cover both tensile and compressive shear loadings.

```

CDECK A4FC
C ELASTIC ANALYSIS OF UNBALANCED SCARF JOINTS
C NON-DIMENSIONALIZED FORMULATION
C AVERAGE STRESSES
C STIFFNESS AND THERMAL IMBALANCES ACCOUNTED FOR
C DATA PRESENTATION FOR TENSILE SHEAR LOADING
C CHANGE SIGN OF CTHERM TO USE FOR COMPRESSIVE SHEAR LOADS
C DIMENSION DL(J), ETR(K), CTHERM(I), A(N,2), TRATIO(N,2),
C   I TAUAVG(J,K), ICRTND(J,K), STRGTH(J,K), SIG(N,2),
C     DIMENSION DL(40), ETR(10), CTHERM(20), A(50,2),
C       1 TRATIO(50,2), TAUAVG(40,10), TCRTND(40,10), STRGTH(40,10)      SIG(50,2),
C READ IN ARRAY SIZES
C   READ (5,10) IMAX, JMAX, KMAX, NMAX
C     10 FORMAT (4I5)
C   IMAX .LE. 20, JMAX .LE. 40, KMAX .LE. 10, NMAX .LE. 50 .AND. .GE. 10
C READ IN NON-DIMENSIONALIZED OVERLAP ARRAY
C   DL(1) = 0.
C DL(J) MUST BE .LE. 0.5 BECAUSE OF A SUBSEQUENT TREND CHECK
C DL(J) MUST BE IN ASCENDING ORDER
C   READ (5,20) (DL(J), J = 2, JMAX)
C     20 FORMAT (12F6.2)
C READ IN STIFFNESS IMBALANCE ARRAY
C IDENTIFY ADHERENDS 1 AND 2 SUCH THAT ETR(K) = (ET)1/(ET)2 .LE. 1.
C   READ (5,30) (ETR(K), K = 1, KMAX)
C     30 FORMAT (10F5.2)
C READ IN NON-DIMENSIONALIZED THERMAL MISMATCH COEFFICIENTS
C   READ (5,40) (CTHERM(I), I = 1, IMAX)
C     40 FORMAT (10F7.3)
C PRINT OUT INPUT DATA
C   WRITE (6,50) IMAX, JMAX, KMAX, NMAX
C     50 FORMAT (1H1, 9H JMAX, KMAX, NMAX = , 12, 9H JMAX = , 12, 9H KMAX = , 12,
C       1 9H NMAX = , 12)
C   WRITE (6,60) (DL(J), J = 1, JMAX)
C     60 FORMAT (10H OVERLAPS/, 12F6.2)
C   WRITE (6,70) (ETR(K), K = 1, KMAX)
C     70 FORMAT (22H STIFFNESS IMBALANCES/, 10F5.2)
C   WRITE (6,80) (CTHERM(I), I = 1, IMAX)
C     80 FORMAT (20H THERMAL MISMATCHES/, 10F7.3)
C SET UNIFORM STRESS FOR ZERO OVERLAP
C DO 90 K = 1, KMAX
C   TAUAVG(1,K) = 1.
C   90 STRGTH(1,K) = 0.
C START OF COMPUTATION DO LOOPS
C DO 310 I = 1, IMAX
C   DO 180 K = 1, KMAX
C     DO 180 J = 2, JMAX
C ESTABLISH ADHEREND 1 END OF JOINT AS REFERENCE
C SUBSEQUENTLY CHECK WHETHER ADHEREND 1 END OR ADHEREND 2 END IS CRITICAL
C NCRTND = 1
C THERMC = CTHERM(I)
C VR = ETR(K)
C IF ((VR .NE. 1.) .OR. (THERMC .NE. 0.)) GO TO 100
C SET UNIFORM STRESS FOR BALANCED JOINTS
C   TAUAVG(J,K) = 1.
C   STRGTH(J,K) = DL(J)
C   ICRTND(J,K) = 0
C   GO TO 180
C 100 V1 = 1. + VR
C   V2 = VR / V1
C   V3 = (1. - VR) / V1
C   OLAP = DL(J)
C   OLAP2 = OLAP * OLAP
C COMPUTE INITIAL TERMS OF SERIES, ASSUMING A(1,1)=A(2,2)=1. & A(1,2)=A(2,1)=0.
C   A(1,1) = 1.
C   A(2,1) = 0.
C   A(3,1) = (OLAP/6.) * (-THERMC + V3*OLAP)
C   A(1,2) = 0.
C   A(2,2) = 0.5
C   A(3,2) = (OLAP2 / 6.) * (V2/2.) + 1./6.
C COMPUTE NMAX TERMS OF AVERAGE STRESS POWER SERIES
C DO 110 N = 4, NMAX
C   DO 110 M = 1, 2
C     110 A(N,M) = 1. ((N-2)*(N-1) + OLAP2 * V2) * A(N-1,M)
C       + V3 * OLAP2 * A(N-2,M) ) / (N*(N-1))
C COMPUTE A2 THROUGH RAPID CONVERGENCE OF AVERAGE STRESS
C NOTE THAT INDIVIDUAL TERMS IN DISTRIBUTION DO NOT CONVERGE AS RAPIDLY
C   SIG(3,1) = 1. + A(3,1)
C   SIG(3,2) = 0.5 + A(3,2)
C   DO 120 N = 4, NMAX
C     SIG(N,1) = SIG(N-1,1) + A(N,1)
C   120 SIG(N,2) = SIG(N-1,2) + A(N,2)
C COMPUTE A2(NMAX)
C   A2SAVE = (THERMC/OLAP + V2 - (SIG(NMAX,1)/V1) ) /
C     1. ( (SIG(NMAX,2)/V1) + (1./OLAP2) )
C COMPUTE AVERAGE SHEAR STRESS IN BOND
C   TRATIO(J,NCRTND) = V1 * (V2 + THERMC/OLAP - A2SAVE/OLAP2)
C CHECK WHICH END OF JOINT IS CRITICAL
C IF (NCRTND .EQ. 2) GO TO 130

```

```

IF (THFRMC .LT. 0.) GO TO 130                                A4EC0890
C IF ADHEREND 2 END IS CRITICAL INTERCHANGE 1 AND 2 AND RECOMPUTE   A4EC0900
  NCRTND = 2                                                 A4EC0910
  VR = 1. / VR                                              A4EC0920
  THERMC = - THERMC                                         A4EC0930
  GO TO 100                                                 A4EC0940
100 CONTINUE
C IDENTIFY AVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS & CRITICAL END    A4EC0950
C NOTE THAT INITIAL SELECTION CRITERIA ASSUME ONE END OF JOINT OR    A4EC0960
C 1 OTHER IS CRITICAL AND PRECLUDE POSSIBILITY OF MAXIMUM STRESS IN    A4EC0970
C 2 MIDDLE. SUBSEQUENT SEPARATE CHECK ON THIS CONDITION.               A4EC0980
  IF (CTHERM(I) .GT. 0.) GO TO 140                               A4EC0990
C BOTH IMBALANCES WILL INEVITABLY COMPOUND FOR CTHERM(I) .LT. 0.      A4EC1000
  1 SINCE ETR(K) .LE. 1.. HENCE NCRTND = 1                      A4EC1010
  ICRTND(J,K) = 1                                               A4EC1020
  IF (TRATIO(J,1) .GE. 0.) GO TO 160                           A4EC1030
  IF (TRATIO(J,1) .LT. 0.) GO TO 150                           A4EC1040
130 CONTINUE
C IDENTIFY MORE POWERFUL IMBALANCE FOR NULLIFYING BEHAVIOUR (NCRTND = 1) A4EC1050
  1 CTHERM(I) .GT. 0. AND. ETR(K) .LE. 1.                      A4EC1060
C COVER SITUATION WHERE THERMAL IMBALANCE DOMINATES OVER STIFFNESS     A4EC1070
  1 IMBALANCE. NOTE NEED DL(2) .LE. 0.5 FOR THIS CHECK.          A4EC1080
  ICRTND(J,K) = 2                                               A4EC1090
  IF ((TRATIO(2,1).GT.1.).AND.(TRATIO(J,2).LT.0.))GO TO 150       A4EC1100
  IF ((TRATIO(2,1).GT.1.).AND.(TRATIO(J,2).LE.1.))GO TO 170       A4EC1110
C CHECK IF TWO IMBALANCES PRECISELY CANCEL                     A4EC1120
  ICRTND(J,K) = 0                                               A4EC1130
  IF ((TRATIO(J,1) .EQ. 1.) .AND. (TRATIO(J,2) .EQ. 1.)) GO TO 160   A4EC1140
C CHECK IF STIFFNESS IMBALANCE DOMINATES                      A4EC1150
  ICRTND(J,K) = 1                                               A4EC1160
  IF ((TRATIO(J,1) .LE. 1.) .AND. (TRATIO(J,2) .GE. 0.) .AND.     A4EC1170
    1 (TRATIO(J,2) .GT. 1.)) GO TO 160                         A4EC1180
C ALL POSSIBILITIES FOR EITHER END CRITICAL CHECKED OUT        A4EC1190
C ONLY POSSIBILITY REMAINING IS THAT TAUMAX IS IN MIDDLE OF JOINT    A4EC1200
C NOTE THAT THIS PHENOMENON ARISES ONLY FOR JOINTS BROKEN WITHOUT LOAD A4EC1210
C 1 WHEN THE LOAD IN THE OPPOSITE SENSE IS EXAMINED            A4EC1220
C COMBINATION OF SEVERE THERMAL MISMATCH AND EXCESSIVE LENGTH IS NECESSARY A4EC1230
C IDENTIFY FAILURE CASES BY ASTERISKS                         A4EC1240
  TAUAVG(J,K) = 100.                                           A4EC1250
  STRGTH(J,K) = 1000.                                         A4EC1260
  ICRTND(J,K) = 10                                             A4EC1270
  GO TO 180                                                 A4EC1280
C ZERO STRENGTH ATTAINED                                     A4EC1290
150 TAUAVG(J,K) = 0.                                         A4EC1300
  STRGTH(J,K) = 0.                                         A4EC1310
  GO TO 180                                                 A4EC1320
C ADHEREND 1 END OF JOINT CRITICAL                         A4EC1330
160 TAUAVG(J,K) = TRATIO(J,1)                                A4EC1340
  STRGTH(J,K) = TAUAVG(J,K) * DL(J)                          A4EC1350
  GO TO 180                                                 A4EC1360
C ADHEREND 2 END OF JOINT CRITICAL                         A4EC1370
170 TAUAVG(J,K) = TRATIO(J,2)                                A4EC1380
  STRGTH(J,K) = TAUAVG(J,K) * DL(J)                          A4EC1390
180 CONTINUE
C IDENTIFY CRITICAL END OF JOINT FOR ZERO OVERLAP           A4EC1400
  DO 190 K = 1, KMAX                                         A4EC1410
  190 ICRTND(1,K) = ICRTND(2,K)                            A4EC1420
C HENCE NEED DL(2) .LE. 0.2                                  A4EC1430
  IF (CTHERM(I) .NE. 0.) GO TO 210                         A4EC1440
C PRINT OUT SPECIAL HEADING FOR ZERO THERMAL MISMATCH BETWEEN ADHERENDS A4EC1450
  WRITE (6,200) (ETR(K), K = 1, KMAX)                      A4EC1460
200 FORMAT (1H1,10(/),31X, 48HADHESIVE-BONDED SCARF JOINTS (ELASTIC AA4EC1500
  1NALYSIS),, 39X, 31HNON-DIMENSIONALIZED FORMULATION//,          A4EC1510
  2 38X, 33HZERO THERMAL MISMATCH COEFFICIENT//,                A4EC1520
  3 68X, 28H0 = BOTH ENDS EQUALLY LOADED/, 20X, 72HAVERAGE SHEAR STRA4EC1530
  4ESS / MAXIMUM SHEAR STRESS , 1 = SOFT ET END CRITICAL/,     A4EC1540
  5 68X, 25H2 = STIFF ET END CRITICAL//,                      A4EC1550
  6 8H SCALED, 31X, 39HEXTENSIONAL STIFFNESS (THICKNESS) RATIO/, A4EC1560
  7 7H L/T/, 7H RATIO/, 1H+, 4X, 10F10.1//)                  A4EC1570
  GO TO 230                                                 A4EC1580
210 THERMC = - CTHERM(I)                                    A4EC1590
C PRINT OUT HEADING                                         A4EC1600
  WRITE (6,220) CTHERM(I), THERMC, (ETR(K), K = 1, KMAX)      A4EC1610
220 FORMAT (1H1,10(/),31X, 48HADHESIVE-BONDED SCARF JOINTS (ELASTIC AA4EC1620
  1NALYSIS),, 39X, 31HNON-DIMENSIONALIZED FORMULATION//, 17X, 31HTHERA4EC1630
  2MAL MISMATCH COEFFICIENT = , F6.3, 17H FOR TENSION, = ,F6.3, 16H A4EC1640
  3FOR COMPRESSION//,68X, 28H0 = BOTH ENDS EQUALLY LOADED/,      A4EC1650
  4 20X, 72HAVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS ,      A4EC1660
  51 = SOFT ET END CRITICAL/, 68X, 25H2 = STIFF ET END CRITICAL//, A4EC1670
  6 8H SCALED, 31X, 39HEXTENSIONAL STIFFNESS (THICKNESS) RATIO/, A4EC1680
  7 7H L/T/, 7H RATIO/, 1H+, 4X, 10F10.1//)                  A4EC1690
230 CONTINUE
C PRINT OUT TABULATIONS OF AVERAGE BOND STRESSES           A4EC1700
  DO 250 J = 1, JMAX                                         A4EC1710
  WRITE (6,240) DL(J), ((TAUAVG(J,K),ICRTND(J,K)), K = 1, KMAX) A4EC1720
240 FORMAT (1H , F6.2, 2X, 10(F7.5, 1X, 1X))                 A4EC1730
250 CONTINUE
  IF (CTHERM(I) .NE. 0.) GO TO 270                         A4EC1740
                                                A4EC1750
                                                A4EC1760

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C PRINT OUT SPECIAL HEADING FOR ZERO THERMAL MISMATCH BETWEEN ADHERENDS A4EC1770
  WRITE (6,260) (ETR(K), K = 1, KMAX) A4EC1780
260 FORMAT (1H1,10(/),31X, 48ADHESIVE-BONDED SCARF JOINTS (ELASTIC AA4EC1790
1NALYSIS)/, 39X, 31HNON-DIMENSIONALIZED FORMULATION//, A4EC1800
2 38X, 33HZERO THERMAL MISMATCH COEFFICIENT//, A4EC1810
3 68X, 28H0 = ROTH ENDS EQUALLY LOADED/, 20X, 72HNON-DIMENSIONALIZA4EC1820
4ED STRENGTH , 1 = SOFT ET END CRITICAL/, A4EC1830
5 68X, 25H2 = STIFF ET END CRITICAL//, A4EC1840
6 9H SCALED, 31X, 39HEXTENSINAL STIFFNESS (THICKNESS) RATIO/, A4EC1850
7 7H L/T/, 7H RATIO/, 1H+, 4X, 10F10.1//) A4EC1860
GO TO 290 A4EC1870
C PRINT OUT HEADINGS A4EC1880
270 WRITE (6,280) CTHERM(I), THERMC, (ETR(K), K = 1, KMAX) A4EC1890
280 FORMAT (1H1,10(/),31X, 48ADHESIVE-BONDED SCARF JOINTS (ELASTIC AA4EC1900
1NALYSIS)/, 39X, 31HNON-DIMENSIONALIZED FORMULATION//, 17X, 31HTHERA4EC1910
2MAL MISMATCH COEFFICIENT = , F6.3, 17H FOR TENSION, = ,F6.3, 16H A4EC1920
3FOR COMPRESSION//,68X, 28H0 = ROTH ENDS EQUALLY LOADED/, A4EC1930
4 20X, 72HNON-DIMENSIONALIZED STRENGTH , A4EC1940
51 = SOFT ET END CRITICAL/, 68X, 25H2 = STIFF ET END CRITICAL//, A4EC1950
6 9H SCALED, 31X, 39HEXTENSINAL STIFFNESS (THICKNESS) RATIO/, A4EC1960
7 7H L/T/, 7H RATIO/, 1H+, 4X, 10F10.1/) A4EC1970
290 CONTINUE A4EC1980
C PRINT OUT TABULATIONS OF NON-DIMENSIONALIZED STRENGTHS A4EC1990
DO 310 J = 1, JMAX A4EC2000
  WRITE (6,300) OI(J), ((STRGTH(J,K), ICRTND(J,K)), K = 1, KMAX) A4EC2010
300 FORMAT (1H , F6.2, 2X, 10(F7.4, 1X, 1I, 1X)) A4EC2020
310 CONTINUE A4EC2030
  WRITE (6,320) A4EC2040
320 FORMAT (1H1, 18H PROGRAM COMPLETED/) A4EC2050
STOP A4EC2060
END A4EC2070

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**ADHESIVE-BONDED SCARF JOINTS (ELASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION**

THERMAL MISMATCH COEFFICIENT = 1.000 FOR TENSION, = -1.000 FOR COMPRESSION

SCALED L/T RATIO	EXTENSIONAL STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	0.0	2	0.0	2	0.0	2	0.0	2	0.0	2
0.20	0.1832	2	0.1826	2	0.1818	2	0.1814	2	0.1808	2
0.50	0.4245	2	0.4161	2	0.4093	2	0.4035	2	0.3987	2
1.00	0.9012	2	0.9311	2	0.77765	2	0.7347	2	0.7012	2
1.20	1.1853	2	1.0527	2	0.9576	2	0.8861	2	0.8304	2
1.50	1.3881	1	1.5030	2	1.3016	2	1.6151	2	1.5711	2
1.70	1.4858	1	1.6219	2	1.5915	2	1.3857	2	1.3727	2
2.00	1.6072	1	1.7784	1	1.9453	1	1.7899	2	1.5561	2
2.50	1.7635	1	1.9893	1	2.2144	1	2.4369	1	2.2136	2
3.00	1.8939	1	2.1613	1	2.4410	1	3.0000	2	2.5462	2
4.00	2.0664	1	2.4415	1	2.8424	2	3.2125	1	3.6046	2
5.00	2.2084	1	2.6771	1	3.1597	1	3.6525	1	4.1531	2
6.00	2.3299	1	2.8939	1	3.4722	1	4.0583	1	4.6748	2
8.00	2.5422	1	3.2882	1	4.0670	1	4.8679	1	5.6341	2
10.00	2.7354	1	3.5639	1	4.6469	1	5.6528	1	6.6766	1
12.00	2.9203	1	4.0444	1	5.2233	1	6.4342	1	7.6647	1
15.00	3.1914	1	4.6063	1	6.3890	1	7.6702	1	9.1663	1
17.00	3.3708	1	4.9923	1	6.6683	1	8.3915	1	10.1353	1
20.00	3.6401	1	5.5625	1	7.5412	1	9.5710	1	11.6208	1
25.00	4.0920	1	6.5035	1	9.0051	1	11.5442	1	14.1019	1
30.00	4.5493	1	7.4667	1	10.4775	1	13.5241	1	16.5877	1
35.00	5.0119	1	8.4371	1	11.0563	1	15.5087	1	19.0769	1
40.00	5.4792	1	9.4129	1	13.4390	1	17.4966	1	21.5684	1
45.00	5.9507	1	10.3930	1	14.9252	1	19.4869	1	24.0915	1
50.00	6.4257	1	11.3763	1	16.4138	2	21.4739	1	26.5559	1

# ADHESIVE-BONDED SCARF JOINTS (ELASTIC ANALYSIS) NON-DIMENSIONALIZED FORMULATION

Thermal mismatch coefficient = 1.000 for tension, = -1.000 for compression

AVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS , 0 : BOTH ENDS EQUALLY LOADED  
1 : SOFT ET END CRITICAL  
2 : STIFF ET END CRITICAL

SCALED L/T RATIO		EXTENSIONAL STIFFNESS (THICKNESS) RATIO									
		0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000
0.20	0.91623	2	0.91323	2	0.91026	2	0.90883	2	0.90707	2	0.90554
0.50	0.84893	2	0.83325	2	0.81855	2	0.80708	2	0.79735	2	0.78894
1.00	0.90116	2	0.90309	2	0.77653	2	0.73471	2	0.70116	2	0.67365
1.20	0.98775	2	0.87775	2	0.79011	2	0.73842	2	0.69196	2	0.65475
1.50	0.92538	1	0.60032	2	0.86776	2	0.77430	2	0.73477	2	0.65101
1.70	0.97400	1	0.95406	1	0.93616	1	0.81511	1	0.72779	2	0.66184
2.00	0.80360	1	0.89920	1	0.97265	1	0.89497	2	0.77886	2	0.69251
2.50	0.70542	1	0.79593	1	0.88575	1	0.97474	1	0.88554	2	0.76604
3.00	0.65274	1	0.72061	1	0.81366	1	0.90685	1	1.00000	2	0.84875
4.00	0.51660	1	0.51010	1	0.70760	1	0.80309	1	0.90116	1	1.00000
5.00	0.44169	1	0.53563	1	0.63194	1	0.73503	1	0.83061	1	0.93192
6.00	0.38831	1	0.48181	1	0.57870	1	0.67802	1	0.77914	1	0.88161
8.00	0.31778	1	0.41102	1	0.52838	1	0.60489	1	0.71552	1	0.81390
10.00	0.27354	1	0.36589	1	0.46469	1	0.56528	1	0.66766	1	0.77127
12.00	0.24233	1	0.33707	1	0.43528	1	0.53619	1	0.63973	1	0.74233
15.00	0.21276	1	0.30709	1	0.40593	1	0.50716	1	0.60976	1	0.71319
17.00	0.19828	1	0.29238	1	0.39226	1	0.49326	1	0.59619	1	0.69949
20.00	0.18200	1	0.27748	1	0.37706	1	0.47895	1	0.53104	1	0.68411
25.00	0.16368	1	0.26014	1	0.35020	1	0.46177	1	0.56408	1	0.66640
30.00	0.15164	1	0.24389	1	0.34925	1	0.45030	1	0.55292	1	0.65535
35.00	0.14320	1	0.24136	1	0.34160	1	0.44311	1	0.54505	1	0.64724
40.00	0.13598	1	0.23532	1	0.33597	1	0.43742	1	0.53921	1	0.64119
45.00	0.13224	1	0.23095	1	0.33167	1	0.43304	1	0.53470	1	0.63651
50.00	0.12851	1	0.22753	1	0.32828	1	0.42958	1	0.53112	1	0.63278

ADHESIVE-BONDED SCARF JOINTS (ELASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

THERMAL MISMATCH COEFFICIENT = -1.000 FOR TENSION, = 1.000 FOR COMPRESSION

NON-DIMENSIONALIZED STRENGTH :

0 = BOTH ENDS EQUALLY LOADED  
1 = SOFT ET END CRITICAL  
2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	0.0	1	0.0	1	0.0	1	0.0	1	0.0	1
0.20	0.1773	1	0.1778	1	0.1783	1	0.1786	1	0.1790	1
0.50	0.3464	1	0.3527	1	0.3581	1	0.3629	1	0.3671	1
1.00	0.4249	1	0.4515	1	0.4759	1	0.4981	1	0.5185	1
1.20	0.4206	1	0.4567	1	0.4901	1	0.5213	1	0.5503	1
1.50	0.4031	1	0.4335	1	0.5015	1	0.5247	1	0.5907	1
1.70	0.3999	1	0.4504	1	0.5086	1	0.5647	1	0.6137	1
2.00	0.3727	1	0.4498	1	0.5236	1	0.5964	1	0.6678	1
2.50	0.3547	1	0.4593	1	0.5642	1	0.6688	1	0.7729	1
3.00	0.3492	1	0.4595	1	0.6241	1	0.7644	1	0.9056	1
4.00	0.3648	1	0.5715	1	0.7862	1	1.0070	1	1.2324	1
5.00	0.4022	1	0.8181	1	0.9848	1	1.2943	1	1.6123	1
6.00	0.4521	1	0.8181	1	1.2058	1	1.6093	1	2.0245	1
8.00	0.5739	1	1.1135	1	1.6893	1	2.2896	1	2.9051	1
10.00	0.7123	1	1.4343	1	2.2052	1	3.0066	1	3.8240	1
12.00	0.8610	1	1.7712	1	2.7417	1	3.7473	1	4.7745	1
15.00	1.0963	1	2.2966	1	3.5766	1	4.4338	1	6.2195	1
17.00	1.2593	1	2.6553	1	4.1342	1	5.6525	1	7.1932	1
20.00	1.5104	1	3.2062	1	4.9906	1	6.8162	1	8.6635	1
25.00	1.9426	1	4.1423	1	6.4386	1	8.7739	1	11.1297	1
30.00	2.3874	1	5.0948	1	9.009	1	10.7451	1	12.6073	1
35.00	2.8417	1	6.0583	1	9.3735	1	12.7243	1	16.0913	1
40.00	3.3031	1	7.0295	1	10.4525	1	14.7086	1	18.5795	1
45.00	3.7703	1	8.0053	1	12.3362	1	16.6964	1	21.0703	1
50.00	4.2420	1	8.9972	1	13.8226	1	18.6866	1	23.5630	1

ADHESIVE-BONDED SCARF JOINTS (ELASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

THERMAL MISMATCH COEFFICIENT = -1.000 FOR TENSION, = 1.000 FOR COMPRESSION

AVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS :

0 = BOTH ENDS EQUALLY LOADED  
1 = SOFT ET END CRITICAL  
2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	1.00000	1	1.00000	1	1.00000	1	1.00000	1	1.00000	1
0.20	0.88632	1	0.88930	1	0.89129	1	0.89325	1	0.89496	1
0.50	0.69238	1	0.70537	1	0.71621	1	0.72571	1	0.73410	1
1.00	0.42491	1	0.45156	1	0.47586	1	0.49810	1	0.51854	1
1.20	0.35053	1	0.38056	1	0.40944	1	0.43440	1	0.45862	1
1.50	0.26871	1	0.30236	1	0.33435	1	0.36479	1	0.39378	1
1.70	0.22938	1	0.26442	1	0.29017	1	0.33217	1	0.36396	1
2.00	0.19637	1	0.22442	1	0.26171	1	0.29821	1	0.33389	1
2.50	0.14187	1	0.18374	1	0.22567	1	0.26752	1	0.30916	1
3.00	0.11640	1	0.15182	1	0.20804	1	0.25479	1	0.30187	1
4.00	0.09121	1	0.14287	1	0.19655	1	0.25175	1	0.30810	1
5.00	0.08043	1	0.13722	1	0.19695	1	0.25887	1	0.32245	1
6.00	0.07536	1	0.13635	1	0.20097	1	0.26822	1	0.33741	1
8.00	0.07173	1	0.13219	1	0.21112	1	0.28608	1	0.36313	1
10.00	0.07123	1	0.14343	1	0.22052	1	0.30066	1	0.38280	1
12.00	0.07175	1	0.14706	1	0.22948	1	0.31227	1	0.39788	1
15.00	0.07309	1	0.15310	1	0.23747	1	0.32659	1	0.41466	1
17.00	0.07497	1	0.15625	1	0.24119	1	0.32925	1	0.42313	1
20.00	0.07552	1	0.16031	1	0.24953	1	0.34026	1	0.43317	1
25.00	0.07770	1	0.16569	1	0.25752	1	0.35096	1	0.44519	1
30.00	0.07958	1	0.16943	1	0.25936	1	0.35981	1	0.45358	1
35.00	0.08119	1	0.17310	1	0.26781	1	0.36355	1	0.46076	1
40.00	0.08258	1	0.17574	1	0.27131	1	0.36772	1	0.46449	1
45.00	0.08378	1	0.17792	1	0.27413	1	0.37103	1	0.46923	1
50.00	0.08484	1	0.17974	1	0.27645	1	0.37373	1	0.47126	1

## A.2 Computer Program A4ED For Lower Bound Elastic-Plastic Strength of Bonded Scarf Joints

This FORTRAN IV digital computer program covers a simple efficient approximate solution for the elastic-plastic strength of most bonded scarf joints of practical proportions and materials. Its development was needed as a sufficiently close starting point for convergence to proceed in the more precise program A4EE. It transpired, on examination of the equivalent results computed by A4EE that the quicker computations of A4ED were satisfactory as final answers provided that (1) and adhesive non-linear behavior was not negligible, i.e., that  $\gamma_p/\gamma_e > 0.5$ , (2) the thermal mismatch coefficient is not too high, i.e., that  $C_{THERM} < 2$ , and (3) that the stiffness mismatch between adherends be not too great, i.e., that  $0.2 \leq ETR \leq 1$ .

The input data for program A4ED is precisely the same as for program A4EE with the exception that  $\gamma_p/\gamma_e$  for the adhesive cannot be equal to zero for A4EE. In other words, perfectly elastic adhesive behavior must be excluded from A4EE. On the other hand, the values computed by A4ED for zero adhesive plasticity are unduly conservative.

A listing of the program and sample outputs follow.

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CDECK A4FD
C ELASTIC-PLASTIC ANALYSIS OF UNBALANCED SCARF JOINTS          A4ED0010
C LOWER BOUND ANALYSIS PROVIDED WHICH IS ACCURATE FOR DESIGN      A4ED0020
C NON-DIMENSIONALIZED AVERAGE SHEAR STRESSES COMPUTED           A4ED0030
C NON-DIMENSIONALIZED JOINT STRENGTHS COMPUTED                  A4ED0040
C RANGE OF ADHESIVE DUCTILITIES INCLUDED                         A4ED0050
C RANGES OF ADHEREND STIFFNESS AND THERMAL IMBALANCES ACCOUNTED FOR A4ED0060
C DATA PRESENTATION FOR TENSILE SHEAR LOADING                  A4ED0070
C CHANGE SIGN OF CTERM TO USE FOR COMPRESSIVE SHEAR LOADS       A4ED0080
C SET CTERM = 0. AND REPLACE ADHEREND ET'S WITH GT'S FOR IN-PLANF A4ED0090
C   1 (EDGEWISE) SHEAR LOADING                                     A4ED0100
C
C DIMENSION DL(J), ETR(K), CTHERM(I), GPOVGE(L), A(N), TRATIO(J,NCRTND), A4ED0130
C   1 TAUAVG(J,K), ICRTND(J,K), STRGTH(J,K), THERMC(NCRTND), A4ED0140
C   2 VR(NCRTND), VU(NCRTND), VL(NCRTND), OLTRNT(NCRTND), A4ED0150
C   3 OLTRNC(NCRTND), TRANSL(K)                                 A4ED0160
C
C DIMENSION DL(40), ETR(10), CTHERM(20), GPOVGE(20), A(50), A4ED0170
C   1 TRATIO(40,2), TAUAVG(40,10), STRGTH(40,10), ICRTND(40,10), A4ED0180
C   2 THERMC(2), VR(2), VU(2), VL(2), OLTRNT(2), OLTRNC(2), TRANSL(10) A4ED0190
C
C READ IN INPUT DATA                                         A4ED0200
C READ IN ARRAY SIZES                                       A4ED0210
C   READ (5,10) IMAX, JMAX, KMAX, LMAX, NMAX                 A4ED0220
C   10 FORMAT (5I5)                                         A4ED0230
C IMAX .LE. 20, JMAX .LE. 40, KMAX .LE. 10, LMAX .LE. 20, A4ED0250
C   1 NMAX .LE. 50 AND .GE. 10.                                A4ED0260
C READ IN NON-DIMENSIONALIZED OVERLAP ARRAY                  A4ED0270
C   DL(1) = 0.                                                 A4ED0280
C DL(J) MUST BE IN ASCENDING ORDER                          A4ED0290
C DL(2) MUST BE LESS THAN 0.2 FOR IDENTIFICATION OF CRITICAL END A4ED0300
C   1 OF JOINT AT ZERO OVERLAP (LIMITING CASE)             A4ED0310
C DL(J) .LT. 100. FOR COMPATIBILITY WITH FORMAT STATEMENTS 470 & 590 A4ED0320
C   READ (5,20) (DL(J), J = 2, JMAX)                         A4ED0330
C NOTE JMAX ONE MORE THAN INPUT VALUES ON CARD(S)          A4ED0340
C   20 FORMAT (12F6.2)                                         A4ED0350
C READ IN STIFFNESS IMBALANCE ARRAY                         A4ED0360
C IDENTIFY ADHERENDS SUCH THAT ETR(K) = (ET11/(ET12 .LE. 1. A4ED0370
C STIFFNESS RATIOS SHOULD BE IN ASCENDING OR DESCENDING ORDER A4ED0380
C ETR(K) SHOULD INCLUDE VALUE 1. BUT MUST EXCLUDE VALUE 0.     A4ED0390
C   READ (5,30) (ETR(K), K = 1, KMAX)                         A4ED0400
C   30 FORMAT (10F5.2)                                         A4ED0410
C READ IN NON-DIMENSIONALIZED THERMAL MISMATCH COEFFICIENTS A4ED0420
C CTHERM = PROPNL * (ALPHA(2)-ALPHA(1)) * (OPERATING TEMP. - CURE TEMP.) A4ED0430
C NEED CTHERM(I) ARRAY TO CONTAIN BOTH POSITIVE AND NEGATIVE VALUES A4ED0440
C   1 TO COVER BOTH TENSILE AND COMPRESSIVE LOADS            A4ED0450
C   READ (5,40) (CTHERM(I), I = 1, IMAX)                      A4ED0460
C   40 FORMAT (10F7.3)                                         A4ED0470
C READ IN PLASTIC-TO-ELASTIC STRAIN RATIO ARRAY           A4ED0480
C GPOVGE(L) MUST BE .GT. 0. FOR ELASTIC-PLASTIC ANALYSIS    A4ED0490
C PURELY ELASTIC SOLUTION OBTAINED FROM SEPARATE PROCEDURE A4ED0500
C   READ (5,50) (GPOVGE(L), L = 1, LMAX)                      A4ED0510
C   50 FORMAT (14F5.2)                                         A4ED0520
C
C PRINT OUT INPUT DATA                                     A4ED0530
C   WRITE (6,60) IMAX, JMAX, KMAX, LMAX, NMAX                A4ED0540
C   60 FORMAT (1H1, 9H IMAX = ,12, 9H JMAX = ,12, 9H KMAX = ,12, A4ED0550
C   1 9H LMAX = ,12, 9H NMAX = ,12)                           A4ED0560
C   WRITE (6,70)                                             A4ED0570
C   70 FORMAT (10H OVERLAPS)                                    A4ED0580
C   WRITE (6,80) (DL(J), J = 1, JMAX)                         A4ED0590
C   80 FORMAT (12F6.2)                                         A4ED0600
C   WRITE (6,90)                                             A4ED0610
C   90 FORMAT (22H STIFFNESS IMBALANCES)                      A4ED0620
C   WRITE (6,100) (ETR(K), K = 1, KMAX)                         A4ED0630
C   100 FORMAT (14F5.2)                                         A4ED0640
C   WRITE (6,110)                                             A4ED0650
C   110 FORMAT (20H THERMAL MISMATCHES)                        A4ED0660
C   WRITE (6,120) (CTHERM(I), I = 1, IMAX)                   A4ED0670
C   120 FORMAT (10F7.3)                                         A4ED0680
C   WRITE (6,130)                                             A4ED0690
C   130 FORMAT (34H PLASTIC TO ELASTIC STRAIN RATIOS)         A4ED0700
C   WRITE (6,140) (GPOVGE(L), L = 1, LMAX)                   A4ED0710
C   140 FORMAT (14F5.1)                                         A4ED0720
C
C START COMPUTATIONAL DO LOOPS                            A4ED0730
C   DO 620 L = 1, LMAX                                     A4ED0740
C   GAMMAR = GPOVGE(L)                                      A4ED0750
C
C ENSURE EXCLUSION OF NEGATIVE PLASTICITY IN ADHESIVE (ERROR IN DATA) A4ED0760
C IF (GAMMAR .LT. 0.) GO TO 620                           A4ED0770
C   620 I = i, IMAX                                         A4ED0780
C   THERMC(1) = CTHERM(I)                                   A4ED0790
C   THERMC(2) = - THERMC(1)                                 A4ED0800
C   DO 350 K = 1, KMAX                                     A4ED0810
C   350 VR(1) = ETR(K)                                     A4ED0820
C   VR(2) = 1. / VR(1)                                     A4ED0830
C   VU(1) = 1. - VR(1)                                     A4ED0840
C   VL(1) = 1. + VR(1)                                     A4ED0850
C   VU(2) = 1. - VR(2)                                     A4ED0860
C

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      VL(2) = 1. + VR(2)          A4ED0890
C SPECIAL PROCEDURE FOR PURELY ELASTIC ADHESIVE JOINT          A4ED0900
      IF (GAMMAR .GT. 0.) GO TO 160          A4ED0920
C SET ZERO TRANSITIONAL LENGTH FOR PURELY-ELASTIC JOINTS          A4ED0930
      TRANSI(K) = 0.          A4ED0940
      DO 150 J = 2, JMAX          A4ED0950
      OLAP = OL(J)
      DO 150 NCRTND = 1, 2          A4ED0960
      150 TRATI(J,NCRTND) = 1. - VU(NCRTND) + VL(NCRTND)*THERMC(NCRTND)/OLAP          A4ED0970
      GO TO 270          A4ED0990
C ESTABLISH TRANSITIONAL OVERLAPS FROM FULLY-PLASTIC TO ELASTIC-PLASTIC          A4ED1000
      1 BEHAVIOUR AS REFERENCE LENGTH FOR START OF ITERATIONS          A4ED1010
C SPECIAL PROCEDURE FOR LESS THAN COMPLETELY UNBALANCED JOINTS          A4ED1020
      160 IF ((THERMC(1) .EQ. 0.) .AND. (VR(1) .EQ. 1.)) GO TO 170          A4ED1030
      IF (THERMC(1) .EQ. 0.) GO TO 180          A4ED1040
      IF (VR(1) .EQ. 1.) GO TO 190          A4ED1050
C IF NONE OF THESE, JOINT CONTAINS BOTH IMBALANCES          A4ED1060
      GO TO 200          A4ED1070
C SET INFINITE TRANSITIONAL OVERLAP FOR IDENTICAL ADHERENDS          A4ED1080
      170 OLTRNT(1) = 1000000.          A4ED1090
      OLTRNT(2) = 1000000.          A4ED1100
C OLTRNC(1) = 1000000.          A4ED1110
C OLTRNC(2) = 1000000.          A4ED1120
      GO TO 210          A4ED1130
C SET TRANSITIONAL OVERLAPS FOR STIFFNESS IMBALANCE ONLY          A4ED1140
C IN THE ABSENCE OF THERMAL MISMATCH, SAME END IS CRITICAL FOR BOTH          A4ED1150
C 1 TENSILE SHEAR AND COMPRESSIVE SHEAR LOADING          A4ED1160
      180 IF (VU(1) .GT. 0.) OLTRNT(1) = SQRT(GAMMAR*VL(1)/VU(1))          A4ED1170
      IF (VU(1) .LE. 0.) OLTRNT(1) = 1000000.          A4ED1180
      IF (VU(2) .GT. 0.) OLTRNT(2) = SQRT(GAMMAR*VL(2)/VU(2))          A4ED1190
      IF (VU(2) .LE. 0.) OLTRNT(2) = 1000000.          A4ED1200
C IF (VU(1) .GT. 0.) OLTRNC(1) = SQRT(GAMMAR*VL(1)/VU(1))          A4ED1210
C IF (VU(1) .LE. 0.) OLTRNC(1) = 1000000.          A4ED1220
C IF (VU(2) .GT. 0.) OLTRNC(2) = SQRT(GAMMAR*VL(2)/VU(2))          A4ED1230
C IF (VU(2) .LE. 0.) OLTRNC(2) = 1000000.          A4ED1240
      GO TO 210          A4ED1250
C SET TRANSITIONAL OVERLAPS FOR THERMAL MISMATCH ONLY          A4ED1260
      190 IF (THERMC(1) .LT. 0.) OLTRNT(1) = -GAMMAR/THERMC(1)          A4ED1270
      IF (THERMC(1) .GE. 0.) OLTRNT(1) = 1000000.          A4ED1280
      IF (THERMC(2) .LT. 0.) OLTRNT(2) = -GAMMAR/THERMC(2)          A4ED1290
      IF (THERMC(2) .GE. 0.) OLTRNT(2) = 1000000.          A4ED1300
C IF (THERMC(1) .GE. 0.) OLTRNC(1) = GAMMAR/THERMC(1)          A4ED1310
C IF (THERMC(1) .LT. 0.) OLTRNC(1) = 1000000.          A4ED1320
C IF (THERMC(2) .GE. 0.) OLTRNC(2) = GAMMAR/THERMC(2)          A4ED1330
C IF (THERMC(2) .LT. 0.) OLTRNC(2) = 1000000.          A4ED1340
      GO TO 210          A4ED1350
200 CONTINUE          A4ED1360
C STANDARD PROCEDURE FOR COMPLETELY UNBALANCED JOINTS          A4ED1370
      V1 = THERMC(1) * VL(1) / (2. * VU(1))          A4ED1380
      V2 = THERMC(2) * VL(2) / (2. * VU(2))          A4ED1390
      V3 = V1*V1 + GAMMAR*VL(1)/VU(1)          A4ED1400
      V4 = V2*V2 + GAMMAR*VL(2)/VU(2)          A4ED1410
C ESTABLISH TRANSITIONAL OVERLAPS BELOW WHICH JOINT IS FULLY PLASTIC          A4ED1420
C NEXT FOUR STATEMENTS APPLY FOR TENSILE SHEAR LOADING          A4ED1430
      IF (V3 .GE. 0.) OLTRNT(1) = V1 + SQRT(V3)          A4ED1440
C IF NOT, OTHER END OF JOINT CRITICAL          A4ED1450
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT          A4ED1460
C SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS          A4ED1470
      IF ((V3 .LT. 0.) .OR. (OLTRNT(1) .LE. 0.)) OLTRNT(1) = 1000000.          A4ED1480
      IF (V4 .GE. 0.) OLTRNT(2) = V2 + SQRT(V4)          A4ED1490
C IF NOT, OTHER END OF JOINT CRITICAL          A4ED1500
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT          A4ED1510
C SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS          A4ED1520
      IF ((V4 .LT. 0.) .OR. (OLTRNT(2) .LE. 0.)) OLTRNT(2) = 1000000.          A4ED1530
C IF ICRTND .EQ. 2 FOR SHORT OVERLAPS, OLTRNT(1) WILL BE COMPUTED VERY          A4ED1540
C 1 LARGE, AND VICE VERSA          A4ED1550
C THIS IS PHYSICALLY REALISTIC AND DOES NOT LEAD TO IMPOSSIBLE COMPUTING          A4ED1560
C IF BOTH V3 AND V4 ARE POSITIVE, EITHER OLTRNT(1) OR OLTRNT(2) WILL BE          A4ED1570
C 1 COMPUTED NEGATIVE. NEED TO PREVENT COMPUTATIONS BASED ON THIS          A4ED1580
C 2 UNREAL SITUATION. HENCE CHECKS ABOVE AND BELOW          A4ED1590
C NEXT FOUR STATEMENTS WOULD APPLY FOR COMPRESSIVE SHEAR LOADING          A4ED1600
      IF (V3 .GE. 0.) OLTRNC(1) = -V1 + SQRT(V3)          A4ED1610
C IF NOT, OTHER END OF JOINT CRITICAL          A4ED1620
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT          A4ED1630
C SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS          A4ED1640
      IF ((V3 .LT. 0.) .OR. (OLTRNC(1) .LE. 0.)) OLTRNC(1) = 1000000.          A4ED1650
      IF (V4 .GE. 0.) OLTRNC(2) = -V2 + SQRT(V4)          A4ED1660
C IF NOT, OTHER END OF JOINT CRITICAL          A4ED1670
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT          A4ED1680
C SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS          A4ED1690
      IF ((V4 .LT. 0.) .OR. (OLTRNC(2) .LE. 0.)) OLTRNC(2) = 1000000.          A4ED1700
C 210 DO 260 NCRTND = 1, 2          A4ED1710
C SET UNIFORM STRESS FOR SHORT OVERLAPS          A4ED1720
      DO 220 J = 2, JMAX          A4ED1730
      JSAVE = J          A4ED1740
      A4ED1750
      A4ED1760

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      IF (OL(J) .GT. OLTRNT(NCRTND)) GO TO 230          A4ED1770
C IF NOT, JOINT IS FULLY PLASTIC                         A4ED1780
 220 TRATIO(J,NCRTND) = 1.                                A4ED1790
      IF (JSAVE .EQ. JMAX) GO TO 260                      A4ED1800
C COMPUTE JOINT STRENGTH FOR ELASTIC-PLASTIC ADHESIVE BEHAVIOUR A4ED1810
 230 DO 250 J = JSIZE, JMAX                            A4ED1820
      OLAP = OL(J)                                         A4ED1830
      OLAP2 = OLAP * OLAP                                 A4ED1840
C COMPUTE ADOVERL FOR MINIMUM VALUE OF TAVOTP BY ITERATION A4ED1850
C SET INITIAL ESTIMATE OF EXTENT OF PLASTIC ZONE FROM TRANSITIONAL OLAP A4ED1860
      ADOVERL = OLTRNT(NCRTND) / OLAP                     A4ED1870
      DD 240 N = 1, NMAX                                A4ED1880
      ARMDR = 1. - ADOVERL                               A4ED1890
      ADOVERL = -ARMDR*ALOG(ARMDR) + (GAMMAR / ((VU(NCRTND)/VL(NCRTND))* A4ED1900
      1. - OLAP2 - THERMC(NCRTND)*OLAP))                A4ED1910
      IF (ADOVERL .GT. 0.999) ADOVERL = 0.999            A4ED1920
      IF (ADOVERL .LT. 0.001) ADOVERL = 0.001            A4ED1930
 240 CONTINUE                                              A4ED1940
      TRATIO(J,NCRTND) = 1.                                A4ED1950
      IF (ADOVERL .GT. 0.9999) GO TO 250                 A4ED1960
C COMPUTE CORRESPONDING AVERAGE SHEAR STRESS             A4ED1970
      TRATIO(J,NCRTND) = 1. - (VL(NCRTND)*GAMMAR/OLAP2 + (VL(NCRTND)* A4ED1980
      1. - THERMC(NCRTND)/OLAP - VU(NCRTND)) * ADOVERL) / ALOG(ARMDR) A4ED1990
 250 CONTINUE                                              A4ED2000
 260 CONTINUE                                              A4ED2010
C VALUES COMPUTED ARE NOW STORED IN TRATIO(J,NCRTND)       A4ED2020
 270 DO 340 J = 2, JMAX                                A4ED2030
      OLAP = OL(J)                                         A4ED2040
      TAU1 = TRATIO(J,1)                                    A4ED2050
      TAU2 = TRATIO(J,2)                                    A4ED2060
      IF ( (TAU1 .LT. 1.) .OR. (TAU2 .LT. 1.) ) GO TO 280 A4ED2070
C IF SO, JOINT IS NOT FULLY PLASTIC                      A4ED2080
C IF NOT, IDENTIFY CRITICAL END OF JOINT FROM SHEAR STRAIN GRADIENT A4ED2090
      GRADNT = THERMC(1) - OLAP*VU(1)/VL(1)               A4ED2100
      IF (GRADNT .LT. 0.) ICRTND(J,K) = 1                  A4ED2110
      IF (GRADNT .EQ. 0.) ICRTND(J,K) = 0                  A4ED2120
      IF (GRADNT .GT. 0.) ICRTND(J,K) = 2                  A4ED2130
      TAUAVG(J,K) = 1.                                     A4ED2140
      STRGTH(J,K) = OLAP                                  A4ED2150
C TRANSITIONAL OVERLAPS ALREADY COMPUTED FOR ELASTIC ADHESIVE A4ED2160
C BYPASS RECOMPUTATION. THIS APPLIES TO ELASTIC-PLASTIC ADHESIVES A4ED2170
      IF (GAMMAR .EQ. 0.1) GO TO 340                      A4ED2180
      MCRTND = ICRTND(J,K)                                A4ED2190
      IF (MCRTND .EQ. 0) MCRTND = 1                        A4ED2200
      TRANS(K) = OLTRNT(MCRTND)                            A4ED2210
      GO TO 340                                            A4ED2220
 280 DIFFNC = TAU1 - TAU2                                A4ED2230
C IF DIFFNC .LT. 0., NCRTND .EQ. 1                      A4ED2240
C IF DIFFNC .EQ. 0., NCRTND .EQ. 0                      A4ED2250
C IF DIFFNC .GT. 0., NCRTND .EQ. 2                      A4ED2260
      IF (DIFFNC) 290, 300, 310                           A4ED2270
C ADHEREND (1) END OF JOINT CRITICAL                   A4ED2280
 290 TAUAVG(J,K) = TAU1                                  A4ED2290
      STRGTH(J,K) = TAU1 * OLAP                          A4ED2300
      ICRTND(J,K) = 1                                     A4ED2310
C COVER SITUATION WHERE TRANSITIONAL LENGTH IS LESS THAN OL(2) A4ED2320
      IF (J .EQ. 2) TRANS(K) = OLTRNT(1)                  A4ED2330
      GO TO 320                                           A4ED2340
C BOTH ENDS OF JOINT EQUALLY CRITICAL FROM NULLIFYING (OR ZERO) A4ED2350
C 1 ADHEREND IMBALANCES                                A4ED2360
 300 TAUAVG(J,K) = TAU1                                  A4ED2370
      STRGTH(J,K) = TAU1 * OLAP                          A4ED2380
      ICRTND(J,K) = 0                                     A4ED2390
C COVER SITUATION WHERE TRANSITIONAL LENGTH IS LESS THAN OL(2) A4ED2400
      IF (J .EQ. 2) TRANS(K) = OLTRNT(1)                  A4ED2410
      GO TO 320                                           A4ED2420
C ADHEREND (2) END OF JOINT CRITICAL                   A4ED2430
 310 TAUAVG(J,K) = TAU2                                  A4ED2440
      STRGTH(J,K) = TAU2 * OLAP                          A4ED2450
      ICRTND(J,K) = 2                                     A4ED2460
C COVER SITUATION WHERE TRANSITIONAL LENGTH IS LESS THAN OL(2) A4ED2470
      IF (J .EQ. 2) TRANS(K) = OLTRNT(2)                  A4ED2480
C COVER CASES OF ZERO OR NEGATIVE ESTIMATED STRENGTHS A4ED2490
 320 IF (TAUAVG(J,K) .GT. 0.) GO TO 330                 A4ED2500
C IF NOT, JOINT HAS BROKEN DUE TO THERMAL STRESSES WITHOUT EXTERNAL LOAD A4ED2510
      TAUAVG(J,K) = 0.                                     A4ED2520
      STRGTH(J,K) = 0.                                     A4ED2530
      GO TO 340                                           A4ED2540
 330 IF (TAUAVG(J,K) .LE. 1.) GO TO 340                 A4ED2550
C IF NOT, THERE HAS BEEN A COMPUTATIONAL MISTAKE        A4ED2560
C PRINT ASTERISKS TO IDENTIFY ERROR                    A4ED2570
C RERUN WITH GREATER VALUE OF NMAX                   A4ED2580
      TAUAVG(J,K) = 100.                                   A4ED2590
      STRGTH(J,K) = 1000.                                 A4ED2600
 340 CONTINUE                                              A4ED2610
 350 CONTINUE                                              A4ED2620

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C SET UNIFORM STRESS FOR ZERO OVERLAP          A4ED2650
  DO 360 K = 1, KMAX                         A4ED2660
    TAUAVG(1,K) = 1.                           A4ED2670
    STRGTH(1,K) = 0.                           A4ED2680
  360 ICRTND(1,K) = ICRTND(2,K)               A4ED2690
C HENCE NEED FOR OL(2) TO BE SMALL ENOUGH TO BE LESS THAN THAT AT WHICH A4ED2700
  1 NCRTND CHANGES                            A4ED2710
C END OF COMPUTATIONS. START PRINTING OUT OF TABULATED RESULTS      A4ED2720
C PRINT OUT AVERAGE STRESS HEADING            A4ED2730
  WRITE (6,370)                                A4ED2740
  370 FORMAT (1H1/, 5(1H0/), 27X, 56HADHESIVE-BONDED SCARF JOINTS (ELAS A4ED2780
    1TIC-PLASTIC ANALYSIS)/,                   A4ED2790
    2 39X, 31HNON-DIMENSIONALIZED FORMULATION/) A4ED2800
    IF (GAMMAR .NE. 0.) GO TO 390             A4ED2810
    WRITE (6,380)                                A4ED2820
  380 FORMAT (1H0, 42X, 23HPURELY ELASTIC ADHESIVE)                 A4ED2830
    GO TO 410                                  A4ED2840
  390 WRITE (6,400) GAMMAR                     A4ED2850
  400 FORMAT (1H0, 27X, 49HPLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO A4ED2860
    1 = F5.2)                                 A4ED2870
  410 IF (CTHERM(1) .NE. 0.) GO TO 430           A4ED2880
    WRITE (6,420)                                A4ED2890
  420 FORMAT (1H , 37X, 33HZERO THERMAL MISMATCH COEFFICIENT)        A4ED2900
    GO TO 450                                  A4ED2910
  430 WRITE (6,440) THERMC(1), THERMC(2)       A4ED2920
  440 FORMAT (1H , 16X, 31HTHERMAL MISMATCH COEFFICIENT = , F6.3,     A4ED2930
    1 17H FOR TENSION, = , F6.3, 16H FOR COMPRESSION)                A4ED2940
  450 WRITE (6,460) (ETRI(K), K = 1, KMAX)          A4ED2950
  460 FORMAT( 1H0, 67X, 30HO = BOTH ENDS EQUALLY CRITICAL/, 20X,     A4ED2960
    1 72HAVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS , 1 = SOFT ET E A4ED2970
    2ND CRITICAL/, 68X, 25H2 = STIFF ET END CRITICAL/,               A4ED2980
    3 8HO SCALED, 31X, 39HEXTENSINAL STIFFNESS (THICKNESS) RATIO/,   A4ED2990
    4 7H L/T/, 7H RATIO, F7.1, 9F10.1/, 1H )                      A4ED3000
C WRITE OUT TABULATIONS OF AVERAGE BOND STRESSES                  A4ED3010
  DO 480 J = 1, JMAX                         A4ED3020
    WRITE (6,470) OL(J), ((TAUAVG(J,K), ICRTND(J,K)), K = 1, KMAX) A4ED3030
  470 FORMAT (1H , F6.2, 2X, 10(F7.5, 1X, 1I, 1X))                 A4ED3040
  480 CONTINUE                                A4ED3050
C PRINT OUT JOINT STRENGTH HEADING            A4ED3060
  WRITE (6,490)                                A4ED3070
  490 FORMAT (1H1/, 5(1H0/), 27X, 56HADHESIVE-BONDED SCARF JOINTS (ELAS A4ED3090
    1TIC-PLASTIC ANALYSIS)/,                   A4ED3100
    2 39X, 31HNON-DIMENSIONALIZED FORMULATION/) A4ED3110
    IF (GAMMAR .NE. 0.) GO TO 510             A4ED3120
    WRITE (6,500)                                A4ED3130
  500 FORMAT (1H0, 42X, 23HPURELY ELASTIC ADHESIVE)                 A4ED3140
    GO TO 530                                  A4ED3150
  510 WRITE (6,520) GAMMAR                     A4ED3160
  520 FORMAT (1H0, 27X, 49HPLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO A4ED3170
    1 = F5.2)                                 A4ED3180
  530 IF (CTHERM(1) .NE. 0.) GO TO 550           A4ED3190
    WRITE (6,540)                                A4ED3200
  540 FORMAT (1H , 37X, 33HZERO THERMAL MISMATCH COEFFICIENT)        A4ED3210
    GO TO 570                                  A4ED3220
  550 WRITE (6,560) THERMC(1), THERMC(2)       A4ED3230
  560 FORMAT (1H , 16X, 31HTHERMAL MISMATCH COEFFICIENT = , F6.3,     A4ED3240
    1 17H FOR TENSION, = , F6.3, 16H FOR COMPRESSION)                A4ED3250
  570 WRITE (6,580) (ETRI(K), K = 1, KMAX)          A4ED3260
  580 FORMAT( 1H0, 67X, 30HO = BOTH ENDS EQUALLY CRITICAL/, 20X,     A4ED3270
    1 72HNON-DIMENSIONALIZED JOINT STRENGTH, 1 = SOFT ET E A4ED3280
    2ND CRITICAL/, 68X, 25H2 = STIFF ET END CRITICAL/,               A4ED3290
    3 8HO SCALED, 31X, 39HEXTENSINAL STIFFNESS (THICKNESS) RATIO/,   A4ED3300
    4 7H L/T/, 7H RATIO, F7.1, 9F10.1/, 1H )                      A4ED3310
C WRITE OUT TABULATIONS OF JOINT STRENGTHS                  A4ED3320
  DO 600 J = 1, JMAX                         A4ED3330
    WRITE (6,590) OL(J), ((STRGTH(J,K), ICRTND(J,K)), K = 1, KMAX) A4ED3340
  590 FORMAT (1H , F6.2, 2X, 10(F7.4, 1X, 1I, 1X))                 A4ED3350
  600 CONTINUE                                A4ED3360
C WRITE OUT TRANSITIONAL JOINT STRENGTHS          A4ED3370
  WRITE (6,610) (TRANSL(K), K = 1, KMAX)          A4ED3380
  610 FORMAT (8HO TRANSL, 1X, 10(F7.4, 3X))          A4ED3390
  620 CONTINUE                                A4ED3400
C WRITE (6,630)                                A4ED3410
  630 FORMAT (1H1, 18H PROGRAM COMPLETED)          A4ED3420
  STOP                                         A4ED3430
  END                                           A4ED3440
                                                A4ED3450

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ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = 1.000 FOR TENSION, = -1.000 FOR COMPRESSION

NON-DIMENSIONALIZED JOINT STRENGTH , O = BOTH ENDS EQUALLY CRITICAL  
1 = SOFT ET END CRITICAL  
2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	0.0	2	0.0	2	0.0	2	0.0	2	0.0	2
0.20	0.2000	2	0.2000	2	0.2000	2	0.2000	2	0.2000	2
0.50	0.5000	2	0.5000	2	0.5000	2	0.5000	2	0.5000	2
1.00	1.0000	2	1.0000	2	1.0000	2	1.0000	2	1.0000	2
1.20	1.2000	2	1.2000	2	1.2000	2	1.2000	2	1.2000	2
1.50	1.5000	2	1.5000	2	1.5000	2	1.5000	2	1.5000	2
1.70	1.7000	2	1.7000	2	1.7000	2	1.7000	2	1.7000	2
2.00	2.0000	2	2.0000	2	2.0000	2	2.0000	2	2.0000	2
2.50	2.5000	2	2.5000	2	2.5000	2	2.5000	2	2.5000	2
3.00	3.0000	2	3.0000	2	3.0000	2	3.0000	2	3.0000	2
4.00	3.6044	2	3.9980	2	4.0000	2	4.0000	2	4.0000	2
5.00	3.9361	2	4.1495	2	4.7374	2	4.9984	2	5.0000	2
6.00	4.1817	2	4.7215	2	5.2105	2	5.6279	2	5.9935	2
8.00	4.5565	2	5.3297	2	6.0557	2	6.7756	2	7.3222	2
10.00	4.8542	2	5.8486	2	6.7955	2	7.6917	2	8.5245	2
12.00	5.1248	2	6.3263	2	7.4952	2	8.5945	2	9.6526	2
15.00	5.4906	2	7.0028	2	8.4752	2	9.9027	2	11.2764	2
17.00	5.7213	2	7.6394	2	9.1166	2	10.7510	2	12.3334	2
20.00	6.0556	2	8.0782	2	10.0562	2	12.0045	2	13.9765	2
25.00	6.5942	2	9.1229	2	11.6144	2	13.6647	2	16.4570	2
30.00	7.1195	2	10.1524	2	13.1484	2	16.1044	2	19.2133	2
35.00	7.6363	2	11.1731	2	14.6723	2	18.1325	2	21.5462	2
40.00	8.1490	2	12.1981	2	15.1907	2	19.0707	2	24.9034	2
45.00	8.6574	2	13.1993	2	17.7044	2	22.1695	2	26.5894	2
50.00	9.1630	2	14.2075	2	19.2150	2	24.1825	2	29.1049	2
TRANSL	3.1576	3.5895	4.1142	4.7761	5.6533	6.8990	8.8633	12.5777	23.1106	5.0000

ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = 1.000 FOR TENSION, = -1.000 FOR COMPRESSION

AVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS , O = BOTH ENDS EQUALLY CRITICAL  
1 = SOFT ET END CRITICAL  
2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
0.20	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
0.50	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.00	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.20	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.50	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.70	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
2.00	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
2.50	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
3.00	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
4.00	0.90110	2	0.9953	2	1.00000	2	1.00000	2	1.00000	2
5.00	0.78732	2	0.8699	2	0.94749	2	0.99268	2	1.00000	2
6.00	0.69695	2	0.78725	2	0.86882	2	0.93798	2	0.99979	2
8.00	0.56956	2	0.65571	2	0.75494	2	0.84070	2	0.91527	2
10.00	0.48592	2	0.58485	2	0.67955	2	0.74917	2	0.85249	2
12.00	0.42707	2	0.52719	2	0.62395	2	0.71654	2	0.80449	2
15.00	0.36624	2	0.46685	2	0.56502	2	0.66018	2	0.75176	2
17.00	0.33455	2	0.43755	2	0.53627	2	0.53241	2	0.72549	2
20.00	0.30278	2	0.40391	2	0.50313	2	0.60923	2	0.69483	2
25.00	0.26377	2	0.36491	2	0.46458	2	0.54259	2	0.55388	2
30.00	0.23732	2	0.33841	2	0.43829	2	0.53681	2	0.63378	2
35.00	0.21820	2	0.31923	2	0.41922	2	0.51807	2	0.61561	2
40.00	0.20372	2	0.30470	2	0.40477	2	0.50384	2	0.59177	2
45.00	0.19239	2	0.29332	2	0.39343	2	0.49266	2	0.59088	2
50.00	0.18326	2	0.28415	2	0.38430	2	0.49365	2	0.59210	2

ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = -1.000 FOR TENSION, = 1.000 FOR COMPRESSION  
NON-DIMENSIONALIZED JOINT STRENGTH + 0 = BOTH ENDS EQUALLY CRITICAL  
1 = SOFT FT END CRITICAL  
2 = STIFF FT END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	0.0	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
0.20	0.20000	1.20200	1.20200	1.20200	1.20200	1.20200	1.20200	1.20200	1.20200	1.20200
0.50	0.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000
1.00	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
1.20	1.20000	1.12000	1.12000	1.12000	1.12000	1.12000	1.12000	1.12000	1.12000	1.12000
1.50	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000	1.50000
1.70	1.70000	1.70000	1.70000	1.70000	1.70000	1.70000	1.70000	1.70000	1.70000	1.70000
2.00	1.99711	2.00000	2.00000	2.00000	2.00000	2.00000	2.00000	2.00000	2.00000	2.00000
2.50	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661
3.00	2.25021	2.44131	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661	2.49661
4.00	2.39921	2.68711	2.95391	2.95391	2.95391	2.95391	2.95391	2.95391	2.95391	2.95391
5.00	2.52001	2.90451	3.26551	3.26551	3.26551	3.26551	3.26551	3.26551	3.26551	3.26551
6.00	2.63061	3.11161	3.56721	3.56721	3.56721	3.56721	3.56721	3.56721	3.56721	3.56721
8.00	2.84031	3.51581	4.16501	4.16501	4.16501	4.16501	4.16501	4.16501	4.16501	4.16501
10.00	3.04431	3.91611	4.75011	5.57571	5.36091	5.11271	5.82621	6.49391	7.10261	7.62661
12.00	3.24521	4.31511	5.35551	6.36651	7.34551	8.28531	9.15211	10.34501	10.83181	11.51891
15.00	3.54751	4.91341	6.24991	7.55591	8.82821	10.94321	11.25431	12.39031	13.45111	14.39081
17.00	3.74781	5.31231	6.84649	9.35021	9.81941	11.25011	12.63511	13.96241	15.29931	16.32171
20.00	4.04801	5.91081	7.74331	9.54361	11.30901	13.03451	14.71231	16.12221	17.95891	19.23521
25.00	4.54731	6.90361	9.23861	11.53561	13.79641	16.01531	18.18511	20.28911	22.29701	24.12301
30.00	5.04661	7.90571	12.73471	13.52981	16.29751	19.00271	21.56601	24.26081	27.75251	29.03721
35.00	5.53871	8.90121	12.23041	15.52471	18.78081	21.98301	25.15101	28.21981	31.21921	33.96881
40.00	6.02921	9.99401	13.72481	17.51951	21.27501	24.98541	28.64101	32.22361	35.62291	38.21261
45.00	6.51591	10.99391	15.21711	19.51341	23.76921	27.97881	32.13221	36.21051	40.17181	43.86541
50.00	6.99901	11.37081	16.70711	21.50571	26.26291	30.97271	35.62441	40.20011	44.65461	48.82491
TRANSL	1.9354	2.0895	2.2570	2.4427	2.6533	2.8990	3.1967	3.5777	4.1107	5.0000

ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = -1.000 FOR TENSION, = 1.000 FOR COMPRESSION  
AVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS + 0 = BOTH ENDS EQUALLY CRITICAL  
1 = SOFT FT END CRITICAL  
2 = STIFF FT END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
0.20	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
0.50	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
1.00	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
1.20	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
1.50	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
1.70	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
2.00	0.99855	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
2.50	0.99866	0.99872	0.99878	0.99884	0.99893	0.99900	0.99907	0.99914	0.99920	0.99926
3.00	0.99877	0.81377	0.99887	0.99893	0.99900	0.99907	0.99914	0.99920	0.99926	0.99932
4.00	0.95970	0.27192	0.73847	0.79944	0.85466	0.92937	0.9924	0.99335	0.99424	0.99511
5.00	0.50400	0.58091	0.65311	0.72056	0.79299	0.84043	0.95202	0.9944	0.99952	0.99967
6.00	0.43843	0.51860	0.59465	0.66653	0.73406	0.79695	0.85473	0.90655	0.99958	0.99967
8.00	0.35504	0.39481	0.52082	0.59238	0.67257	0.74285	0.80853	0.86969	0.92416	0.99780
10.00	0.30443	0.39150	0.47691	0.55759	0.63509	0.71127	0.78262	0.84933	0.91026	0.95266
12.00	0.27051	0.35959	0.44630	0.53054	0.61213	0.69077	0.76021	0.83704	0.90265	0.95990
15.00	0.23650	0.32755	0.41665	0.50372	0.58455	0.67038	0.75028	0.82602	0.89674	0.95319
17.00	0.22046	0.31249	0.40276	0.49119	0.57761	0.66217	0.74324	0.82132	0.89467	0.96010
20.00	0.20240	0.29554	0.38716	0.47718	0.56545	0.65172	0.73562	0.81646	0.89295	0.96176
25.00	0.18189	0.27634	0.36954	0.46143	0.55186	0.64053	0.72740	0.81154	0.89188	0.96492
30.00	0.16815	0.25352	0.35782	0.45099	0.54299	0.6334	0.71220	0.80869	0.89175	0.96781
35.00	0.15825	0.25432	0.34946	0.44356	0.53659	0.62837	0.71181	0.80625	0.89197	0.97054
40.00	0.15073	0.24735	0.34312	0.43799	0.53187	0.62463	0.711602	0.80559	0.89232	0.97282
45.00	0.14480	0.24187	0.33815	0.43363	0.52820	0.62175	0.711405	0.80463	0.89271	0.97479
50.00	0.13994	0.23742	0.33414	0.43011	0.52526	0.61945	0.71250	0.80400	0.89309	0.97650

### A.3 Computer Program A4EE For Elastic-Plastic Strength of Bonded Scarf Joints

This FORTRAN IV digital computer program provides for the precise series solution for the average shear stress on bonded scarf joints with small scarf angles. It accounts for adherend stiffness and thermal imbalance as well as adhesive plasticity. The governing analysis is presented in Sections 3 and 4. This program A4EE will not handle perfectly elastic adhesives for which the program A4EC was developed. Severe convergence difficulties were encountered in the development of the numerical program. This contributed to the omission of a solution for the adherend and adhesive shear stress distributions. Whether or not the adherend allowable stresses are exceeded can be determined simply by evaluating the ratio of the adhesive peak shear stress to the adherend allowable direct stress. If this ratio exceeds the tangent of the scarf angle, the scarf angle is too small and the tip will either break off or be yielded depending on the nature of the adherend material.

The input data required to operate program A4EE is as follows.

CARD 1:

FORMAT (515)

IMAX = Number of thermal mismatch coefficients. IMAX .LE. 20.

JMAX = Number of non-dimensionalized overlaps. JMAX .LE. 40.

(Note that this is one more than the number of overlaps to be read in. The limiting case of OL(1)=0 is set by the program.)

KMAX = Number of adherend stiffness imbalances. KMAX .LE. 10.

(Note that this controls the number of answers printed across the page and cannot be increased indefinitely for a single pass through the program.)

LMAX = Number of plastic-to-elastic adhesive shear strain ratios.

LMAX .LE. 20.

NMAX = Number of terms in power series. 10 .LE. NMAX .LE. 50.

(Note NMAX = 20 is recommended.)

CARDS 2, 2A, 2B, etc.:

FORMAT (12F6.2)

OL(J) = Non-dimensionalized overlaps. Number restricted to 40 by dimension statement. (Note that OL(J) must be read in in ascending order and that OL(2), which is the first entry on card 2, must not exceed 0.5 because of internal computations. OL(1) = 0 is set by the program as a limiting case.) Values of OL(J) exceeding 50 are impractically large.

CARDS 3, 3A, 3B, etc.:

FORMAT (10F5.2)

ETR(K) = Adherend stiffness ratios ( $E_1 t_1$ )/( $E_2 t_2$ ).

Number of values restricted to 10 by dimension statement.

(Subscripts 1 and 2 must be identified such that  $0 < ETR(K) \leq 1$ .  
Array should be read in in ascending or descending order.)

CARDS 4, 4A, 4B, etc.:

FORMAT (10F7.3)

CTHERM(I) = Adherend thermal mismatch coefficients in non-dimensionalized form. Number restricted to 20 by dimension statement. (Note that equal and opposite values must be read in consecutively to account for the difference between tensile and compressive application of the shear load. Values up to  $\pm 5$  are sufficient for the available range of adhesives and adherends. Greater values are usually associated with failure of the joint under residual thermal stresses alone.)

CARDS 5, 5A, 5B, etc.:

FORMAT (14F5.2)

GPOVGE(L) = Ratio of adhesive plastic-to-elastic strain ratios. Number of entries restricted to 20 by dimension statement. (Value of zero, for elastic case, is rejected by program A4EE to prevent breakdown of the computational sequence, but accepted by A4ED.)

A complete listing and sample outputs follow. The output tables come in pairs with the ratio of the average to maximum adhesive shear stress ( $\tau_{av}/\tau_p$ ) and the non-dimensionalized joint strength ( $\tau_{av}/\tau_p$ )( $\lambda\ell$ ) as functions of the adherend extensional stiffness ratio  $ETR = E_1 t_1/E_2 t_2 \leq 1$  horizontally and the non-dimensionalized joint overlap  $\lambda\ell = \sqrt{\frac{G}{n} \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right) \ell^2}$  vertically. Each table is prepared for a single value of thermal mismatch coefficient

$CTHERM = \frac{(\alpha_2 - \alpha_1)\Delta T\lambda}{\tau_p \left( \frac{1}{E_1 t_1} + \frac{1}{E_2 t_2} \right)}$  and equal and opposite values are treated in turn to cover both tensile and compressive shear loadings. Each table is prepared for a single value of the plastic-to-elastic adhesive shear strain ratio  $\gamma_p/\gamma_e$ . The quantity TRANSL listed at the foot of each column of the non-dimensionalized strength table defines the transitional overlap at which the adhesive behavior changes from fully-plastic to elastic-plastic.

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CDECK A4EE
C ELASTIC-PLASTIC ANALYSIS OF UNBALANCED SCARF JOINTS          A4EE0010
C PRECISE SOLUTION, NOT LOWER BOUND                                A4EE0020
C NON-DIMENSIONALIZED AVERAGE SHEAR STRESSES COMPUTED           A4EE0030
C NON-DIMENSIONALIZED JOINT STRENGTHS COMPUTED                   A4EE0040
C RANGE OF ADHESIVE DUCTILITIES INCLUDED                         A4EE0050
C RANGES OF ADHEREND STIFFNESS AND THERMAL IMBALANCES ACCOUNTED FOR A4EE0060
C DATA PRESENTATION FOR TENSILE SHEAR LOADING                  A4EE0070
C CHANGE SIGN OF CTERM TO USE FOR COMPRESSIVE SHEAR LOADS      A4EE0080
C SET CTERM = 0. AND REPLACE ADHEREND ET'S WITH GT'S FOR IN-PLANE A4EE0090
C   1 (EDGEWISE) SHEAR LOADING                                     A4EE0100
C
C DIMENSION DL(J), ETR(K), CTERM(I), GPOVGE(L), TRATIO(J,NCRTND), A4EE0110
C   1 A(N), TAUAVG(J,K), STRGTH(J,K), ICRTND(J,K), THERMC(NCRTND), A4EE0120
C   2 VR(NCRTND), VU(NCRTND), VL(NCRTND), OLTRNT(NCRTND),          A4EE0130
C   3 OLTRNC(NCRTND), TRANSL(K), TAUEND(N1), SAVOTP(M), TAVOTP(M), A4EE0140
C   4 BOVERL(N1)                                                 A4EE0150
C
C   DIMENSION DL(40), ETR(10), CTERM(20), GPOVGE(20), A(50),          A4EE0160
C   1 TRATIO(40,2), TAUAVG(40,10), STRGTH(40,10), ICRTND(40,10), A4EE0170
C   2 THERMC(2), VR(2), VU(2), VL(2), OLTRNT(2), OLTRNC(2), TRANSL(10), A4EE0180
C   3 TAUEND(50), SAVOTP(50), TAVOTP(50), BOVERL(50)             A4EE0190
C
C READ IN INPUT DATA                                              A4EE0200
C READ IN ARRAY SIZES                                             A4EE0210
C   READ (5,10) IMAX, JMAX, KMAX, LMAX, NMAX                      A4EE0220
C   10 FORMAT (5I5)                                                 A4EE0230
C   IMAX .LE. 20, JMAX .LE. 40, KMAX .LE. 10, LMAX .LE. 20,          A4EE0240
C   1 NMAX .LE. 50 .AND. .GE. 10.                                    A4EE0250
C READ IN NON-DIMENSIONALIZED OVERLAP ARRAY                         A4EE0260
C   DL(1) = 0.                                                       A4EE0270
C   DL(J) MUST BE IN ASCENDING ORDER                               A4EE0280
C   DL(2) MUST BE LESS THAN 0.2 FOR IDENTIFICATION OF CRITICAL END A4EE0290
C   1 OF JOINT OF ZERO OVERLAP (LIMITING CASE)                   A4EE0300
C   DL(J) .LT. 100. FOR COMPATIBILITY WITH FORMAT STATEMENTS 550 & 640 A4EE0310
C   READ (5,20) (DL(J), J = 2, JMAX)                                A4EE0320
C
C NOTE JMAX ONE MORE THAN INPUT VALUES ON CARD(S)                 A4EE0330
C   20 FORMAT (12F6.2)                                              A4EE0340
C
C READ IN STIFFNESS IMBALANCE ARRAY                               A4EE0350
C IDENTIFY ADHERENDS SUCH THAT ETR(K) = (ET)1/(ET)2 .LE. 1.          A4EE0360
C STIFFNESS RATIOS SHOULD BE IN ASCENDING OR DESCENDING ORDER     A4EE0370
C ETR(K) SHOULD INCLUDE VALUE 1. BUT MUST EXCLUDE VALUE 0.          A4EE0380
C   READ (5,30) (ETR(K), K = 1, KMAX)                                A4EE0390
C   30 FORMAT (10F5.2)                                              A4EE0400
C
C READ IN NON-DIMENSIONALIZED THERMAL MISMATCH COEFFICIENTS       A4EE0410
C CTERM .PROPNL. (ALPHA(2)-ALPHA(1))*(OPERATING TEMP. - CURE TEMP.) A4EE0420
C NEED CTERM(I) ARRAY TO CONTAIN BOTH POSITIVE AND NEGATIVE VALUES A4EE0430
C   1 TO COVER BOTH TENSILE AND COMPRESSIVE LOADS                  A4EE0440
C   READ (5,40) (CTERM(I), I = 1, IMAX)                            A4EE0450
C   40 FORMAT (10F7.3)                                              A4EE0460
C
C READ IN PLASTIC-TO-ELASTIC STRAIN RATIO ARRAY                  A4EE0470
C GPOVGE(L) MUST BE .GT. 0. FOR ELASTIC-PLASTIC ANALYSIS          A4EE0480
C PURELY ELASTIC SOLUTION OBTAINED FROM SEPARATE PROCEDURE        A4EE0490
C   READ (5,50) (GPOVGE(L), L = 1, LMAX)                           A4EE0500
C   50 FORMAT (14F5.2)                                              A4EE0510
C
C PRINT OUT INPUT DATA                                            A4EE0520
C   WRITE (6,60) IMAX, JMAX, KMAX, LMAX, NMAX                      A4EE0530
C   60 FORMAT (1H1, 9H IMAX = ,I2, 9H JMAX = ,I2, 9H KMAX = ,I2, A4EE0540
C   1 9H LMAX = ,I2, 9H NMAX = ,I2)                                 A4EE0550
C   WRITE (6,70)                                              A4EE0560
C   70 FORMAT (10H OVERLAPS)                                         A4EE0570
C   WRITE (6,80) (DL(J), J = 1, JMAX)                                A4EE0580
C   80 FORMAT (12F6.2)                                              A4EE0590
C   WRITE (6,90)                                              A4EE0600
C   90 FORMAT (22H STIFFNESS IMBALANCES)                            A4EE0610
C   WRITE (6,100) (ETR(K), K = 1, KMAX)                            A4EE0620
C   100 FORMAT (14F5.2)                                             A4EE0630
C   WRITE (6,110)                                              A4EE0640
C   110 FORMAT (20H THERMAL MISMATCHES)                            A4EE0650
C   WRITE (6,120) (CTERM(I), I = 1, IMAX)                           A4EE0660
C   120 FORMAT (10F7.3)                                              A4EE0670
C   WRITE (6,130)                                              A4EE0680
C   130 FORMAT (34H PLASTIC TO ELASTIC STRAIN RATIOS)              A4EE0690
C   WRITE (6,140) (GPOVGE(L), L = 1, LMAX)                           A4EE0700
C   140 FORMAT (14F5.1)                                             A4EE0710
C
C STORE CONSTANTS                                                 A4EE0720
C   ANMAX = NMAX                                                 A4EE0730
C   A(1) = 1.                                                    A4EE0740
C   TAUEND(1) = L.                                              A4EE0750
C
C START COMPUTATIONAL DO LOOPS                                     A4EE0760
C   DO 670 L = 1, LMAX                                         A4EE0770
C   GAMMAR = GPOVGE(L)                                         A4EE0780
C
C ENSURE EXCLUSION OF NEGATIVE PLASTICITY IN ADHESIVE (ERROR IN DATA) A4EE0790
C EXCLUDE PURELY-ELASTIC ADHESIVE. SEPARATE PROGRAM NEEDED          A4EE0800
C   IF (GAMMAR .LE. 0.) GO TO 670                                A4EE0810
C   DO 670 I = 1, IMAX                                         A4EE0820

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THERMC(1) = CTHERM(I) A4EE0890
THERMC(2) = - THERMC(1) A4EE0900
DO 460 K = 1, KMAX A4EE0910
VR(1) = ETR(K) A4EE0920
VR(2) = 1. / VR(1) A4EE0930
VU(1) = 1. - VR(1) A4EE0940
VL(1) = 1. + VR(1) A4EE0950
VU(2) = 1. - VR(2) A4EE0960
VL(2) = 1. + VR(2) A4EE0970
A4EE0980
C ESTABLISH TRANSITIONAL OVERLAPS FROM FULLY-PLASTIC TO ELASTIC-PLASTIC A4EE0990
C 1 BEHAVIOUR AS REFERENCE LENGTH FOR START OF ITERATIONS A4EE1000
C SPECIAL PROCEDURE FOR LESS THAN COMPLETELY UNBALANCED JOINTS A4EE1010
C IF ( (THERMC(1) .EQ. 0.) AND. (VR(1) .EQ. 1.) ) GO TO 150 A4EE1020
C IF ( (THERMC(1) .EQ. 0.) GO TO 160 A4EE1030
C IF (VR(1) .EQ. 1.) GO TO 170 A4EE1040
C IF NONE OF THESE, JOINT CONTAINS BOTH IMBALANCES A4EE1050
C GO TO 180 A4EE1060
C SET INFINITE TRANSITIONAL OVERLAP FOR IDENTICAL ADHERENDS A4EE1070
150 OLTRNT(1) = 1000000. A4EE1080
OLTRNT(2) = 1000000. A4EE1090
C OLTRNC(1) = 1000000. A4EE1100
C OLTRNC(2) = 1000000. A4EE1110
C GO TO 190 A4EE1120
C SET TRANSITIONAL OVERLAPS FOR STIFFNESS IMBALANCE ONLY A4EE1130
C IN THE ABSENCE OF THERMAL MISMATCH, SAME END IS CRITICAL FOR BOTH A4EE1140
C 1 TENSILE SHEAR AND COMPRESSIVE SHEAR LOADING A4EE1150
160 IF (VU(1) .GT. 0.) OLTRNT(1) = SQRT(GAMMAR*VL(1)/VU(1)) A4EE1160
IF (VU(1) .LE. 0.) OLTRNT(1) = 1000000. A4EE1170
IF (VU(2) .GT. 0.) OLTRNT(2) = SQRT(GAMMAR*VL(2)/VU(2)) A4EE1180
IF (VU(2) .LE. 0.) OLTRNT(2) = 1000000. A4EE1190
C IF (VU(1) .GT. 0.) OLTRNC(1) = SQRT(GAMMAR*VL(1)/VU(1)) A4EE1200
IF (VU(1) .LE. 0.) OLTRNC(1) = 1000000. A4EE1210
C IF (VU(2) .GT. 0.) OLTRNC(2) = SQRT(GAMMAR*VL(2)/VU(2)) A4EE1220
IF (VU(2) .LE. 0.) OLTRNC(2) = 1000000. A4EE1230
C GO TO 190 A4EE1240
C SET TRANSITIONAL OVERLAPS FOR THERMAL MISMATCH ONLY A4EE1250
170 IF ( (THERMC(1) .LT. 0.) OLTRNT(1) = -GAMMAR/THERMC(1) A4EE1260
IF ( (THERMC(1) .GE. 0.) OLTRNT(1) = 1000000. A4EE1270
IF ( (THERMC(2) .LT. 0.) OLTRNT(2) = -GAMMAR/THERMC(2) A4EE1280
IF ( (THERMC(2) .GE. 0.) OLTRNT(2) = 1000000. A4EE1290
C IF ( (THERMC(1) .GE. 0.) OLTRNC(1) = GAMMAR/THERMC(1) A4EE1300
IF ( (THERMC(1) .LT. 0.) OLTRNC(1) = 1000000. A4EE1310
C IF ( (THERMC(2) .GE. 0.) OLTRNC(2) = GAMMAR/THERMC(2) A4EE1320
IF ( (THERMC(2) .LT. 0.) OLTRNC(2) = 1000000. A4EE1330
C GO TO 190 A4EE1340
180 CONTINUE A4EE1350
C STANDARD PROCEDURE FOR COMPLETELY UNBALANCED JOINTS A4EE1360
V1 = THERMC(1) * VL(1) / (2. * VU(1)) A4EE1370
V2 = THERMC(2) * VL(2) / (2. * VU(2)) A4EE1380
V3 = V1*V1 + GAMMAR*VL(1)/VU(1) A4EE1390
V4 = V2*V2 + GAMMAR*VL(2)/VU(2) A4EE1400
C ESTABLISH TRANSITIONAL OVERLAPS BELOW WHICH JOINT IS FULLY PLASTIC A4EE1410
C NEXT FOUR STATEMENTS APPLY FOR TENSILE SHEAR LOADING A4EE1420
IF (V3 .GE. 0.) OLTRNT(1) = V1 + SQRT(V3) A4EE1430
C IF NOT, OTHER END OF JOINT CRITICAL A4EE1440
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT A4EE1450
C SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS A4EE1460
IF ( (V3 .LT. 0.) .OR. (OLTRNT(1) .LE. 0.) ) OLTRNT(1) = 1000000. A4EE1470
IF (V4 .GE. 0.) OLTRNT(2) = V2 + SQRT(V4) A4EE1480
C IF NOT, OTHER END OF JOINT CRITICAL A4EE1490
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT A4EE1500
C SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS A4EE1510
IF ( (V4 .LT. 0.) .OR. (OLTRNT(2) .LE. 0.) ) OLTRNT(2) = 1000000. A4EE1520
C IF ICRTND .EQ. 2 FOR SHORT OVERLAPS, OLTRNT(1) WILL BE COMPUTED VERY A4EE1530
C 1 LARGE, AND VICE VERSA A4EE1540
C THIS IS PHYSICALLY REALISTIC AND DOES NOT LEAD TO IMPOSSIBLE COMPUTING A4EE1550
C IF BOTH V3 AND V4 ARE POSITIVE, EITHER OLTRNT(1) OR OLTRNT(2) WILL BE A4EE1560
C 1 COMPUTED NEGATIVE. NEED TO PREVENT COMPUTATIONS BASED ON THIS A4EE1570
C 2 UNREAL SITUATION. HENCE CHECKS ABOVE AND BELOW A4EE1580
C NEXT FOUR STATEMENTS WOULD APPLY FOR COMPRESSIVE SHEAR LOADING A4EE1590
IF (V3 .GF. 0.) OLTRNC(1) = -V1 + SQRT(V3) A4EE1600
C IF NOT, OTHER END OF JOINT CRITICAL A4EE1610
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT A4EE1620
SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS A4EE1630
IF ( (V3 .LT. 0.) .OR. (OLTRNC(1) .LE. 0.) ) OLTRNC(1) = 1000000. A4EE1640
IF (V4 .GE. 0.) OLTRNC(2) = -V2 + SQRT(V4) A4EE1650
C IF NOT, OTHER END OF JOINT CRITICAL A4EE1660
C OTHER END OF JOINT IDENTIFIED AS CRITICAL BY SHEAR STRAIN GRADIENT A4EE1670
C SET INFINITE TRANSITIONAL OVERLAP TO ACCOUNT FOR THIS A4EE1680
IF ( (V4 .LT. 0.) .OR. (OLTRNC(2) .LE. 0.) ) OLTRNC(2) = 1000000. A4EE1690
C 190 DO 380 NCRTND = 1, 2 A4EE1700
THERM = THERMC(NCRTND) A4EE1720
VRREF = VR(NCRTND) A4EE1730
VUREF = VU(NCRTND) A4EE1740
VLREF = VL(NCRTND) A4EE1750
C SET UNIFORM STRESS FOR SHORT OVERLAPS A4EE1760

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DO 200 J = 2, JMAX A4EE1770
  JSAVE = J A4EE1780
  IF (OL(J) .GT. OLTRNT(NCRTND)) GO TO 210 A4EE1790
C IF NOT, JOINT IS FULLY PLASTIC A4EE1800
  200 TRATIO(J,NCRTND) = 1. A4EE1810
  IF (JSAVE .EQ. JMAX) GO TO 380 A4FF1820
C COMPUTE JOINT STRENGTH FOR ELASTIC-PLASTIC ADHESIVE BEHAVIOUR A4EE1830
  210 DO 380 J = JSAVE, JMAX A4FF1840
    OLAP = OL(J) A4EE1850
    OLAP2 = OLAP * OLAP A4EE1860
C COMPUTE ADOVERL FOR MINIMUM VALUE OF TAVOTP BY ITERATION A4EE1870
C SET INITIAL ESTIMATE OF EXTENT OF PLASTIC ZONE FROM TRANSITIONAL OLAP A4EF1880
  ADOVERL = OLTRNT(NCRTND) / OLAP A4EE1890
  N2MAX = 2 * NMAX A4EE1900
  DO 220 N = 1, N2MAX A4EE1910
    ARMDR = 1. - ADOVERL A4EE1920
    ADOVERL = -ARMDR*ALOG(ARMDR) + (GAMMAR / ((VUREF/VLREF)*OLAP2 A4EE1930
      1 - THERM * OLAP)) A4EE1940
    IF (ADOVERL .GT. 0.9999) ADOVERL = 0.9999 A4EE1950
    IF (ADOVERL .LT. 0.001) ADOVERL = 0.001 A4EE1960
  220 CONTINUE A4EE1970
    TRATIO(J,NCRTND) = 1. A4EE1980
    IF (ADOVERL .EQ. 0.9999) GO TO 230 A4EE1990
C COMPUTE CORRESPONDING AVERAGE SHEAR STRESS A4EE2000
    TAUREF = 1. - (VLREF*GAMMAR/OLAP2 + (VLREF*THERM/OLAP - VUREF) * A4FF2010
      1 - ADOVERL) / ALOG(1.-ADOVERL) A4EE2020
    TRATIO(J,NCRTND) = TAUREF A4EE2030
  230 CONTINUE A4EE2040
    AREF = ADOVERL * 0.999 A4EE2050
C THE FACTOR IS TO PREVENT DIVERGENCE IN THE SERIES COEFFICIENTS A4EE2060
C SET MINIMJ4 POSSIBLE VALUE OF ADOVERL, AT WHICH TAVOTP .EQ. 1. A4EE2070
  AMIN = GAMMAR / ((VUREF/VLREF)*OLAP2 - THERM*OLAP) A4EE2080
C TRUE EXTENT OF FIRST PLASTIC ZONE BOUNDED WITHIN AMIN AND AREF A4EE2090
  ADEL = (AREF - AMIN) / (ANMAX - 1.) A4EE2100
C MINIMUM VALUES OF TAVOTP ARE NOW COMPUTED. THESE APPROXIMATE THE TRUE A4EE2120
C 1 SOLUTIONS FOR ALL BUT SHORT OVERLAPS OR THICK ADHERENDS A4EE2130
C 2 IN CONJUNCTION WITH SEVERE ADHEREND MISMATCH AND/OR BRITTLE A4EE2140
C 3 ADHESIVES. REFINER ANSWER BY PRECISE SOLUTION IN POWER SERIES A4EE2150
C COMPUTE JOINT STRENGTH FOR ELASTIC-PLASTIC ADHESIVE BEHAVIOUR A4EE2160
  DO 360 M = 1, NMAX A4EE2170
    AM = M A4EE2180
    ADOVERL = AREF - (AM - 1.)*ADEL A4EE2190
    ARMDR = 1. - ADOVERL A4EE2200
C COMPUTE ASSOCIATED AVERAGE BOND STRESS A4EE2210
    TAVOTP(M) = 1. - (VLREF*GAMMAR/OLAP2 + (VLREF*THERM/OLAP - VUREF) A4EE2220
      1 * ADOVERL) / ALOG(ARMDR) A4EE2230
C START COMPUTING ELASTIC STRESS SERIES A4EE2240
  A(1) = 1. A4EE2250
C ESTABLISH A(2) AT START OF ELASTIC ZONE FROM CONTINUITY OF SHEAR A4EE2260
  C 1 STRAINS IN ADHESIVE AT TRANSITION. THIS ENSURES ADHEREND STRESS A4EE2270
  C 2 CONTINUITY A4EE2280
  A(2) = THERM*OLAP - OLAP2*(VUREF - (1.-TAVOTP(M))/ARMDR) / VLREF A4EE2290
C A(2) SHOULD BE .LT. 0. FOR ADOVERL .LT. AREF A4EE2310
C A(2) SHOULD BE .EQ. 0. FOR ADOVERL .EQ. AREF A4EE2320
C A(2) SHOULD BE .GT. 0. FOR ADOVERL .GT. AREF A4EE2330
  A(3) = (A(2) - THERM*OLAP + OLAP2*VUREF/VLREF) / (2.*ARMDR) A4EE2340
C CONVERT STRESS TERMS INTO AVERAGE STRESS TERMS BY DIVIDING BY N A4EE2350
  A(2) = A(2) / 2. A4EE2360
  A(3) = A(3) / 3. A4EE2370
C COMPUTE SUBSEQUENT TERMS FROM RECURRENCE FORMULA A4EE2380
  DO 240 N = 4, NMAY A4EE2390
    NSAVE = N A4EE2400
    AN = N A4EE2410
    A(N) = ((2.*ADOVERL - 1.)*(AN-2.)*(AN-1.)*A(N-1) + A4EE2420
      1 (AN-3.)*(AN-2.)*A(N-2) + (OLAP2/VLREF)* A4EE2430
      2 ((ADOVERL*VUREF + VRREF)*A(N-2) + VUREF*A(N-3)) / A4EE2440
      3 (ADOVERL*ARMDR*(AN-1.)*AN) A4EE2450
    IF (ABS(A(N)) .LT. 1.E50) GO TO 240 A4EE2460
C IF NOT, OVERFLOW IS IMMINENT, SO CUT DOWN ON NMAY A4EE2470
    GO TO 250 A4EE2480
  240 CONTINUE A4EE2490
    GO TO 270 A4EE2500
  250 DO 260 N = NSAVE, NMAY A4EE2510
    260 A(N) = 0. A4EE2520
C ESTIMATE ELASTIC ADHESIVE STRESS AT OTHER END OF JOINT, OR IDENTIFY A4EE2530
  C 1 EXISTENCE OF SECOND PLASTIC ADHESIVE ZONE, AS APPROPRIATE A4EE2540
  C START BY ASSUMING NO SECOND PLASTIC ADHESIVE ZONE A4EE2550
  270 COVERL = 0. A4EE2560
C TAUEND(1) = 1. A4EE2570
  TAUEND(NMAX) = 1. A4EE2580
  DO 280 N = 2, NMAY A4EE2590
    AN = N A4EE2600
  280 TAUEND(NMAX) = TAUEND(NMAX) + A(N)*(ARMDR**(N-1))*AN A4EE2610
    IF (TAUEND(NMAX) .LE. 1.) GO TO 310 A4EE2620
C IF SO, ONLY THE ONE PLASTIC ZONE, AT THE NCRTND REFERENCE END A4EE2630
C IF NOT, HAVE IDENTIFIED EXISTENCE OF SECOND PLASTIC ZONE, AT OTHER ENDA4EE2640

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C PROCEDURE FOR SECOND PLASTIC ZONE A4EE2650
C USE LINEAR INTERPOLATION TO ESTIMATE COVERL (EXTENT OF SECOND PLASTIC A4EE2660
C 1 ZONE) A4EE2670
    DELBOL = (1. - AOVERL) / (ANMAX - 1.) A4EE2680
    AOVERL(1) = 1. - AOVERL A4EE2690
    DO 300 N1 = 2, NMAX A4EE2700
    AOVERL(N1) = AOVERL(N1-1) + DELBOL A4EE2710
    TAUEND(N1) = 1. A4EE2720
    DO 290 N = 2, NMAX A4EE2730
    AN = N A4EE2740
    V = AOVERL(N1)**(N-1) A4EE2750
    290 TAUEND(N1) = TAUEND(N1) + A(N)*V*AN A4EE2760
C CHECK FOR CONVERGENCE OF COVERL A4EE2770
    IF ((1.0001.GT.TAUEND(N1)).AND.(0.9999.LT.TAUEND(N1))) GO TO 310 A4EE2780
C IF NOT, ITERATE ON COVERL A4EE2790
C COMPUTE VALUE OF COVERL NEEDED TO RESTRICT STRESSES TO ELASTIC LEVEL A4EE2800
    IF (TAUEND(N1).LT.1.) GO TO 300 A4EE2810
C IF SO, ESTIMATE OF AOVERL IS INSUFFICIENT AND THAT OF COVERL EXCESSIVE A4EE2820
C IF NOT, CORRECT TRANSITION LIFES BETWEEN N1 AND N1-1 LOCATIONS A4EE2830
    COVERL = 1. - AOVERL - AOVERL(N1-1) - A4EE2840
    1 ((1. - TAUEND(N1-1)) / (TAUEND(N1) - TAUEND(N1-1))) A4EE2850
    GO TO 310 A4EE2860
300 CONTINUE A4EE2870
C OFFER CHECK ON WHETHER COVERL IS SO LARGE THAT CRITICAL END OF JOINT A4EE2880
C 1 IS AT OTHER END UNTIL AFTER CONVERGENCE OF AOVERL IS ESTABLISHED A4EE2890
C 310 SAVOTP(M) = 1. A4EE2900
    BOVL = 1. - AOVERL - COVERL A4EE2910
C EVALUATE AVERAGE STRESS IN TERMS OF SERIES COEFFICIENTS A4EE2920
    DO 320 N = 2, NMAX A4EE2930
    AN = N A4EE2940
    320 SAVOTP(M) = SAVOTP(M) + A(N)*(BOVL**N) A4EE2950
C CHECK ON CONVERGENCE OF AOVERL A4EE2960
    IF (SAVOTP(M) .GT. 1.) GO TO 360 A4EE2970
C IF SO, CANNOT HAVE CONVERGED YET A4EE2980
    IF ((SAVOTP(M) .LT. TAVOTP(M)).AND.(M.EQ.1)) GO TO 330 A4EE2990
C IF SO, SOLUTION IS NUMERICALLY INDISTINGUISHABLE FROM THE LOWER BOUND A4EE3000
C NEED BOTH M.EQ.1 VALUES AND M.EQ.2 VALUES FOR FIRST CHECK A4EE3010
    IF (M.EQ.1) GO TO 360 A4EE3020
C PROTECT AGAINST DIVISION BY ZERO A4EE3030
    IF ((TAVOTP(M) .EQ. 0.).AND.(SAVOTP(M) .EQ. 0.)) GO TO 340 A4EE3040
C IF SO, CONVERGENCE ESTABLISHED A4EE3050
    IF ((SAVOTP(M) .LT. 0.00001).AND.(SAVOTP(M) .GT. -0.00001)) A4EE3060
    1 RATIO = 1. + TAVOTP(M) A4EE3070
    IF ((TAVOTP(M) .LT. 0.00001).AND.(TAVOTP(M) .GT. -0.00001)) A4EE3080
    1 RATIO = 1. + SAVOTP(M) A4EE3090
C IF NONE OF THE ABOVE, NO FURTHER FAILURE CASES LEFT TO CHECK FOR A4EE3100
    RATIO = SAVOTP(M) / TAVOTP(M) A4EE3110
C CHECK ON CONVERGENCE OF JOINT STRENGTH PREDICTIONS A4EE3120
    IF ((1.0001.GT.RATIO).AND.(0.9999.LT.RATIO)) GO TO 350 A4EE3130
C IF SO, CONVERGENCE IS ESTABLISHED A4EE3140
C IF NOT, NEED TO RE-ESTIMATE AOVERL A4EE3150
C USE LINEAR INTERPOLATION TO ESTIMATE AOVERL (EXTENT OF FIRST PLASTIC A4EE3160
C 1 ZONE) A4EE3170
    IF (SAVOTP(M) .GT. TAVOTP(M)) GO TO 360 A4EE3180
C IF SO, CONVERGENCE OF AOVERL NOT YET ESTABLISHED A4EE3190
C IF NOT, CORRECT VALUE OF AOVERL LIES BETWEEN M AND M-1 LOCATIONS A4EE3200
    TRATIO(J,NCRTND) = TAVOTP(M-1) + (TAVOTP(M) - TAVOTP(M-1)) * A4EE3210
    1 (1. - TAVOTP(M-1)) / SAVOTP(M-1) / A4EE3220
    2 (1. - (SAVOTP(M) - TAVOTP(M) + TAVOTP(M-1)) / SAVOTP(M-1)) A4EE3230
    GO TO 370 A4EE3240
330 TRATIO(J,NCRTND) = TAURFF A4EE3250
    GO TO 370 A4EE3260
340 TRATIO(J,NCRTND) = 0. A4EE3270
    GO TO 370 A4EE3280
350 TRATIO(J,NCRTND) = TAVOTP(M) A4EE3290
    GO TO 370 A4EE3300
360 CONTINUE A4EE3310
C IF REFINEMENT HAS NOT CONVERGED, USE LOWER BOUND ESTIMATE A4EE3320
    TRATIO(J,NCRTND) = TAVOTP(1), AS SET EARLIER A4EE3330
C PROTECT AGAINST ACCUMULATED NUMERICAL ERRORS A4EE3340
C USE LOWER BOUND SOLUTION IF REFINEMENT RESULTS IN STILL LOWER VALUES A4EE3350
    370 IF (TRATIO(J,NCRTND) .LT. TAVOTP(1)) TRATIO(J,NCRTND) = TAVOTP(1) A4EE3360
    380 CONTINUE A4EE3370
C CONVERGENCE OF AOVERL ESTABLISHED. RECORD AVERAGE SHEAR STRESS A4EE3380
C VALUES COMPUTED ARE NOW STORED IN TRATIO(J,NCRTND) A4EE3390
C NEED TO SELECT LOWER VALUE TO IDENTIFY CRITICAL END OF JOINT A4EE3400
    DO 450 J = 2, JMAX A4EE3410
    OLAP = OL(J) A4EE3420
    TAU1 = TRATIO(J,1) A4EE3430
    TAU2 = TRATIO(J,2) A4EE3440
    IF ((TAU1 .LT. 1.) .OR. (TAU2 .LT. 1.)) GO TO 390 A4EE3450
C IF SO, JOINT IS NOT FULLY PLASTIC A4EE3460
C IF NOT, IDENTIFY CRITICAL END OF JOINT FROM SHEAR STRAIN GRADIENT A4EE3470
    GRADNT = THERMC(1) - OLAP*VU(1)/VL(1) A4EE3480

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IF (GRADNT .LT. 0.) ICRTND(J,K) = 1 A4EE3530
IF (GRADNT .EQ. 0.) ICRTND(J,K) = 0 A4EE3540
IF (GRADNT .GT. 0.) ICRTND(J,K) = 2 A4EE3550
TAUAVG(J,K) = 1 A4EE3560
STRGTH(J,K) = OLAP A4EE3570
MCRTND = ICRTND(J,K) A4EE3580
IF (MCRTND .EQ. 0) MCRTND = 1 A4EE3590
TRANSL(K) = OLTRNT(MCRTND) A4EE3600
GO TO 450 A4EE3610
390 DIFFNC = TAU1 - TAU2 A4EE3620
C IF DIFFNC .LT. 0., NCRTND .EQ. 1 A4EE3630
C IF DIFFNC .EQ. 0., NCRTND .EQ. 0 A4EE3640
C IF DIFFNC .GT. 0., NCRTND .EQ. 2 A4EE3650
IF (DIFFNC)400,410,420 A4EE3660
C ADHEREND (1) END OF JOINT CRITICAL A4EE3670
400 TAUAVG(J,K) = TAU1 A4EE3680
STRGTH(J,K) = TAU1 * OLAP A4EE3690
ICRTND(J,K) = 1 A4EE3700
C COVER SITUATION WHERE TRANSITIONAL LENGTH IS LESS THAN OL(2) A4EE3710
IF (J .EQ. 2) TRANSL(K) = OLTRNT(1) A4EE3720
GO TO 430 A4EE3730
C BOTH ENDS OF JOINT EQUALLY CRITICAL FROM NULLIFYING (OR ZERO) A4EE3740
C 1 ADHEREND IMBALANCES A4EE3750
410 TAUAVG(J,K) = TAU1 A4EE3760
STRGTH(J,K) = TAU1 * OLAP A4EE3770
ICRTND(J,K) = 0 A4EE3780
C COVER SITUATION WHERE TRANSITIONAL LENGTH IS LESS THAN OL(2) A4EE3790
IF (J .EQ. 2) TRANSL(K) = OLTRNT(1) A4EE3800
GO TO 430 A4EE3810
C ADHEREND (2) END OF JOINT CRITICAL A4EE3820
420 TAUAVG(J,K) = TAU2 A4EE3830
STRGTH(J,K) = TAU2 * OLAP A4EE3840
ICRTND(J,K) = 2 A4EE3850
C COVER SITUATION WHERE TRANSITIONAL LENGTH IS LESS THAN OL(2) A4EE3860
IF (J .EQ. 2) TRANSL(K) = OLTRNT(2) A4EE3870
C COVER CASES OF ZERO OR NEGATIVE ESTIMATED STRENGTHS A4EE3880
430 IF (TAUAVG(J,K) .GT. 0.) GO TO 440 A4EE3890
C IF NOT, JOINT HAS BROKEN DUE TO THERMAL STRESSES WITHOUT EXTERNAL LOAD A4EE3900
TAUAVG(J,K) = 0. A4EE3910
STRGTH(J,K) = 0. A4EE3920
GO TO 450 A4EE3930
440 IF (TAUAVG(J,K) .LE. 1.) GO TO 450 A4EE3940
C IF NOT, THERE HAS BEEN A COMPUTATIONAL MISTAKE A4EE3950
C PRINT ASTERISKS TO IDENTIFY ERROR A4EE3960
C RERUN WITH GREATER VALUE OF NMAX A4EE3970
TAUAVG(J,K) = 100. A4EE3980
STRGTH(J,K) = 1000. A4EE3990
450 CONTINUE A4EE4000
460 CONTINUE A4EE4010
C SET UNIFORM STRESS FOR ZERO OVERLAP A4EE4020
DO 470 K = 1, KMAX A4EE4040
TAUAVG(1,K) = 1. A4EE4050
STRGTH(1,K) = 0. A4EE4060
470 ICRTND(1,K) = ICRTND(2,K) A4EE4070
C HENCE NEED FOR OL(2) TO BE SMALL ENOUGH TO BE LESS THAN THAT AT WHICH A4EE4080
1 NCRTND CHANGES A4EE4090
A4EE4100
C END OF COMPUTATIONS. START PRINTING OUT OF TABULATED RESULTS A4EE4110
A4EE4120
C PRINT OUT AVERAGE STRESS HEADING A4EE4130
WRITE (6,480) A4EE4140
480 FORMAT (1H1/, 5(1H0/), 27X, 56HADHESIVE-BONDED SCARF JOINTS (ELAS A4EE4150
1TIC-PLASTIC ANALYSIS), A4EE4160
2 39X, 31HNND-DIMENSIONALIZED FORMULATION/) A4EE4170
WRITE (6,490) GAMMAR A4EE4180
490 FORMAT (1H0, 27X, 49HPLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO A4EE4190
1 = , F5.2) A4EE4200
IF (CTHERM(I) .NE. 0.) GO TO 510 A4EE4210
WRITE (6,500) A4EE4220
500 FORMAT (1H, 37X, 33HZERO THERMAL MISMATCH COEFFICIENT) A4EE4230
GO TO 530 A4EE4240
510 WRITE (6,520) THERMC(1), THERMC(2) A4EE4250
520 FORMAT (1H, 16X, 31HTHERMAL MISMATCH COEFFICIENT = , F6.3, A4EE4260
1 17H FOR TENSION, = , F6.3, 16H FOR COMPRESSION) A4EE4270
530 WRITE (6,540) (ETR(K), K = 1, KMAX) A4EE4280
540 FORMAT (1H0, 67X, 30HO = BOTH ENDS EQUALLY CRITICAL/, 20X, A4EE4290
1 72HAVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS , 1 = SOFT ET E A4EE4300
2ND CRITICAL/, 68X, 25H2 = STIFF ET END CRITICAL/, A4EE4310
3 8HO SCALED, 31X, 39HEXTENSINAL STIFFNESS (THICKNESS) RATIO/, A4EE4320
4 7H L/T/, 7H RATIO, F7.1, 9F10.1/, 1H ) A4EE4330
C WRITE OUT TABULATIONS OF AVERAGE BOND STRESSES A4EE4340
DO 560 J = 1, JMAX A4EE4350
WRITE (6,550) OL(J), ((TAUAVG(J,K), ICRTND(J,K)), K = 1, KMAX) A4EE4360
550 FORMAT (1H , F6.2, 2X, 10(F7.5, 1X, 1I, 1X)) A4EE4370
560 CONTINUE A4EE4380
C PRINT OUT JOINT STRENGTH HEADING A4EE4390
A4EE4400

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      WRITE (6,570)                                     A4EE4410
570 FORMAT (1H/, 5(1H0/), 27X, 56HADHESIVE-BONDED SCARF JOINTS (ELAS A4EE4420
1TIC-PLASTIC ANALYSIS)/,                           A4EE4430
2 30X, 31HNON-DIMENSIONALIZED FORMULATION/)       A4EE4440
      WRITE (6,580) GAMMAR                         A4EE4450
580 FORMAT(1H0, 27X, 49HPLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO A4EE4460
1 = F5.2)                                         A4EE4470
1 IF (CTHERM(I) .NE. 0.) GO TO 600               A4EE4480
      WRITE (6,590)                                     A4EE4490
590 FORMAT (1H , 37X, 33HZERO THERMAL MISMATCH COEFFICIENT) A4EE4500
GO TO 620                                         A4EE4510
600 WRITE (6,610) THERMC(1), THERMC(2)           A4EE4520
610 FORMAT (1H , 16X, 31HTHERMAL MISMATCH COEFFICIENT = , F6.3, A4EE4530
1 !TH FOR TENSION, = , F6.3, 16H FOR COMPRESSION) A4EE4540
620 WRITE (6,630) (ETR(K), K = 1, KMAX)          A4EE4550
630 FORMAT( 1H0, 67X, 30H0 = BOTH ENDS EQUALLY CRITICAL/, 20X, A4EE4560
1 72HN0N-DIMENSIONALIZED JOINT STRENGTH          1 = SOFT ET EA4EE4570
2ND CRITICAL/, 68X, 25H2 = STIFF ET END CRITICAL/, A4EE4580
3 8H0 SCALED, 31X, 39HEXTENSIONAL STIFFNESS (THICKNESS) RATIO/, A4EE4590
4 7H L/T/, 7H RATIO, F7.1, 9F10.1/, 1H )        A4EE4600
C WRITE OUT TABULATIONS OF JOINT STRENGTHS         A4EE4610
DO 650 J = 1, JMAX                               A4EE4620
      WRITE (6,640) DL(J), ((STPGTH(J,K), ICRTND(J,K)), K = 1, KMAX) A4EE4630
640 FORMAT (1H , F6.2, 2X, 10(F7.4, 1X, 1I, 1X)) A4EE4640
650 CONTINUE                                       A4EE4650
C WRITE OUT TRANSITIONAL JOINT STRENGTHS          A4EE4660
      WRITE (6,660) (TRANSL(K), K = 1, KMAX)        A4EE4670
660 FORMAT (8H0 TRANSL, 1X, 10(F7.4, 3X))        A4EE4680
670 CONTINUE                                       A4EE4690
C
      WRITE (6,680)                                     A4EE4700
680 FORMAT (1H1, 18H PROGRAM COMPLETED)           A4EE4710
STOP                                              A4EE4720
END                                              A4EE4730
                                         A4EE4740

```

ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = 1.000 FOR TENSION, = -1.000 FOR COMPRESSION  
NON-DIMENSIONALIZED JOINT STRENGTH , O = BOTH ENDS EQUALY CRITICAL  
1 = SOFT ET END CRITICAL  
2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	0.0	2	0.0	2	0.0	2	0.0	2	0.0	2
0.20	0.2000	2	0.2000	2	0.2000	2	0.2000	2	0.2000	2
0.50	0.5000	2	0.5000	2	0.5000	2	0.5000	2	0.5000	2
1.00	1.0000	2	1.0000	2	1.0000	2	1.0000	2	1.0000	2
1.20	1.2000	2	1.2000	2	1.2000	2	1.2000	2	1.2000	2
1.50	1.5000	2	1.5000	2	1.5000	2	1.5000	2	1.5000	2
1.70	1.7000	2	1.7000	2	1.7000	2	1.7000	2	1.7000	2
2.00	2.0000	2	2.0000	2	2.0000	2	2.0000	2	2.0000	2
2.50	2.5000	2	2.5000	2	2.5000	2	2.5000	2	2.5000	2
3.00	3.0000	2	3.0000	2	3.0000	2	3.0000	2	3.0000	2
4.00	3.6484	1	3.9998	1	4.0000	1	4.0000	1	4.0000	1
5.00	4.0026	1	4.3962	1	4.7267	1	5.0000	1	5.3000	1
6.00	4.2596	1	4.7826	1	5.2518	1	5.6524	1	5.9999	1
8.00	4.5376	1	5.3763	1	6.1055	1	6.7624	1	7.3451	1
10.00	4.8582	1	5.512	1	6.8081	1	7.7193	1	8.5517	1
12.00	5.1248	1	6.3263	1	7.4862	1	8.6009	1	9.6666	1
15.00	5.4906	1	7.0028	1	8.4752	1	9.9027	1	11.2764	1
17.00	5.7213	1	7.4384	1	9.1166	1	10.7510	1	12.3334	1
20.00	6.0557	1	8.0782	1	10.0626	1	12.0046	1	13.8965	1
25.00	6.5943	1	9.1229	1	11.6144	1	14.0647	1	16.4670	1
30.00	6.1200	1	10.1525	1	13.1486	1	16.1044	1	19.0134	1
35.00	7.6382	1	11.1736	1	14.6729	1	18.1325	1	21.5462	1
40.00	8.1518	1	12.1893	1	16.1911	1	20.1535	1	24.0706	1
45.00	8.6624	1	13.2016	1	17.7052	1	22.1698	1	26.5898	1
50.00	9.1709	1	14.2113	1	19.2164	1	24.1928	1	29.1047	1
TRANSL	3.1576		3.5895		4.1142		4.7761		5.6533	
										6.8990
										8.8633
										12.5777
										23.1106
										5.0000

ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = 1.000 FOR TENSION, = -1.000 FOR COMPRESSION  
AVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS , O = BOTH ENDS EQUALY CRITICAL  
1 = SOFT ET END CRITICAL  
2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
0.20	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
0.50	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.00	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.20	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.50	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
1.70	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
2.00	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
2.50	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
3.00	1.00000	2	1.00000	2	1.00000	2	1.00000	2	1.00000	2
4.00	0.91211	1	0.99994	1	1.00000	1	1.00000	1	1.00000	1
5.00	0.80052	1	0.87925	1	0.94534	1	0.99999	1	1.00000	1
6.00	0.70996	1	0.79709	1	0.87531	1	0.94206	1	0.99999	1
8.00	0.57345	1	0.67203	1	0.76319	1	0.84530	1	0.91814	1
10.00	0.44582	1	0.58512	1	0.68081	1	0.77193	1	0.85517	1
12.00	0.42707	1	0.52713	1	0.62385	1	0.71675	1	0.80555	1
15.00	0.36604	1	0.46685	1	0.56502	1	0.66018	1	0.75176	1
17.00	0.33655	1	0.43755	1	0.53627	1	0.63241	1	0.72549	1
20.00	0.30278	1	0.40391	1	0.50313	1	0.6023	1	0.69483	1
25.00	0.26377	1	0.36492	1	0.46458	1	0.56259	1	0.65868	1
35.00	0.21823	1	0.31925	1	0.43829	1	0.53681	1	0.63378	1
40.00	0.20380	1	0.30473	1	0.40478	1	0.50384	1	0.60177	1
45.00	0.19250	1	0.29337	1	0.39345	1	0.49266	1	0.59088	1
50.00	0.18342	1	0.28423	1	0.38433	1	0.49366	1	0.58209	1
										0.67945
										0.77538
										0.86914
										0.95840
										0.97650

ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = -1.000 FOR TENSION, = 1.000 FOR COMPRESSION  
NON-DIMENSIONALIZED JOINT STRENGTH :      0 = BOTH ENDS EQUIALLY CRITICAL  
    1 = SOFT ET END CRITICAL  
    2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	0.0	1	0.0	1	0.0	1	0.0	1	0.0	1
0.20	0.2000	1	0.2000	1	0.2000	1	0.2000	1	0.2000	1
0.50	0.5000	1	0.5000	1	0.5000	1	0.5000	1	0.5000	1
1.00	1.0000	1	1.0000	1	1.0000	1	1.0000	1	1.0000	1
1.20	1.2000	1	1.2000	1	1.2000	1	1.2000	1	1.2000	1
1.50	1.5000	1	1.5000	1	1.5000	1	1.5000	1	1.5000	1
1.70	1.7000	1	1.7000	1	1.7000	1	1.7000	1	1.7000	1
2.00	2.0000	1	2.0000	1	2.0000	1	2.0000	1	2.0000	1
2.50	2.4992	1	2.4996	1	2.4999	1	2.5000	1	2.5000	1
3.00	2.3553	1	2.3529	1	2.3599	1	2.3995	1	3.0000	1
4.00	2.5137	1	2.7894	1	3.0434	1	3.2754	1	3.8805	1
5.00	2.6328	1	3.0062	1	3.3544	1	3.6794	1	3.9806	1
6.00	2.6965	1	3.1912	1	3.6503	1	4.0727	1	4.4663	1
8.00	2.8421	1	3.5216	1	4.1808	1	4.8180	1	5.4241	1
10.00	3.0443	1	3.9160	1	4.7601	1	5.5762	1	6.3663	1
12.00	3.2662	1	4.3151	1	5.3556	1	6.3665	1	7.3455	1
15.00	3.5475	1	4.9134	1	6.2500	1	7.5558	1	8.8282	1
17.00	3.7478	1	5.3123	1	6.8469	1	8.3502	1	9.8194	1
20.00	4.0681	1	5.9108	1	7.7433	1	9.5437	1	11.3090	1
25.00	4.5482	1	6.9290	1	9.2397	1	11.5357	1	13.7965	1
30.00	5.0482	1	7.9076	1	10.7355	1	13.5301	1	16.2877	1
35.00	5.5482	1	8.9065	1	12.2331	1	15.5259	1	18.7812	1
40.00	6.0484	1	9.9058	1	13.7313	1	17.5227	1	21.2763	1
45.00	6.5487	1	10.9053	1	15.2299	1	19.5203	1	23.7723	1
50.00	7.0494	1	11.9051	1	16.7289	1	21.5183	1	26.2692	1
TRANSL	1.9354	2.0895	2.2570	2.4427	2.6533	2.8990	3.1967	3.5777	4.1107	5.0000

ADHESIVE-BONDED SCARF JOINTS (ELASTIC-PLASTIC ANALYSIS)  
NON-DIMENSIONALIZED FORMULATION

PLASTIC TO ELASTIC ADHESIVE SHEAR STRAIN RATIO = 5.0  
THERMAL MISMATCH COEFFICIENT = -1.000 FOR TENSION, = 1.000 FOR COMPRESSION  
AVERAGE SHEAR STRESS / MAXIMUM SHEAR STRESS :      0 = BOTH ENDS EQUIALLY CRITICAL  
    1 = SOFT ET END CRITICAL  
    2 = STIFF ET END CRITICAL

SCALED L/T RATIO	EXTENSIVE STIFFNESS (THICKNESS) RATIO									
	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
0.0	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
0.20	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
0.50	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
1.00	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
1.20	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
1.50	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
1.70	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
2.00	0.0000	1	0.0000	1	0.0000	1	0.0000	1	0.0000	1
2.50	0.99967	1	0.99984	1	0.99996	1	1.00000	1	1.00000	1
3.00	0.78511	1	0.84430	1	0.99968	1	0.99984	1	0.99995	1
4.00	0.62842	1	0.69715	1	0.76086	1	0.81986	1	0.87112	1
5.00	0.52655	1	0.60124	1	0.67088	1	0.73587	1	0.79611	1
6.00	0.424942	1	0.53186	1	0.60838	1	0.67878	1	0.74438	1
8.00	0.35526	1	0.44120	1	0.52260	1	0.60225	1	0.67901	1
10.00	0.30443	1	0.39160	1	0.47601	1	0.55762	1	0.63663	1
12.00	0.27051	1	0.35950	1	0.44630	1	0.53054	1	0.61213	1
15.00	0.23650	1	0.32756	1	0.41666	1	0.50372	1	0.58955	1
17.00	0.20246	1	0.31249	1	0.40276	1	0.49119	1	0.57761	1
20.00	0.17241	1	0.29554	1	0.37916	1	0.47718	1	0.56545	1
25.00	0.14193	1	0.27636	1	0.36955	1	0.46143	1	0.55186	1
30.00	0.116827	1	0.26359	1	0.35785	1	0.45100	1	0.54292	1
35.00	0.108852	1	0.25447	1	0.34952	1	0.44360	1	0.53661	1
40.00	0.102521	1	0.24764	1	0.34328	1	0.4307	1	0.53191	1
45.00	0.104553	1	0.24234	1	0.33844	1	0.43378	1	0.52827	1
50.00	0.104099	1	0.23810	1	0.33458	1	0.43037	1	0.52538	1

#### A.4 Computer Program A4EF For Elastic Strength of Stepped-Lap Bonded Joints

The analysis in Section 5 has been prepared as the FORTRAN IV digital computer program A4EF. The program computes the elastic joint strength of any stepped-lap bonded joint and prints out the most critical adherend and adhesive stresses for each step of the joint. In order to obtain a more complete internal stress distribution, each step can be subdivided and a series of shorter steps input instead. The input data is printed out to supplement the solution output. Eccentricities are excluded from the joint and a symmetric two-sided bonded joint is analyzed in which the thicknesses of the two outer adherends are lumped together in evaluating the joint strengths. The reason for this is the greater utilization of the back-to-back stepped-lap joint than of the single-sided joint. A single-sided joint can be analyzed with this program in one of two ways. One can add a mirror image of the actual joint and halve the strength predicted for this joint of twice the actual thickness and twice the bond area or one can change certain factors of 2, identified in the listing, to 1 for single-sided joints. The program accounts for arbitrary combinations of adherend stiffness and thermal imbalances as well as non-uniform step thickness increments and step lengths. It has been used successfully in optimizing the joint proportions in order to maximize the joint strength.

A complete listing of the program A4EF follows after the input and output have been described.

CARD 1:

FORMAT (I2)

M = Number of configurations (each requiring a complete set of data) to be solved.

CARDS 2, 2A:

FORMAT (8F10.3)

TAUMAX =  $\tau_p$  = Peak adhesive shear stress.

G = Elastic adhesive shear modulus.

GAMMAX =  $\gamma_e + \gamma_p$  = Maximum adhesive shear strain. (This may be set less than  $\gamma_e$  to cover partial loads.)

GAMMAE =  $\gamma_e$  = Elastic adhesive shear strain.

ETA =  $\eta$  = Bond line thickness.

ALPHAO =  $\alpha_o$  = Coefficient of thermal expansion of outer adherend.

ALPHA1 =  $\alpha_i$  = Coefficient of thermal expansion of inner adherend.

DELTMP =  $\Delta T = T_{operating} - T_{stress-free} \approx T_{operating} - T_{cure}$   
= Temperature differential.

SGNLD = +1 for tensile shear load, and  
= -1 for compressive shear load.

ANSTEP = Number of steps in the joint. This serves to control the number of adherend property cards read in.

CARDS 3, 3A, 3B, ..etc., 3(N = ANSTEP + 1)

FORMAT (7F10.3)

THICKO(N) = Sum of thicknesses of outer adherends for nth step.

THICKI(N) = Thickness of nth step of inner adherend.

STEPL(N) = Length of nth step.

ETOTR(N) = Net extensional stiffness of outer adherends at nth step.

ETINR(N) = Extensional stiffness of inner adherend at nth step.

STROTR(N) = Net strength of outer adherends at nth step.

STRINR(N) = Strength of inner adherend at nth step.

The output is in tabular form with one row devoted to each step or step portion. Those entries not defined in the input description above are: TAU the adhesive shear stress, GAMMA the adhesive shear strain, DELTA0 the displacement of the outer adherends, DELTA1 the displacement of the inner adherend, with TOUTER and TINNER being the loads ( $\sigma_t$ ) in the outer and inner adherends, respectively.

The more accurate solution is obtained by starting the iterative solution from the more critically loaded end. Therefore, in those cases in which the a priori identification of the more critical end is not possible, the program outputs solutions from each end, and the second one is to be preferred. Such cases have been run and the computational procedure in double precision has been shown to be sufficiently accurate from either end. The need for this higher precision on IBM computers arises from the precision loss throughout the nested do loops in the iteration sequence. The greater number of significant digits employed by CDC machines has been found to obviate the need for this and the program can be modified to single-precision operation on CDC machines in a straightforward manner.

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CDECK A4FF
C STEPPED-LAP ADHESIVE-BONDED-JOINTS
C PERFECTLY-PLASTIC SOLUTIONS
C JOINT ANALYSIS PROGRAM
C SOLUTION EXAMINES ADHESIVE SHEAR STRESS AND ADHEREND NORMAL (AXIAL)
C STRESS BUT OMITS CONSIDERATION OF ADHESIVE PEEL STRESS ON THE
C GROUND THAT OUTER END STEP IS USUALLY SUFFICIENTLY THIN FOR
C PEEL STRESS PROBLEMS NOT TO ARISE
C NOTE THAT CONVERGENCE PROBLEM IS ACUTE FOR STEPPED-LAP JOINTS, EVEN
C WITH DOUBLE-PRECISION. STEPS TAKEN HERE TO CONSTRAIN TENDENCY
C TO DIVERGE (BY FREEZING SOLUTION ONE STEP AT A TIME) HAVE BEEN
C ADOPTED AFTER TRYING BOTH MORE AND LESS STRINGENT TECHNIQUES
C NOTE ALSO THAT CONVERGENCE DIFFICULTIES ARE PROBLEM DEPENDENT, BEING
C MORE SEVERE FOR BRITTLE (HIGH MODULUS) ADHESIVES. LOW MODULUS
C ADHESIVES PROVEN AMENABLE TO A CONVERGENT SOLUTION IN ONLY A
C SINGLE PASS STRAIGHT THROUGH THE JOINT FROM END TO END, IN
C SINGLE-PRECISION, WITH ONLY A SMALL LOSS OF ACCURACY IN LATER
C STEPS.
C THE UNDERRYING DIFFICULTY IS ONE OF NUMERICAL ACCURACY LOSS IN THE
C PRESENCE OF EXTREMELY HIGH ADHESIVE SHEAR STRESS GRADIENTS AT
C BOTH ENDS OF EACH OF THE OUTER STEPS.
C NOTE THAT PROGRAM CANNOT HANDLE PROPERLY A JOINT WITH SUCH HIGH
C RESIDUAL THERMAL STRESSES THAT IT BREAKS APART PRIOR TO
C APPLICATION OF MECHANICAL LOADS. ANSWER FROM ONE END OF JOINT
C WILL BE ZERO, BUT FROM OTHER END WILL BE LARGE AND POSITIVE.
C ACTUALLY, THE LATTER ANSWER IS FOR A LOAD OF REVERSED SIGN, WITH
C THE THERMAL STRESSES HELPING RATHER THAN HINDERING. SUCH A
C SITUATION CAN BE SPOTTED IF THE SHEAR STRESS IN THE FORMER
C SOLUTION IS NEGATIVE AT THE START OF THE JOINT
C DIMENSION TOUTFP(50), TINNER(50), GAMMA(50), TAU(50), DELTA0(50),
C 1 DELTA1(50), STEP1(50), THICK0(50), THICK1(50), ETOTR(50),
C 2 FTJNR(50), STROTP(50), STRINR(50), STEP(50), THCKND(50),
C 3 THCKNI(50), ETOUTR(50), FTINNR(50), STRGTP(50), STRGNR(50)
C DOUBLE PRECISION TOUTER, TINNER, GAMMA, TAU, DELTA0, DELTA1,
C 1 TMAX, TMIN, TLOAD, A, B, C, D, E, F, ALAMDA, DELT, DELDT,
C 2 C1, C2, C3, C4, C5, V, STEP, THCKND, THCKNI, ETOUTR, FTINNR,
C 3 STRGTR, STRGNR, TCHECK, STEPL, THICK0, THICK1, ETOTR, ETINR,
C 4 STROTP, STRINR, TAUJUPR, TAUWR
C READ (5,10) M
C 10 FORMAT (I2)
C M = NO. NUMBER OF JOINT CONFIGURATIONS TO BE SOLVED
C READ IN MATERIAL PROPERTIES
C READ IN 390 MCOUNT = 1, M
C NRVRS = 0
C JFLAG = 1
C JFLAG IDENTIFIES END OF JOINT FROM WHICH ANALYSIS COMMENCES
C READ (5,20) TAUMAX, G, GAMMAX, GAMMAE, ETA, ALPHAO, ALPHAI,
C 1 DELTAMP, SGND, ANSTEP
C 20 FORMAT (8F10.3)
C NSTEPS = ANSTEP
C MSTEPS = NSTEPS + 1
C READ IN JOINT GEOMETRY
C READ (5,30) ((THICK0(N), THICK1(N), STEPL(N), ETOTR(N), ETINR(N),
C 1 STROTP(N), STRINR(N)), N = 1, MSTEPS)
C 30 FORMAT (7F10.3)
C CHECK ON CONSISTENCY OF ADHESIVE DATA
C VCHECK = G * GAMMAE
C P = TAUMAX / VCHECK
C IF ((1.001 .LT. P) .OR. (0.999 .GT. P)) GO TO 390
C IF (GAMMAX .GE. GAMMAE) GO TO 40
C IF NOT, REDUCE PEAK SHEAR STRESS TO LESS THAN MAXIMUM ELASTIC VALUE
C TAUMAX = G * GAMMAX
C SET UP RECURRING CONSTANTS
C 40 C1 = G / ETA
C C2 = 2. * C1
C FACTOR 2 ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND.
C 1 IF BONDED ON ONE SIDE ONLY, REDUCE TO 1.
C 50 C3 = ALPHAO * DELTAMP
C C4 = ALPHAI * DELTAMP
C C5 = C4 - C3
C PRINT OUT INPUT DATA
C WRITE (6,60) ALPHAO, ALPHAI, DELTAMP
C 60 FORMAT (1H1, 5(1H0/), 11H INPUT DATA//, 10H ALPHAO = , F10.7,
C 1 13H (PER DEG. F), 3X, 9HALPHAI = , F10.7, 13H (PER DEG. F),
C 2 3X, 9DELTEMP = , F6.1, 9H (DEG. F)//,
C 3 1H, 5X, 1HN, 2X, 5HSTEP1, 1X, 6HTHICK0, 1X, 6HTHICK1, 4X,
C 4 6HSTROTP, 5X, 6HSTRINR, 5X, SHETOTP, 5X, SHETINR//)
C DO 80 N = 1, MSTEPS
C WRITE (6,70) N, STEPL(N), THICK0(N), THICK1(N), STROTP(N),
C 1 STRINR(N), ETOTR(N), FTINR(N)
C 70 FORMAT (1H , 4X, I2, 3(1X, F6.4), 2(1X, F10.1), 2(1X, F10.1))
C 80 CONTINUE
C ESTIMATE MAXIMUM POSSIBLE BOND CAPACITY FOR FULLY-PLASTIC ADHESIVE
C PBOND = TAUMAX * QLAP * 2
C NOTE FACTOR 2 INCLUDED FOR DOUBLE-SIDED JOINT
C REDUCE TO 1. IF JOINT HAS ONLY ONE SIDE BONDED

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C PERFECTLY-PLASTIC BOND CAPACITY WILL BE CLOSER TO ASYMPTOTE OF SCARE A4FF0990
C 1. JOINT SOLUTION AND IS SIGNIFICANTLY LOWER THAN PLASTIC ESTIMATE A4FF0990
C ACTUAL LOAD CAPACITY MAY BE SIGNIFICANTLY LESS IF THERMAL MISMATCH A4FF0990
C 1. BETWEEN ADHERENDS IS SEVERE A4FF0990
C REDUCTION IN LOAD TO ACCOUNT FOR LIMITED ADHEREND STRENGTH IS A4FF0990
C 1. ACCOMPLISHED LATER IN PROGRAM A4FF0990
C PROVIDE OUTER LOOP TO ADJUST ADHESIVE PEAK SHEAR STRESS AT START OF A4FF0990
C 1. JOINT FOR CASES IN WHICH EITHER ADHESIVE IS MORE CRITICAL AT A4FF0990
C 2. OTHER END OF JOINT OR ADHERENDS ARE MORE CRITICAL THAN ADHESIVE. A4FF0990
C TAUJUPP = 2. * TAUUMAX A4FF0990
C TAUJLWP = 0. A4FF0990
C NOTE THAT PROGRAM IS PREVENTED FROM HANDLING PROBLEM IN WHICH SHEAR A4FF1000
C 1. STRESS IN ADHESIVE REVERSES SIGN, WHEN COMPUTATIONS START FROM A4FF1000
C 2. THE LESS CRITICAL END. SOLUTION IS OBTAINABLE FROM OTHER END. A4FF1020
C NOTE ALSO THAT, IF THE MAXIMUM SHEAR STRESS AND APPLIED LOADS HAVE A4FF1030
C 1. OPPOSITE SIGNS, JOINT MUST BREAK APART UNDER PESTDUAL THERMAL A4FF1040
C 2. STRESS ALONE WITHOUT ANY EXTERNALLY APPLIED LOAD, SO NO CASES OF A4FF1050
C 3. REAL CONCERN ARE EXCLUDED BY THE RESTRICTION ABOVE A4FF1060
DO 290 T = 1, 50 A4FF1070
TAU(1) = (TAUJUPP + TAUJLWP) / 2. A4FF1080
IF (TAU(1) .GT. TAUUMAX) TAU(1) = TAUUMAX A4FF1090
TF (I .EQ. 1) TAU(1) = TAUUMAX A4FF1100
C SET INITIAL CONDITIONS A4FF1110
GAMMA(1) = TAU(1) / G A4FF1120
TOUTER(1) = 5. * PROND A4FF1130
TINNER(1) = 0. A4FF1140
DELTAT(1) = 0. A4FF1150
DELTAT(1) = SGNLD * GAMMA(1) * FTA A4FF1160
TMAX = 10. * PROND A4FF1170
TMIN = 0. A4FF1180
TLOAD = 5. * PROND A4FF1190
C OPERATE ON THE LOAD LEVEL IN INTERMEDIATE LOOP A4FF1200
C LEAVE ADJUSTMENT OF TAUUMAX FOR OUTER LOOP A4FF1210
TCHECK = 0. A4FF1220
DO 190 IFLAG = 1, NSTEPS A4FF1230
SCHECK = 100000000000000. A4FF1240
DO 150 NCOUNT = 1, 100 A4FF1250
C CONVERGENCE NEARLY ALWAYS OCCURRED BETWEEN 20 AND 30 CYCLES IN TEST A4FF1260
C 1. CASES, BUT THERE WERE SOME EXCEPTIONS A4FF1270
C INTERMEDIATE LOOP ADJUSTS LOAD LEVEL A4FF1280
C TLOAD = TOUTER(IFLAG) A4FF1290
C CHECK ON CONVERGENCE OF TOUTER(IFLAG) A4FF1300
P = TOUTER(IFLAG) / SCHECK A4FF1310
IF ((1.000000001 .GT. P) .AND. (0.999999999 .LT. P)) GO TO 160 A4FF1320
DO 100 N = IFLAG, NSTEPS A4FF1330
C INNER LOOP COMPUTES PLASTIC JOINT STRENGTH A4FF1340
A = TAU(N) A4FF1350
ALAMDA = DSORT(C2 * (1. / ETINR(N) + 1. / ETOTR(N))) A4FF1360
B = (TINNER(N) / ETINR(N) - TOUTER(N) / ETOTR(N) + C5 * SGNLD) A4FF1370
1. * C1 / ALAMDA A4FF1380
C NOTE THAT SGNLD SIGNIFIES WHETHER SHEAR LOAD IS TENSILE OR COMPRESSIVE A4FF1390
C E = STEPL(N) A4FF1400
D = ALAMDA * C A4FF1410
F = DSINH(D) A4FF1420
F = DCOSH(D) A4FF1430
TAU(N+1) = A * E + B * F A4FF1440
DELT = (2. / ALAMDA) * (A * E + B * (F - 1.)) A4FF1450
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND. A4FF1460
C 1. IF BONDED ON ONE SIDE ONLY, REDUCE TO 1. A4FF1470
TOUTER(N+1) = TOUTER(N) - DELT A4FF1480
TINNER(N+1) = TINNER(N) + DELT A4FF1490
IF (N .EQ. NSTEPS) GO TO 90 A4FF1500
IF (TINNER(N+1) .LT. (-1. * TOUTER(1))) GO TO 130 A4FF1510
IF (TOUTER(N+1) .LT. (-1. * TOUTER(1))) GO TO 140 A4FF1520
C NOTE THAT THESE CONVERGENCE CHECKS ARE CRITICAL A4FF1530
C IF THE FACTOR -1 IS EITHER TOO LARGE OR TOO SMALL, CONVERGENCE FAILS A4FF1540
90 DELDLT = (2. / (ALAMDA**2)) * (A * (F - 1.) + B * (E - D)) A4FF1550
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND. A4FF1560
C 1. IF BONDED ON ONE SIDE ONLY, REDUCE TO 1. A4FF1570
DELTAN(N+1) = DELTAO(N) + C3 * C + SGNLD * (TOUTER(N) * C - A4FF1580
1. DELDLT) / ETOTR(N) A4FF1590
DELTAT(N+1) = DELTAI(N) + C4 * C + SGNLD * (TINNER(N) * C + A4FF1600
1. DELDLT) / ETINR(N) A4FF1610
GAMMA(N+1) = TAU(N+1) / G A4FF1620
100 CONTINUE A4FF1630
C CHECK WHETHER OR NOT PRECISELY 100 PERCENT OF LOAD HAS TRANSFERRED A4FF1640
R1 = TOUTER(1) / TINNER(MSTEPS) A4FF1650
C CHECK ALSO WHETHER OR NOT CONVERGENCE HAS BEEN OBTAINED A4FF1660
R2 = TCHECK / TINNER(MSTEPS) A4FF1670
IF ((1.000001 .GT. R1) .AND. (0.999999 .LT. R1) .AND. A4FF1680
1. (1.000001 .GT. R2) .AND. (0.999999 .LT. R2)) GO TO 200 A4FF1690
IF (TOUTER(1) .LT. TINNER(MSTEPS)) GO TO 110 A4FF1700
C IF SO, LOAD ESTIMATE IS TOO LOW A4FF1710
C IF NOT, LOAD ESTIMATE IS TOO HIGH A4FF1720
GO TO 120 A4FF1730
C R1 IS UNSUITABLE FOR A CONVERGENCE CHECK BECAUSE NEGATIVE VALUES OF R1 A4FF1740
C 1. REPRESENT TOO HIGH A LOAD ESTIMATE, JUST LIKE THOSE VALUES IN A4FF1750
C 2. EXCESS OF UNITY A4FF1760

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110 TMIN = TOUTER(IFLAG)
TOUTER(IFLAG) = (TMIN + TMAX) / 2.
TCHECK = TINNER(MSTEPS)
GO TO 150
A4FF1770
A4FF1780
A4FF1790
A4FF1800
A4FF1810
A4FF1820
A4FF1830
A4FF1840
A4FF1850
A4FF1860
A4FF1870
A4FF1880
A4FF1890
A4FF1900
A4FF1910
A4FF1920
A4FF1930
A4FF1940
A4FF1950
A4FF1960
A4FF1970
A4FF1980
A4FF1990
A4FF2000
A4FF2010
A4FF2020
A4FF2030
A4FF2040
A4FF2050
A4FF2060
A4FF2070
A4FF2080
A4FF2090
A4FF2100
A4FF2110
A4FF2120
A4FF2130
A4FF2140
A4FF2150
A4FF2160
A4FF2170
A4FF2180
A4FF2190
A4FF2200
A4FF2210
A4FF2220
A4FF2230
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

C NOTE THAT LABELS 26 AND 7 GOVERN FINE ADJUSTMENTS TO THE JOINT LOADS,
C 1 WHILE LABELS 27 AND 28 REPRESENT COARSE ADJUSTMENTS
130 TMAX = TOUTER(IFLAG)
SCHECK = TOUTER(IFLAG)
TOUTER(IFLAG) = (TMIN + TMAX) / 2.
GO TO 150
A4FF1850
A4FF1860
A4FF1870
A4FF1880
A4FF1890
A4FF1900
A4FF1910
A4FF1920
A4FF1930
A4FF1940
A4FF1950
A4FF1960
A4FF1970
A4FF1980
A4FF1990
A4FF2000
A4FF2010
A4FF2020
A4FF2030
A4FF2040
A4FF2050
A4FF2060
A4FF2070
A4FF2080
A4FF2090
A4FF2100
A4FF2110
A4FF2120
A4FF2130
A4FF2140
A4FF2150
A4FF2160
A4FF2170
A4FF2180
A4FF2190
A4FF2200
A4FF2210
A4FF2220
A4FF2230
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

C IF ADHEREND, RATHER THAN ADHESIVE, LIMITS JOINT STRENGTH, NEED TO
C 1 BOOST PLOAD IN PROPORTION TO TAUMAX, EVEN IF IT MEANS EXCEEDING
C 2 ADHEREND STRENGTHS IN INTERMEDIATE COMPUTATIONS. CORRECTIONS
C 3 ARE APPLIED LATER
IF (TMIN .GE. TMAX) TMAX = 5. * TMAX
TOUTER(IFLAG) = (TMIN + TMAX) / 2.
150 CONTINUE
IF (N .EQ. NSTEPS) GO TO 190
A4FF1950
A4FF1960
A4FF1970
A4FF1980
A4FF1990
A4FF2000
A4FF2010
A4FF2020
A4FF2030
A4FF2040
A4FF2050
A4FF2060
A4FF2070
A4FF2080
A4FF2090
A4FF2100
A4FF2110
A4FF2120
A4FF2130
A4FF2140
A4FF2150
A4FF2160
A4FF2170
A4FF2180
A4FF2190
A4FF2200
A4FF2210
A4FF2220
A4FF2230
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

C CONVERGENCE WILL NOT PROCEED TO FAR END OF JOINT IN SINGLE PASS
C 1 BECAUSE OF NUMERICAL ACCURACY PROBLEMS. REMEDY IS TO FREEZE
C 2 EARLIER VALUES, WHICH HAVE CONVERGED AND SLIGHTLY PERTURB
C 3 INTERMEDIATE VALUES, AND TO CHECK FOR CONVERGENCE AT THE FAR END
TMAX = TOUTER(1)
TMIN = -1. * TMAX
GO TO 190
A4FF2050
A4FF2060
A4FF2070
A4FF2080
A4FF2090
A4FF2100
A4FF2110
A4FF2120
A4FF2130
A4FF2140
A4FF2150
A4FF2160
A4FF2170
A4FF2180
A4FF2190
A4FF2200
A4FF2210
A4FF2220
A4FF2230
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

C 150 CONTINUE
IF (N .EQ. NSTEPS) GO TO 190
A4FF1950
A4FF1960
A4FF1970
A4FF1980
A4FF1990
A4FF2000
A4FF2010
A4FF2020
A4FF2030
A4FF2040
A4FF2050
A4FF2060
A4FF2070
A4FF2080
A4FF2090
A4FF2100
A4FF2110
A4FF2120
A4FF2130
A4FF2140
A4FF2150
A4FF2160
A4FF2170
A4FF2180
A4FF2190
A4FF2200
A4FF2210
A4FF2220
A4FF2230
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

C 170 TMAX = 1.1 * TOUTER(ICOUNT)
TMIN = 0.9 * TOUTER(ICOUNT)
GO TO 190
A4FF2140
A4FF2150
A4FF2160
A4FF2170
A4FF2180
A4FF2190
A4FF2200
A4FF2210
A4FF2220
A4FF2230
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

C 180 TMIN = 0.9 * TOUTER(ICOUNT)
TMIN = 1.1 * TOUTER(ICOUNT)
190 CONTINUE
NRVPS = 1
A4FF2190
A4FF2200
A4FF2210
A4FF2220
A4FF2230
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

C 200 IF ( (CS .GT. 0.000001) .OR. (CS .LT. -0.000001) ) GO TO 240
C IF NOT, FIRST SOLUTION MAY BE SCALED IN THE ABSENCE OF ANY THERMAL
C 1 MISMATCH BETWEEN ADHERENDS
C IF SO, SOLUTION MUST BE REFINED BY ITERATION, SINCE THERMAL STRESS
C 1 TERMS DO NOT SCALE LINEARLY, EVEN FOR ELASTIC ADHESIVE AND
C 2 ADHERENDS
APPLY SCALE FACTOR TO SOLUTION FOR ONLY ADHEREND STIFFNESS IMBALANCE
ASCERTAIN WHETHER INTERNAL LOADS ARE CRITICAL FOR ELASTIC ADHESIVE OR
1 WHETHER OTHER END OF JOINT IS MORE CRITICAL FOR ADHESIVE
PROGRAM ASSUMES ADHEREND ALLOWABLES HAVE SAME MAGNITUDE IN TENSION AS
1 IN COMPRESSION. DISTINCTION IS USUALLY UNIMPORTANT SINCE, IN
2 PRACTICAL JOINTS, RESIDUAL THERMAL STRESSES ARE UNLIKELY TO
3 BREAK ADHEREND(S) RATHER THAN ADHESIVE
RSCALE = TOUTER(1) / STROTP(1)
IF (RSCALE .LT. 0.) RSCALE = -1. * RSCALE
RTAUMX = TAU(1) / TAUMAX
IF (RTAUMX .LT. 0.) RTAUMX = -1. * RTAUMX
DO 220 N = 2, MSTEPS
RINR = TINNER(N) / STRINR(N)
IF (RINR .LT. 0.) RINP = -1. * RINR
IF (RINR .GT. 0.) RSCALE = RTNP
IF (N .EQ. MSTEPS) GO TO 210
ROTTR = TOUTER(N) / STROTP(N)
IF (ROTTR .LT. 0.) ROTTR = -1. * ROTTR
IF (ROTTR .GT. RSCALE) RSCALE = ROTTR
210 RTAU = TAU(N) / TAUMAX
IF (RTAU .LT. 0.) RTAU = -1. * RTAU
IF (RTAU .GT. RTAUMX) RTAUMX = RTAU
220 CONTINUE
RFCTR = RSCALE
RFCTR IS PROPORTIONALITY CONSTANT GOVERNING ELASTIC SOLUTION
C IF RTAUMAX .GT. RSCALE, ADHESIVE PLASTICITY CAN INCREASE STRENGTH
C USUALLY ADHESIVE IS CRITICAL AT ONE END OF JOINT OR OTHER, SO RTAUMX
C 1 .GT. 1. MAY WELL JUST SIGNIFY THAT FAR END OF JOINT IS CRITICAL
C NOTE THAT PROGRAM ASSUMES THAT ANY INTERNAL ADHEREND STRESSES OF
C 1 REVERSED SIGN WITH RESPECT TO STRESS OUTSIDE THE JOINT ARE NOT
C 2 CRITICAL. IF THEY ARE, IT MEANS THAT THE JOINT WILL FAIL DUE
C 3 TO RESIDUAL THERMAL STRESSES ALONE WITHOUT ANY MECHANICAL LOADS
IF (RSCALE .LT. RTAUMX) RFCTR = RTAUMX
DO 230 N = 1, MSTEPS
A4FF2240
A4FF2250
A4FF2260
A4FF2270
A4FF2280
A4FF2290
A4FF2300
A4FF2310
A4FF2320
A4FF2330
A4FF2340
A4FF2350
A4FF2360
A4FF2370
A4FF2380
A4FF2390
A4FF2400
A4FF2410
A4FF2420
A4FF2430
A4FF2440
A4FF2450
A4FF2460
A4FF2470
A4FF2480
A4FF2490
A4FF2500
A4FF2510
A4FF2520
A4FF2530
A4FF2540
A4FF2550
A4FF2560
A4FF2570
A4FF2580
A4FF2590
A4FF2600
A4FF2610
A4FF2620
A4FF2630
A4FF2640

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TOUTER(N) = TOUTER(N) / RFACTR          A4FF2650
TINNER(N) = TINNER(N) / RFACTR          A4FF2660
TAU(N) = TAU(N) / RFACTR                A4FF2670
GAMMA(N) = GAMMA(N) / RFACTR            A4FF2680
DELTAD(N) = DELTAD(N) / RFACTR          A4FF2690
230 DELTAT(N) = DELTAT(N) / RFACTR      A4FF2700
GO TO 310                                A4FF2710
C USE ITERATIVE SOLUTION WHEN ADHEREND THERMAL MISMATCH IS PRESENT    A4FF2720
C ASCERTAIN WHETHER INTERNAL LOADS ARE CRITICAL FOR ELASTIC ADHESIVE OR    A4FF2730
C 1 WHETHER OTHER END OF JOINT IS MORE CRITICAL FOR ADHESIVE.           A4FF2740
240 PSCALE = TOUTER(1) / STROTR(1)        A4FF2750
IF (PSCALE .LT. 0.) RSCALE = -1. * RSCALE   A4FF2760
RTAUMX = TAU(1) / TAUMAX                 A4FF2770
IF (RTAUMX .LT. 0.) RTAUMX = -1. * RTAUMX   A4FF2780
DO 260 N = 2, MSTEPS                      A4FF2790
RNIN = TINNER(N) / STRINP(N-1)             A4FF2800
C NEED TO COMPARE LOAD WITH STRENGTH ON THIN SIDE OF STEP. HENCE (N-1) A4FF2810
IF (RNIN .LT. 0.) RINP = -1. * RNIN       A4FF2820
IF (RINP .GT. PSCALE) PSCALE = RNIN       A4FF2830
IF (N .EQ. MSTEPS) GO TO 250              A4FF2840
ROTR = TOUTER(N) / STROTR(N)               A4FF2850
IF (ROTR .LT. 0.) ROTR = -1. * ROTR       A4FF2860
IF (ROTR .GT. PSCALE) PSCALE = ROTR       A4FF2870
250 RTAU = TAU(N) / TAUMAX                A4FF2880
IF (RTAU .LT. 0.) RTAU = -1. * RTAU       A4FF2890
IF (RTAU .GT. RTAUMX) RTAUMX = RTAU       A4FF2900
260 CONTINUE                               A4FF2910
C CHECK ON CONVERGENCE                   A4FF2920
P = RTAUMX                               A4FF2930
IF (RTAUMX .LT. PSCALE) P = PSCALE       A4FF2940
IF ((1.00001 .GT. P) .AND. (0.99999 .LT. P)) GO TO 310   A4FF2950
V = 2. * TAUMAX                           A4FF2960
P = (V + TAUUPP) / (V + TAULWR)         A4FF2970
IF ((1.00001 .GT. P) .AND. (0.99999 .LT. P)) GO TO 310   A4FF2980
C IF EITHER RTAUMX OR PSCALE .GT. UNITY, TAU(1) MUST BE DECREASED   A4FF2990
IF ((RTAUMX .GT. 1.00001) .OR. (PSCALE .GT. 1.00001)) GO TO 270   A4FF3000
C IF BOTH RTAUMX AND PSCALE ARE .LT. UNITY, TAU(1) MUST BE INCREASED   A4FF3010
IF ((RTAUMX .LT. 0.99999) .AND. (PSCALE .LT. 0.99999)) GO TO 280   A4FF3020
C IF NONE OF THE THREE CHECKS ABOVE IS MET, SOLUTION HAS FAILED TO   A4FF3030
C 1 CONVERGE WITHIN SPECIFIED NUMBER OF ITERATIONS. PRINT OUT ANSWER   A4FF3040
C GO TO 14                                A4FF3050
270 IF (TAU(1) .GT. 0.) TAUUPP = TAU(1)   A4FF3060
IF (TAU(1) .LT. 0.) TAULWR = TAU(1)       A4FF3070
GO TO 290                                A4FF3080
280 IF (TAU(1) .GT. 0.) TAULWR = TAU(1)   A4FF3090
IF (TAU(1) .LT. 0.) TAUUPP = TAU(1)       A4FF3100
290 CONTINUE                               A4FF3110
C IF PROGRAM GOES BEYOND PRECEDING CONTINUE STATEMENT, SOLUTION HAS NOT A4FF3120
C 1 CONVERGED                            A4FF3130
WRITE (6,300)                             A4FF3140
300 FORMAT (1H1, 18HDIVERGENT SOLUTION)   A4FF3150
C PRINT OUT RESULTS OF ELASTIC COMPUTATIONS A4FF3160
310 WRITE (6,320) TOUTER(1), TAUMAX, SGNLD, DELTMP   A4FF3170
320 FORMAT (1H1/, 5(1H0/),
1 39H ELASTIC JOINT STRENGTH, PLOAD (LBS) = , F10.1/, A4FF3180
2 49H ALLOWABLE ADHESIVE SHEAR STRESS, TAUMAX (PSI) = , F8.1/, A4FF3190
3 1H, 8HSGNLD = , E4.1, 54H SGNLD = +1 FOR TENSILE SHEAR AND -1 A4FF3200
4 FOR COMPRESSIVE/, 36H TEMPERATURE DIFFERENTIAL (DEG F) = , F6.1// A4FF3210
5, 1H, 5X, 1HN, 2X, 5HSTEPL, 1X, 6HTHICKD, 1X, 6HTHICKT, 3X, A4FF3220
6, 3HTAU, 4X, 5HGAMMA, 1X, 6HDELTAD, 1X, 6HDELTAT, 5X, 6HTOUTER, A4FF3230
7 5X, 6HSTROTR, 5X, 6HTINNER, 5X, 6HSTRINP//) A4FF3240
DO 340 N = 1, MSTEPS                      A4FF3250
WRITE (6,330) N, STEPL(N), THICKD(N), THICKT(N), TAU(N), GAMMA(N), A4FF3260
1 DELTAD(N), DELTAT(N), TOUTER(N), STROTR(N), TINNER(N), STRINP(N) A4FF3270
2,
330 FORMAT (1H , 4X, I2, 1X, F6.4, 1X, F6.4, 1X, F6.4, 1X, F7.1, 1X, A4FF3280
1 F6.3, 1X, F6.4, 1X, F6.4, 1X, F10.1, 1X, F10.1, 1X, F10.1, 1X, A4FF3290
2 F10.1) A4FF3300
340 CONTINUE                               A4FF3310
350 CONTINUE                               A4FF3320
C RECOMPUTE SOLUTION FROM OTHER END OF JOINT, IF APPROPRIATE A4FF3330
C NOTE THAT, IF COMPUTER PRINTS OUT TWO SOLUTIONS TO A GIVEN PROBLEM BY A4FF3340
C 1 REVERSING ENDS AND RE-ANALYZING, IT IS BECAUSE THE FIRST FAILED A4FF3350
C 2 TO CONVERGE, EVEN IF THE ANSWERS PRINTED SEEM TO SUGGEST A4FF3360
C 3 OTHERWISE. THE SECOND SOLUTION IS TO BE PREFERRED, PARTICULARLY A4FF3370
C 4 IF IT STARTS AT THAT END OF THE JOINT AT WHICH THE ADHESIVE A4FF3380
C 5 SHEAR STRESS IS AT ITS HIGHEST. A4FF3390
C IDENTIFY CRITICAL END OF JOINT          A4FF3400
C AVOID REVERSING ENDS BACK AGAIN        A4FF3410
IF (JFLAG .EQ. 2) GO TO 390              A4FF3420
IF (NRVRS .EQ. 1) GO TO 360              A4FF3430
C IF SO, SOLUTION HAS FAILED TO CONVERGE, SO TRY AGAIN FROM OTHER END A4FF3440
C ACCURACY AT FAR END OF JOINT MAY BE POOR IF FAR END IS CRITICAL A4FF3450
IF ((TAU(MSTEPS) .LE. TAU(1)) .AND. (TAU(MSTEPS) .GE. A4FF3460
1 (-1. * TAU(1)))) GO TO 390          A4FF3470

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C IF, AT FAR END OF JOINT, TAU(MSTEPS) .GT. TAU(1) AT NEAR END,
C 1 FAILURE TO CONVERGE MAY BE SIMPLY THE RESULT OF THE FAR END
C 2 OF THE JOINT BEING MORE CRITICAL THAN THE STARTING (NEAR) END
C REVERSE DATA AND REANALYZE
360 DO 370 N = 1, MSTEPS
      STEP(N) = STEP(N)
      THCKNO(N) = THICKO(N)
      THCKNI(N) = THICKT(N)
      ETOTTR(N) = ETOTR(N)
      ETINNR(N) = ETINR(N)
      STRGTR(N) = STROTR(N)
370  STRGNR(N) = STPINR(N)
      DO 380 N = 1, MSTEPS
          STEP(N) = STEP(MSTEPS - N)
          THICKO(N) = THCKNI(MSTEPS - N)
          THICKT(N) = THCKNO(MSTEPS - N)
          ETOTR(N) = ETINNR(MSTEPS - N)
          ETINR(N) = ETOTR(MSTEPS - N)
          STROTR(N) = STRGNR(MSTEPS - N)
380  STRINR(N) = STRGTR(MSTEPS - N)
      STEPL(MSTEPS) = STEP(MSTEPS)
      THICKO(MSTEPS) = 0.
      THICKT(MSTEPS) = THCKNO(1)
      ETOTR(MSTEPS) = 0.
      ETINR(MSTEPS) = ETOTR(1)
      STROTR(MSTEPS) = 0.
      STRINR(MSTEPS) = STRGTR(1)
      V = ALPHA0
      ALPHA0 = ALPHAI
      ALPHAI = V
      JFLAG = 2
      NRVRS = 0
      GO TO 50
390  CONTINUE
      STOP
      END

```

INPUT DATA  
 $\text{ALPHAO} = 0.0$  (PER DEG. F)  $\text{ALPHAT} = 0.0000050$  (PER DEG. F)  $\text{DELTMP} = 0.0$  (DEG. F)

N	STEPL	THICKO	THICKI	STROTR	STRINR	ETOTR	ETINR
1	0.0750	0.2640	0.0300	18216.0	3900.0	2112000.0	480000.0
2	0.0750	0.2640	0.0300	18216.0	3900.0	2112000.0	480000.0
3	0.0750	0.2640	0.0300	18216.0	3900.0	2112000.0	480000.0
4	0.0750	0.2640	0.0300	18216.0	3900.0	2112000.0	480000.0
5	0.0750	0.2640	0.0300	18216.0	3900.0	2112000.0	480000.0
6	0.0750	0.2200	0.0740	18180.0	9600.0	1760000.0	1184000.0
7	0.0750	0.2200	0.0740	18180.0	9600.0	1760000.0	1184000.0
8	0.0750	0.2200	0.0740	18180.0	9600.0	1760000.0	1184000.0
9	0.0750	0.2200	0.0740	18180.0	9600.0	1760000.0	1184000.0
10	0.0750	0.2200	0.0740	18180.0	9600.0	1760000.0	1184000.0
11	0.1250	0.1760	0.1180	12144.0	15300.0	1403000.0	1883000.0
12	0.1250	0.1760	0.1180	12144.0	15300.0	1403000.0	1883000.0
13	0.1250	0.1760	0.1180	12144.0	15300.0	1408000.0	1883000.0
14	0.1250	0.1760	0.1180	12144.0	15300.0	1408000.0	1883000.0
15	0.1250	0.1320	0.1620	9108.0	21060.0	1056000.0	2592000.0
16	0.1250	0.1320	0.1620	9108.0	21060.0	1056000.0	2592000.0
17	0.1250	0.1320	0.1620	9108.0	21060.0	1056000.0	2592000.0
18	0.1250	0.1320	0.1620	9108.0	21060.0	1056000.0	2592000.0
19	0.1250	0.1320	0.1620	9108.0	21060.0	1056000.0	2592000.0
20	0.1250	0.1320	0.1620	9108.0	21060.0	1056000.0	2592000.0
21	0.1250	0.0480	0.2660	6772.0	26780.0	704000.0	3296000.0
22	0.1250	0.0480	0.2660	6772.0	26780.0	704000.0	3296000.0
23	0.1250	0.0480	0.2660	6772.0	26780.0	704000.0	3296000.0
24	0.1250	0.0480	0.2660	6772.0	26780.0	704000.0	3296000.0
25	0.1250	0.0480	0.2660	6772.0	26780.0	704000.0	3296000.0
26	0.1250	0.0440	0.2500	3036.0	32500.0	352000.0	4000000.0
27	0.1250	0.0440	0.2500	3036.0	32500.0	352000.0	4000000.0
28	0.1250	0.0440	0.2500	3036.0	32500.0	352000.0	4000000.0
29	0.1250	0.0440	0.2500	3036.0	32500.0	352000.0	4000000.0
30	0.1250	0.0440	0.2500	3036.0	32500.0	352000.0	4000000.0
31	*****	0.0	0.2500	0.0	32500.0	0.0	4000000.0

ELASTIC JOINT STRENGTH, PLOAD (LBS) = 7828.0  
 ALLOWABLE ADHESIVE SHEAR STRESS, TAUMAX (PSI) = 6000.0  
 $N = 1.0$   $N = +1$  FOR TENSILE SHEAR AND -1 FOR COMPRESSIVE  
 TEMPERATURE DIFFERENTIAL (DEG F) = C.0

N	STEPL	THICKO	THICKI	TAU	GAMMA	DELTA0	DELTA1	TOUTER	STROTR	TINNER	STRINR
1	0.0750	0.2640	0.0300	6000.0	2.100	0.0	0.0025	7828.0	18216.0	0.0	3900.0
2	0.0750	0.2640	0.0300	3534.6	0.1339	0.0003	0.0007	7132.9	18216.0	695.2	3900.0
3	0.0750	0.2640	0.0300	1940.7	0.032	0.0009	0.0012	6705.6	18216.0	1122.4	3900.0
4	0.0750	0.2640	0.0300	12245.9	0.0327	0.0014	0.0015	6394.6	18216.0	1433.5	3900.0
5	0.0750	0.2640	0.0300	2245.9	0.0556	0.0014	0.0016	6039.3	18216.0	1738.7	3900.0
6	0.0750	0.2200	0.0740	2259.1	0.0320	0.0016	0.0018	5581.3	15180.0	2146.7	9620.0
7	0.0750	0.2200	0.0740	1826.0	0.0320	0.0020	0.0022	5259.0	15180.0	2568.1	9620.0
8	0.0750	0.2200	0.0740	1647.6	0.0320	0.0022	0.0024	4952.9	15180.0	2877.1	9620.0
9	0.0750	0.2200	0.0740	1798.0	0.0320	0.0024	0.0026	4694.4	15180.0	3133.6	9620.0
10	0.0750	0.2200	0.0740	2274.7	0.0320	0.0026	0.0028	4464.8	15180.0	3387.2	9620.0
11	0.1250	0.1760	0.1180	2225.1	0.0320	0.0027	0.0027	4108.5	12144.0	3687.2	15340.0
12	0.1250	0.1760	0.1180	767.4	0.013	0.0029	0.0029	3719.5	12144.0	4348.4	15340.0
13	0.1250	0.1760	0.1180	680.6	0.013	0.0032	0.0032	3479.6	12144.0	4522.7	15340.0
14	0.1250	0.1760	0.1180	922.6	0.015	0.0035	0.0035	3305.7	12144.0	4715.6	15340.0
15	0.1250	0.1760	0.1180	1510.6	0.027	0.0037	0.0039	2801.4	12144.0	5020.6	21060.0
16	0.1250	0.1320	0.1620	8555.2	0.014	0.0040	0.0041	2511.4	9108.0	5216.6	21060.0
17	0.1250	0.1320	0.1620	545.4	0.009	0.0043	0.0044	2343.3	9108.0	5484.7	21060.0
18	0.1250	0.1320	0.1620	519.8	0.009	0.0046	0.0046	2215.4	9108.0	5612.6	21060.0
19	0.1250	0.1320	0.1620	765.0	0.013	0.0049	0.0049	2061.2	9108.0	5766.8	21060.0
20	0.1250	0.1320	0.1620	1428.6	0.023	0.0051	0.0052	1800.3	6072.0	6027.7	26780.0
21	0.1250	0.0480	0.2660	674.8	0.011	0.0054	0.0054	1553.0	6072.0	6275.0	26780.0
22	0.1250	0.0480	0.2660	674.8	0.011	0.0056	0.0057	1425.3	6072.0	6402.7	26780.0
23	0.1250	0.0480	0.2660	601.3	0.007	0.0059	0.0059	1330.0	6072.0	6498.0	26780.0
24	0.1250	0.0480	0.2660	675.4	0.011	0.0061	0.0062	1202.2	6072.0	6625.8	26780.0
25	0.1250	0.0480	0.2660	1410.0	0.024	0.0063	0.0064	955.7	3036.0	6733.3	32500.0
26	0.1250	0.0440	0.2500	513.0	0.029	0.0065	0.0066	735.2	3036.0	7092.8	42500.0
27	0.1250	0.0440	0.2500	271.4	0.006	0.0066	0.0069	645.7	3036.0	7182.3	32500.0
28	0.1250	0.0440	0.2500	372.6	0.016	0.0071	0.0071	572.3	3036.0	7255.7	32500.0
29	0.1250	0.0440	0.2500	359.1	0.016	0.0072	0.0073	421.3	3036.0	7425.7	32500.0
30	0.1250	0.0440	0.2500	2739.5	0.046	0.0073	0.0075	0.0	7828.0	7828.0	32500.0
31	*****	0.0	0.2500	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

ELASTIC JOINT STRENGTH, PLOAD (LBS) = 6764.2  
 ALLOWABLE ADHESIVE SHEAR STRESS, TAUMAX (PSI) = 9000.0  
 $N = 1.0$   $N = +1$  FOR TENSILE SHEAR AND -1 FOR COMPRESSIVE  
 TEMPERATURE DIFFERENTIAL (DEG F) = C.0

N	STEPL	THICKO	THICKI	TAU	GAMMA	DELTA0	DELTA1	TOUTER	STROTR	TINNER	STRINR
1	0.0750	0.2640	0.0300	9000.0	0.745	0.0	0.0002	6764.2	18216.0	0.0	3900.0
2	0.0750	0.2640	0.0300	3133.7	0.016	0.0002	0.0003	5932.4	18216.0	1131.1	3900.0
3	0.0750	0.2640	0.0300	1232.3	0.006	0.0004	0.0005	5633.1	18216.0	1276.5	3900.0
4	0.0750	0.2640	0.0300	889.0	0.004	0.0006	0.0006	5487.7	18216.0	1452.3	3900.0
5	0.0750	0.2640	0.0300	1673.5	0.008	0.0008	0.0009	5311.9	18216.0	1880.6	9620.0
6	0.0750	0.2200	0.0740	4574.4	0.023	0.0010	0.0011	4883.6	15180.0	2360.4	9620.0
7	0.0750	0.2200	0.0740	2150.4	0.011	0.0012	0.0012	4463.8	15180.0	2598.9	9620.0
8	0.0750	0.2200	0.0740	1016.1	0.006	0.0014	0.0014	4008.2	15180.0	2756.0	9620.0
9	0.0750	0.2200	0.0740	1524.2	0.009	0.0017	0.0018	3827.1	15180.0	2937.1	9620.0
10	0.0750	0.2200	0.0740	1524.2	0.009	0.0017	0.0019	3500.9	12144.0	3263.3	15340.0
11	0.1250	1.761	0.1180	3053.7	0.015	0.0022	0.0022	3053.7	12144.0	3701.6	15340.0
12	0.1250	1.761	0.1180	3053.7	0.015	0.0024	0.0024	3053.7	12144.0	3835.2	15340.0
13	0.1250	1.761	0.1180	3053.7	0.015	0.0024	0.0027	2862.6	12144.0	3899.4	15340.0
14	0.1250	1.761	0.1180	268.9	0.001	0.0027	0.0030	2862.8	12144.0	4007.2	15340.0
15	0.1250	1.761	0.1180	77.1	0.004	0.0029	0.0034	2415.9	12144.0	4348.3	21060.0
16	0.1250	1.320	0.1620	2370.6	0.012	0.0030	0.0034	2081.8	9108.0	4682.4	21060.0
17	0.1250	1.320	0.1620	663.3	0.003	0.0034	0.0034	1984.2	9108.0	4780.0	21060.0
18	0.1250	1.320	0.1620	223.0	0.001	0.0037	0.0037	1936.7	9108.0	4827.5	21060.0
19	0.1250	1.320	0.1620	204.9	0.001	0.0039	0.0039	1936.7	9108.0	4916.0	21060.0
20	0.1250	1.320	0.1620	594.0	0.003	0.0041	0.0041	1848.3	9108.0	5214.2	21060.0
21	0.1250	0.880	0.2660	2114.0	0.011	0.0043	0.0044	1550.0	6072.0	5491.8	26780.0
22	0.1250	0.880	0.2660	492.8	0.002	0.0046	0.0046	1272.4	6072.0	5559.1	2678

### A.5 Computer Program A4EG For Elastic-Plastic Strength of Stepped-Lap Bonded Joints

The elastic-plastic strength of stepped-lap joints is covered by the analysis in Section 6. The digital computer program A4EG has been prepared as a design tool for the analysis of such joints. By printing out detailed internal stresses, the program can serve to aid in design improvement by changing the joint proportions in such a manner as to reduce the load transfer in the more critical regions and to increase it in those less severely loaded areas.

In addition to those features of the elastic solution A4EF, this elastic-plastic program A4EG seeks the existence and extent of any plastic adhesive zones within any step or step portion. The convergence of the nested iterative do loops is complicated by the addition of an extra loop accounting for the maximum adhesive shear strain. This is only rarely a known quantity for ductile adhesives because the end step of the stiffer adherend is usually the most critical detail.

A complete listing of the program A4EG follows. Precisely the same input data is used as for program A4EF and the output format is the same except inasmuch as A4EG prints out separate elastic and elastic-plastic solutions.

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C DECK A4FG
C STEPPED-LAP ADHESIVE-BONDED-JOINTS
C ELASTIC-PLASTIC SOLUTIONS
C JOINT ANALYSIS PROGRAM
C PROGRAM CAN BE USED TO OPTIMIZE JOINT DESIGN PROPORTIONS
C SOLUTION EXAMINES ADHESIVE SHEAR STRESS AND ADHEREND NORMAL (AXIAL)
C 1 STRESS BUT DOWNSIZES CONSIDERATION OF ADHESIVE PEEL STRESS ON THE
C 2 GROUND THAT OUTER END STEP IS USUALLY SUFFICIENTLY THIN FOR
C 3 PEEL STRESS PROBLEMS NOT TO ARISE
C NOTE THAT CONVERGENCE PROBLEM IS ACUTE FOR STEPPED-LAP JOINTS, EVEN
C 1 WITH DOUBLE-PRECISION. STEPS TAKEN HERE TO CONSTRAIN TENDENCY
C 2 TO DIVERGE (BY FREEZING SOLUTION ONE STEP AT A TIME) HAVE BEEN
C 3 ADOPTED AFTER TRYING BOTH MORE AND LESS STRINGENT TECHNIQUES
C NOTE ALSO THAT CONVERGENCE DIFFICULTIES ARE PROBLEM DEPENDENT, BEING
C 1 MORE SEVERE FOR BRITTLE (HIGH MODULUS) ADHESIVES. LOW MODULUS
C 2 ADHESIVES PROVED AMENABLE TO A CONVERGENT SOLUTION IN ONLY A
C 3 SINGLE PASS STRAIGHT THROUGH THE JOINT FROM END TO END, IN
C 4 SINGLE-PRECISION, WITH ONLY A SMALL LOSS OF ACCURACY IN LATER
C 5 STEPS.
C THE UNDERLYING DIFFICULTY IS ONE OF NUMERICAL ACCURACY LOSS IN THE
C 1 PRESENCE OF EXTREMELY HIGH ADHESIVE SHEAR STRESS GRADIENTS AT
C 2 BOTH ENDS OF EACH OF THE OUTER STEPS.
C PROGRAM HAS BEEN ADAPTED TO RUN ON CDC COMPUTERS IN SINGLE PRECISION
C 1 BUT ONLY WORKS INTERMITTENTLY IN SINGLE PRECISION ON IBM MACHINES
C 1 DIMENSION TOUTFR(150), TINNER(150), GAMMA(150), TAU(150),
C 1 DELTAO(150), DELTAI(150), STEPL(50), THICKO(50), THICKI(50),
C 2 ETOTR(50), ETINR(50), STRDTR(50), STRINR(50), STEP(150),
C 3 THCKNO(150), THCKNI(150), ETOUTR(150), ETINNP(150), STRGTR(150),
C 4 STRGNR(150)
C 5 DOUBLE PRECISION TOUTFR, TINNER, GAMMA, TAU, DELTAO, DELTAI,
C 1 TMAX, TMIN, A, B, C, D, F, ALAMDA, DELT, DELDLT, C1, C2, C3,
C 2 C4, C5, C7, C10, V, V4, V5, V6, V7, V8, V9, V10, STEP, TCHCK,
C 3 THICKNO, THCKNI, ETOUTR, ETINR, STRDTR, STRINR, STEP, THCKO,
C 4 THICKI, ETOTR, ETINR, STRDTR, STRINR, TAUUPR, TAUULR, GAMUUPR,
C 5 GAMLWR, ELSTR, XP, EL, ELMAX, ELMIN
C READ (5,10) M
C 10 FORMAT (I2)
C M = EQ. NUMBER OF JOINT CONFIGURATIONS TO BE SOLVED
C READ IN MATERIAL PROPERTIES
C READ 820 MCOUNT = 1, M
C NRVRS = 0
C JFLAG = 1
C JFLAG IDENTIFIES END OF JOINT FROM WHICH ANALYSIS COMMENCES
C READ (5,20) TAUMAX, G, GAMMAX, GAMMAE, ETA, ALPHAO, ALPHAT,
C 1 DELTM, SGND, ANSTEP
C 20 FORMAT (8F10.3)
C NSTEPS = ANSTEP
C MSTEPS = NSTEPS + 1
C LMAX = 3 * NSTEPS
C MCHECK = LMAX + 1
C READ IN JOINT GEOMETRY
C READ (5,30) (THICKO(N), THICKI(N), STEPL(N), ETOTR(N), ETINR(N),
C 1 STRDTR(N), STRINR(N)), N = 1, MSTEPS
C 30 FORMAT (7F10.3)
C CHECK ON CONSISTENCY OF ADHESIVE DATA
C VCHECK = G * GAMMAE
C R = TAUMAX / VCHECK
C IF ((1.001 .LT. R) .OR. (0.999 .GT. R)) GO TO 820
C IF (GAMMAX .GE. GAMMAE) GO TO 40
C IF NOT, REDUCE PEAK SHEAR STRESS TO LESS THAN MAXIMUM ELASTIC VALUE
C TAUMAX = G * GAMMAX
C GAMMAE = GAMMAX
C SUM LAP LENGTHS
C 40 OLAP = STEPL(1)
C DO 50 N = 2, NSTEPS
C 50 OLAP = OLAP + STEPL(N)
C A CHECK ON THE CONSTANCY OF THE TOTAL THICKNESS OF THE STEPPED-LAP JOINT
C 1 NT ADHERENDS IS NOT PROVIDED BECAUSE STRONGER JOINTS ARE OBTAINED
C 2 BY MATCHING THE ADHEREND EXTENSIONAL STIFFNESSES AT THE ENDS OF
C 1 THE JOINT AND MAINTAINING THIS TOTAL APPROXIMATELY CONSTANT THROU
C 1 GHOUT THE LENGTH OF THE JOINT. THE OMISSION OF A CHECK ON THE
C 1 ADHEREND THICKNESSES MAKES THE PROGRAM MORE VERSATILE.
C SET UP RECURRING CONSTANTS
C 40 C1 = G / ETA
C 41 C2 = 2. * C1
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND.
C 1 IF BONDED ON ONE SIDE ONLY, REDUCE TO 1.
C 42 C7 = -1. * GAMMAE
C 43 C10 = -1. * TAUMAX
C 60 C3 = ALPHAO * DELTM
C 44 C4 = ALPHAT * DELTM
C 45 C5 = C4 - C3
C PREPARE FOR SEPARATE COMPUTATION OF POTENTIAL ADHESIVE SHEAR STRENGTH
C 1 ON ASSUMPTION OF ACTUAL ADHEREND STIFFNESSES & INFINITE STRENGTH
C KADHSV = 0
C PRINT OUT INPUT DATA

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70 WRITE (6,80) ALPHAO, ALPHAT, DELTMRP A4FG0890
80 FORMAT (1H1/, 5(1H0/), 11H INPUT DATA//, 10H ALPHAO = F10.7, A4EG0900
1 13H (PER DEG. F1), 3X, 9HALPHAI = F10.7, 13H (PER DEG. F1), A4EG0910
2 3X, 9HDELTMRP = , F6.1, 9H (DEG. F)/, A4EG0920
3 1H, 5X, 1HN, 2X, 5HSTEPL, IX, 6HTHICKO, 1X, 6HTHICKI, 4X, A4EG0930
4 6HSTR0TR, 5X, 6HSTRINR, 5X, SHETOTR, 5X, SHETINR//) A4EG0940
DO 100 N = 1, NSTEPS A4EG0950
1 WRITE (6,90) N, STEPL(N), THICKO(N), THICKI(N), STROTR(N), A4EG0960
1 STRINR(N), ETOTR(N), ETINR(N) A4EG0970
90 FORMAT (1H , 4X, 12, 3(1X, F6.4), 2(1X, F10.1), 2(1X, F10.1)) A4EG0980
100 CONTINUE A4EG0990
1 IF (KADHSV .EQ. 1) GO TO 410 A4EG1000
C A4EG1010
C START WITH ELASTIC ANALYSIS A4EG1020
C PROCEED TO ELASTIC-PLASTIC ANALYSIS ONLY IF ADHESIVE IS MORE CRITICAL A4EG1030
1 THAN ADHEREND(S) A4EG1040
C NEED ELASTIC SOLUTION TO IDENTIFY CRITICAL END FOR ELASTIC-PLASTIC A4EG1050
1 SOLUTION WHEN BOTH THERMAL AND STIFFNESS ADHEREND MISMATCHES A4EG1060
2 ARE PRESENT A4EG1070
C ESTIMATE MAXIMUM POSSIBLE BOND CAPACITY FOR FULLY-PLASTIC ADHESIVE A4EG1080
PROND = TAUMAX * OLAP * 2. A4EG1090
C NOTE FACTOR 2. INCLUDED FOR DOUBLE-SIDED JOINT A4EG1100
C REDUCE TO 1. IF JOINT HAS ONLY ONE SIDE BONDED A4EG1110
C PERFECTLY-PLASTIC BOND CAPACITY WILL BE CLOSER TO ASYMPTOTE OF SCARF A4EG1120
1 JOINT SOLUTION AND IS SIGNIFICANTLY LOWER THAN PLASTIC ESTIMATE A4EG1130
SCARF JOINT STRENGTH ESTIMATE WOULD BE THE LESSER OF PROND = 2.*TAUMAX*A4EG1140
1 *OLAP*(F1T1)/(E2T2) AND PROND = 2.*TAUMAX*OLAP*(E2T2)/(F1T1) A4EG1150
C NOTE, HOWEVER, THAT STEPPED-LAP JOINTS EXHIBIT CHARACTERISTICS OF A4EG1160
1 DOUBLE-LAP JOINTS TO THE EXTENT THAT THE LOAD TRANSFERRED ON ANY A4EG1170
2 ONE STEP IS INDEPENDENT OF THAT STEP LENGTH ONCE THE LENGTH A4EG1180
3 EXCEEDS A TRANSITIONAL VALUE. LIKEWISE, THE TOTAL LOAD TRANSFER A4EG1190
4 BECOMES INDEPENDENT OF EACH AND EVERY (LONG) STEP IN THE JOINT A4EG1200
ACTUAL LOAD CAPACITY MAY BE SIGNIFICANTLY LESS IF THERMAL MISMATCH A4EG1210
1 BETWEEN ADHERENDS IS SEVERE A4EG1220
C REDUCTION IN LOAD TO ACCOUNT FOR LIMITED ADHEREND STRENGTH IS A4EG1230
1 ACCOMPLISHED LATER IN PROGRAM A4EG1240
C PROVIDE OUTER LOOP TO ADJUST ADHESIVE PEAK SHEAR STRAIN AT START OF A4EG1250
1 JOINT FOR CASES IN WHICH EITHER ADHESIVE IS MORE CRITICAL AT A4EG1260
2 OTHER END OF JOINT OR ADHERENDS ARE MORE CRITICAL THAN ADHESIVE. A4EG1270
TAUUPR = 2. * TAUMAX A4EG1280
TAULWR = 0. A4EG1290
C NOTE THAT PROGRAM IS PREVENTED FROM HANDLING PROBLEM IN WHICH SHEAR A4EG1300
1 STRESS IN ADHESIVE REVERSES SIGN, WHEN COMPUTATIONS START FROM A4EG1310
2 THE LESS CRITICAL END. SOLUTION IS OBTAINABLE FROM OTHER END. A4EG1320
C NOTE ALSO THAT, IF THE MAXIMUM SHEAR STRESS AND APPLIED LOADS HAVE A4EG1330
1 OPPOSITE SIGNS, JOINT MUST BREAK APART UNDER PESTRIAL THERMAL A4EG1340
2 STRESS ALONE WITHOUT ANY EXTERNALLY APPLIED LOAD, SO NO CASES OF A4EG1350
3 REAL CONCERN ARE EXCLUDED BY THE RESTRICTION ABOVE A4EG1360
DO 310 T = 1, 50 A4EG1370
TAU(1) = (TAUUPR + TAULWR) / 2. A4EG1380
IF (TAU(1) .GT. TAUMAX) TAU(1) = TAUMAX A4EG1390
IF (T .EQ. 1) TAU(1) = TAUMAX A4EG1400
C SET INITIAL CONDITIONS A4EG1410
GAMMA(1) = TAU(1) / G A4EG1420
TOUTER(1) = 5. * PROND A4EG1430
TINNER(1) = 0. A4EG1440
DELTAO(1) = 0. A4EG1450
DELTAI(1) = SGNLD * GAMMA(1) * ETA A4EG1460
TMAX = 10. * PROND A4EG1470
TMIN = 0. A4EG1480
C OPERATE ON THE LOAD LEVEL IN INTERMEDIATE LOOP A4EG1490
C LEAVE ADJUSTMENT OF TAUMAX FOR OUTER LOOP A4EG1500
TCHECK = 0. A4EG1510
DO 210 IFLAG = 1, NSTEPS A4EG1520
SCHECK = 1000000000000000. A4EG1530
DO 170 NCOUNT = 1, 100 A4EG1540
C CONVERGENCE NEARLY ALWAYS OCCURRED BETWEEN 20 AND 30 CYCLES IN TEST A4EG1550
1 CASES, BUT THERE WERE SOME EXCEPTIONS A4EG1560
C INTERMEDIATE LOOP ADJUSTS LOAD LEVEL A4EG1570
V10 = -1. * TOUTER(1) A4EG1580
C CHECK ON CONVERGENCE OF TOUTER(IFLAG) A4EG1590
R = TOUTER(IFLAG) / SCHECK A4EG1600
IF (|1.000000001 .GT. R| .AND. |0.99999999 .LT. R|) GO TO 180 A4EG1610
DO 120 N = IFLAG, NSTEPS A4EG1620
C INNER LOOP COMPUTES ELASTIC JOINT STRENGTH A4EG1630
A = TAU(N) A4EG1640
ALAMDA = DSORT(1C2 * (1. / ETINR(N) + 1. / ETOTR(N))) A4EG1650
B = (TINNER(N) / ETINR(N) - TOUTER(N) / ETOTR(N) + C5 * SGNLD) A4EG1660
1 * C1 / ALAMDA A4EG1670
C NOTE THAT SGNLD SIGNIFIES WHETHER SHEAR LOAD IS TENSILE OR COMPRESSIVE A4EG1680
C = STEPL(N) A4EG1690
D = ALAMDA * C A4EG1700
E = DSINHD(J) A4EG1710
F = DCOSH(D) A4EG1720
TAU(N+1) = A * F + B * E A4EG1730
DELT = (2. / ALAMDA) * (A * E + B * (F - 1.)) A4EG1740
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND. A4EG1750
1 IF BONDED ON ONE SIDE ONLY, REDUCE TO 1. A4EG1760

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TOUTER(N+1) = TOUTER(N) - DELT          A4EG1770
TINNER(N+1) = TINNER(N) + DELT          A4EG1790
IF (N .EQ. NSTEPS) GO TO 110            A4EG1790
IF (TAU(N+1) .LT. C10) GO TO 150        A4EG1800
IF (TOUTER(N+1) .LT. V10) GO TO 160        A4EG1810
C NOTE THAT THESE CONVERGENCE CHECKS ARE CRITICAL      A4EG1820
C IF THE FACTOR -1. IS TOO LARGE OR TOO SMALL, CONVERGENCE FAILS      A4EG1830
110 DELDT = (2. / (ALAMDA**2)) * (A * (F - 1.) + R * (F - D))      A4EG1840
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND.      A4EG1850
C 1. IF ROUNDED ON ONE SIDE ONLY, REDUCE TO 1.      A4EG1860
    DELTAO(N+1) = DELTAO(N) + C3 * C + SGNLD * (TOUTER(N) * C -      A4EG1870
    1 DELDLT) / ETOTR(N)      A4EG1880
    1 DELTAI(N+1) = DELTAI(N) + C4 * C + SGNLD * (TINNER(N) * C +      A4EG1890
    1 DELDLT) / ETINR(N)      A4EG1900
    GAMMA(N+1) = TAU(N+1) / G      A4EG1910
120 CONTINUE      A4EG1920
C CHECK WHETHER OR NOT PRECISELY 100 PERCENT OF LOAD HAS TRANSFERRED      A4EG1930
    R1 = TOUTER(1) / TINNER(MSTEPS)      A4EG1940
C CHECK ALSO WHETHER OR NOT CONVERGENCE HAS BEEN OBTAINED      A4EG1950
    R2 = TCHECK / TINNER(MSTEPS)      A4EG1960
    IF ((1.000001 .GT. R1) .AND. (0.999999 .LT. R1) .AND.      A4EG1970
    1 (1.000001 .GT. R2) .AND. (0.999999 .LT. R2)) GO TO 220      A4EG1980
    IF (TOUTER(1) .LT. TINNER(MSTEPS)) GO TO 130      A4EG1990
C IF SO, LOAD ESTIMATE IS TOO LOW      A4EG2000
C IF NOT, LOAD ESTIMATE IS TOO HIGH      A4EG2010
    GO TO 140      A4EG2020
C R1 IS UNSUITABLE FOR A CONVERGENCE CHECK BECAUSE NEGATIVE VALUES OF R1      A4EG2030
C 1 REPRESENT TOO HIGH A LOAD ESTIMATE, JUST LIKE THOSE VALUES IN      A4EG2040
C 2 EXCESS OF UNITY      A4EG2050
130 TMIN = TOUTER(IFLAG)      A4EG2060
    TOUTER(IFLAG) = (TMIN + TMAX) / 2.      A4EG2070
    TCHECK = TINNER(MSTEPS)      A4EG2080
    GO TO 170      A4EG2090
140 TMAX = TOUTER(IFLAG)      A4EG2100
    TOUTER(IFLAG) = (TMIN + TMAX) / 2.      A4EG2110
    TCHECK = TINNER(MSTEPS)      A4EG2120
    GO TO 170      A4EG2130
C NOTE THAT LABELS 26 AND 7 GOVERN FINE ADJUSTMENTS TO THE JOINT LOADS,      A4EG2140
C 1 WHILE LABELS 27 AND 28 REPRESENT COARSE ADJUSTMENTS      A4EG2150
150 TMAX = TOUTER(IFLAG)      A4EG2160
    SCHECK = TOUTER(IFLAG)      A4EG2170
    TOUTER(IFLAG) = (TMIN + TMAX) / 2.      A4EG2180
    GO TO 170      A4EG2190
160 TMIN = TOUTER(IFLAG)      A4EG2200
    SCHECK = TOUTER(IFLAG)      A4EG2210
C IF ADHEREND, RATHER THAN ADHESIVE, LIMITS JOINT STRENGTH, NEED TO      A4EG2220
C 1 BOOST PLOAD IN PROPORTION TO TAUMAX, EVEN IF IT MEANS EXCEEDING      A4EG2230
C 2 ADHEREND STRENGTHS IN INTERMEDIATE COMPUTATIONS. CORRECTIONS      A4EG2240
C 3 ARE APPLIED LATER      A4EG2250
    IF (TMIN .GE. TMAX) TMAX = 5. * TMAX      A4EG2260
    TOUTER(IFLAG) = (TMIN + TMAX) / 2.      A4EG2270
170 CONTINUE      A4EG2280
    IF (N .EQ. NSTEPS) GO TO 210      A4EG2290
C CONVERGENCE WILL NOT PROCEED TO FAR END OF JOINT IN SINGLE PASS      A4EG2300
    1 BECAUSE OF NUMERICAL ACCURACY PROBLEMS. REMEDY IS TO FREEZE      A4EG2310
    2 EARLIER VALUES, WHICH HAVE CONVERGED AND SLIGHTLY PERTURB      A4EG2320
    3 INTERMEDIATE VALUES, AND TO CHECK FOR CONVERGENCE AT THE FAR END      A4EG2330
    TMAX = TOUTER(1)      A4EG2340
    TMIN = -1. * TMAX      A4EG2350
    GO TO 210      A4EG2360
180 ICOUNT = IFLAG + 1      A4EG2370
    IF (TOUTER(ICOUNT) .GT. 0.) GO TO 190      A4EG2380
    IF (TOUTER(ICOUNT) .LT. 0.) GO TO 200      A4EG2390
    TMAX = TOUTER(1) / 10.      A4EG2400
    TMIN = -1. * TMAX      A4EG2410
    GO TO 210      A4EG2420
190 TMAX = 1.1 * TOUTER(ICOUNT)      A4EG2430
    TMIN = 0.9 * TOUTER(ICOUNT)      A4EG2440
    GO TO 210      A4EG2450
200 TMIN = 0.9 * TOUTER(ICOUNT)      A4EG2460
    TMIN = 1.1 * TOUTER(ICOUNT)      A4EG2470
C THE BOUNDS ABOVE ARE CRITICAL IN ENSURING CONVERGENCE      A4EG2480
C THEY MUST BE NEITHER TOO LARGE NOR TOO SMALL      A4EG2490
210 CONTINUE      A4EG2500
    NRVRS = 1      A4EG2510
C 220 IF ((C5 .GT. 0.000001) .OR. (C5 .LT. -0.000001)) GO TO 260      A4EG2520
    IF NOT, FIRST SOLUTION MAY BE SCALED IN THE ABSENCE OF ANY THERMAL      A4EG2530
    1 MISMATCH BETWEEN ADHERENDS      A4EG2540
    IF SO, SOLUTION MUST BE REFINED BY ITERATION, SINCE THERMAL STRESS      A4EG2550
    1 TERMS DO NOT SCALE LINEARLY, EVEN FOR ELASTIC ADHESIVE AND      A4EG2560
    2 ADHERENDS      A4EG2570
    A4EG2580
C APPLY SCALE FACTOR TO SOLUTION FOR ONLY ADHEREND STIFFNESS IMBALANCE      A4EG2590
C ASCERTAIN WHETHER INTERNAL LOADS ARE CRITICAL FOR ELASTIC ADHESIVE OR      A4EG2600
    1 WHETHER OTHER END OF JOINT IS MORE CRITICAL FOR ADHESIVE      A4EG2610
    PROGRAM ASSUMES ADHEREND ALLOWABLES HAVE SAME MAGNITUDE IN TENSION AS      A4EG2620
    1 IN COMPRESSION. DISTINCTION IS USUALLY UNIMPORTANT SINCE, IN      A4EG2630
    A4EG2640

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C   2 PRACTICAL JOINTS, RESIDUAL THERMAL STRESSES ARE UNLIKELY TO      A4EG2650
C   3 BREAK ADHEREND(S) RATHER THAN ADHESIVE                         A4EG2660
C     RSCALE = TOUTER(1) / STROTR(1)
C     IF (PSCALE .LT. 0.) RSCALE = -1. * PSCALE
C     RTAUMX = TAU(1) / TAUMAX
C     IF (RTAUMX .LT. 0.) RTAUMX = -1. * RTAUMX
C     DO 240 N = 2, MSTEPS
C       RINP = TINNER(N) / STRINR(N-1)
C     C NEED STRENGTH ON THIN SIDE OF STEP. HENCE THE (N-1) IN COMPARISON    A4EG2670
C     IF (RINP .LT. 0.) RINP = -1. * RTNR
C     IF (RINP .GT. RSCALE) RSCALE = RTNR
C     IF (N .EQ. MSTEPS) GO TO 230
C     ROTR = TOUTER(N) / STROTR(N)
C     IF (ROTR .LT. 0.) ROTR = -1. * ROTR
C     IF (ROTR .GT. PSCALE) PSCALE = ROTR
C   230 RTAU = TAU(N) / TAUMAX
C     IF (RTAU .LT. 0.) RTAU = -1. * RTAU
C     IF (RTAU .GT. RTAUMX) RTAUMX = RTAU
C   240 CONTINUE
C     RCTR = RSCALE
C     C RCTR IS PROPORTIONALITY CONSTANT GOVERNING ELASTIC SOLUTION
C     IF RTAUMAX .GT. RSCALE, ADHESIVE PLASTICITY CAN INCREASE STRENGTH
C     C USUALLY ADHESIVE IS CRITICAL AT ONE END OF JOINT OR OTHER, SO RTAUMX    A4EG2680
C     C 1 .GT. 1. MAY WELL JUST SIGNIFY THAT FAR END OF JOINT IS CRITICAL    A4EG2690
C     C NOTE THAT PROGRAM ASSUMES THAT ANY INTERNAL ADHEREND STRESSES OF    A4EG2700
C     C 1 REVERSED SIGN WITH RESPECT TO STRESS OUTSIDE THE JOINT ARE NOT    A4EG2710
C     C 2 CRITICAL. IF THEY ARE, IT MEANS THAT THE JOINT WILL FAIL DUE    A4EG2720
C     C 3 TO RESIDUAL THERMAL STRESSES ALONE WITHOUT ANY MECHANICAL LOADS    A4EG2730
C     IF (PSCALE .LT. RTAUMX) RCTR = RTAUMX
C     DO 250 N = 1, MSTEPS
C       TOUTER(N) = TOUTER(N) / RCTR
C       TINNER(N) = TINNER(N) / RCTR
C       TAU(N) = TAU(N) / RCTR
C       GAMMA(N) = GAMMA(N) / RCTR
C       DELTAO(N) = DELTAO(N) / RCTR
C   250 DELTAT(N) = DELTAT(N) / RCTR
C     GO TO 330
C     C USE ITERATIVE SOLUTION WHEN ADHEREND THERMAL MISMATCH IS PRESENT
C     C ASCERTAIN WHETHER INTERNAL LOADS ARE CRITICAL FOR ELASTIC ADHESIVE OR    A4EG2740
C     C 1 WHETHER OTHER END OF JOINT IS MORE CRITICAL FOR ADHESIVE.          A4EG2750
C   260 RSCALE = TOUTER(1) / STROTR(1)
C     IF (RSCALE .LT. 0.) RSCALE = -1. * RSCALE
C     RTAUMX = TAU(1) / TAUMAX
C     IF (RTAUMX .LT. 0.) RTAUMX = -1. * RTAUMX
C     DO 280 N = 2, MSTEPS
C       RINP = TINNER(N) / STRINR(N)
C       IF (RINP .GT. RSCALE) RSCALE = RTNR
C       IF (N .EQ. MSTEPS) GO TO 270
C       ROTR = TOUTER(N) / STROTR(N)
C       IF (ROTR .GT. RSCALE) RSCALE = ROTR
C   270 RTAU = TAU(N) / TAUMAX
C     IF (RTAU .LT. 0.) RTAU = -1. * RTAU
C     IF (RTAU .GT. RTAUMX) RTAUMX = RTAU
C   280 CONTINUE
C     C CHECK ON CONVERGENCE
C     R = RTAUMX
C     IF (RTAUMX .LT. RSCALE) R = RSCALE
C     IF ((1.00001 .GT. R) .AND. (0.99999 .LT. R)) GO TO 330
C     V = 2. * TAUMAX
C     R = (V + TAUUPP) / (V + TAULWR)
C     IF ((1.00001 .GT. R) .AND. (0.99999 .LT. R)) GO TO 330
C     C IF EITHER RTAUMX OR RSCALE .GT. UNITY, TAU(1) MUST BE DECREASED    A4EG3030
C     C IF ((RTAUMX .GT. 1.00001) .OR. (RSCALE .GT. 1.00001)) GO TO 290    A4EG3040
C     C IF BOTH RTAUMX AND RSCALE ARE .LT. UNITY, TAU(1) MUST BE INCREASED    A4EG3050
C     C IF ((RTAUMX .LT. 0.99999) .AND. (RSCALE .LT. 0.99999)) GO TO 300    A4EG3060
C     C IF NEITHER CHECK IS MET, SOLUTION HAS CONVERGED                  A4EG3070
C     GO TO 330
C   290 TAUUPP = TAU(1)
C     GO TO 310
C   300 TAULWR = TAU(1)
C   310 CONTINUE
C     C IF PROGRAM GOES BEYOND PRECEDING CONTINUE STATEMENT, SOLUTION HAS NOTA4EG3370
C     C 1 CONVERGED
C     WRITE (6,320)
C   320 FORMAT (1H1, 18HNOT CONVERGENT SOLUTION)
C     C PRINT OUT RESULTS OF ELASTIC COMPUTATIONS
C   330 WRITE (6,340) TOUTER(1), TAUMAX, SGNLD, DELTMD
C   340 FORMAT (1H1/ 5(1H0/))
C     1 39H ELASTIC JOINT STRENGTH, PLLOAD (LBS) = , F10.1/, A4EG3420
C     2 49H ALLOWABLE ADHESIVE SHEAR STRESS, TAUMAX (PSI) = , F8.1/, A4EG3460
C     3 1H , 8HSGNLD = , F4.1, 54H SGNLD = +1 FOR TENSILE SHEAR AND -1A4EG3470
C     4 FOR COMPRESSIVE/, 36H TEMPERATURE DIFFERENTIAL (DEG F) = , F6.1//A4EG3480
C     5, 1H , 5X, 1HN, 2X, 5HSTFPL, 1X, 6HTHICK0, 1X, 6HTHICK1, 3X, A4EG3490
C     6 3HTAU, 4X, 5HGMMA, 1X, 6HDELTAO, 1X, 6HDELTAI, 5X, 6HTOUTER, A4EG3500
C     7 5X, 6HSTROTR, 5X, 6HTINNER, 5X, 6HSTRINR//) A4EG3510
C     DO 360 N = 1, MSTEPS

```

```

      WRITE (6,350) N, STEPL(N), THICK0(N), THICK1(N), TAU(N), GAMMA(N), A4EG3530
      1 DELTA0(N), DELTA1(N), TOUTER(N), STROTR(N), TINNER(N), STRINP(N) A4EG3540
      2) A4EG3550
  350 FORMAT (1H , 4X, I2, 1X, F6.4, 1X, F6.4, 1X, F6.4, 1X, F7.1, 1X,
      1 F6.3, 1X, F6.4, 1X, F6.4, 1X, F10.1, 1X, F10.1, 1X, F10.1, 1X, A4EG3560
      2 F10.1) A4EG3570
  360 CONTINUE A4EG3580
C RECOMPUTE SOLUTION FROM OTHER END OF JOINT, IF APPROPRIATE A4EG3590
C NOTE THAT, IF COMPUTER PRINTS OUT TWO SOLUTIONS TO A GIVEN PROBLEM BY A4EG3600
C 1 REVERSING ENDS AND RE-ANALYZING, IT IS BECAUSE THE FIRST FAILED A4EG3630
C 2 TO CONVERGE, EVEN IF THE ANSWERS PRINTED SEEM TO SUGGEST A4EG3640
C 3 OTHERWISE. THE SECOND SOLUTION IS TO BE PREFERRED, PARTICULARLY A4EG3650
C 4 IF IT STARTS AT THAT END OF THE JOINT AT WHICH THE ADHESIVE A4EG3660
C 5 SHEAR STRESS IS AT ITS HIGHEST. A4EG3670
C IDENTIFY CRITICAL END OF JOINT A4EG3680
C AVOID REVERSING ENDS BACK AGAIN A4EG3690
C IF (JFLAG .EQ. 2) GO TO 400 A4EG3700
C IF (NPVRS .EQ. 1) GO TO 370 A4EG3710
C IF SO, SOLUTION HAS FAILED TO CONVERGE, SO TRY AGAIN FROM OTHER END A4EG3720
C ACCURACY AT FAR END OF JOINT MAY BE POOR IF FAR END IS CRITICAL A4EG3730
C IF ((TAU(MSTEPS) .LE. TAU(1)) .AND. (TAU(MSTEPS) .GE.
      1 (-L * TAU(1)))) GO TO 400 A4EG3740
C IF, AT FAR END OF JOINT, TAU(MSTEPS) .GT. TAU(1) AT NEAR END A4EG3750
C 1 FAILURE TO CONVERGE MAY BE SIMPLY THE RESULT OF THE FAR END A4EG3770
C 2 OF THE JOINT BEING MORE CRITICAL THAN THE STARTING (NEAR) END A4EG3780
C REVERSE DATA AND REANALYZE A4EG3790
  370 DO 380 N = 1, MSTEPS A4EG3800
      STEP(N) = STEPL(N) A4EG3810
      THCK0(N) = THICK0(N) A4EG3820
      THCK1(N) = THICK1(N) A4EG3830
      FTOUTR(N) = FTOTR(N) A4EG3840
      ETINNR(N) = ETINR(N) A4EG3850
      STRGTR(N) = STROTR(N) A4EG3860
  380 STRGNR(N) = STRTNR(N) A4EG3870
      DO 390 N = 1, MSTEPS A4EG3880
      STEPL(N) = STEP(MSTEPS - N) A4EG3890
      THCK0(N) = THCKNI(MSTEPS - N) A4EG3900
      THICK1(N) = THCKNO(MSTEPS - N) A4EG3910
      FTOTR(N) = ETINNR(MSTEPS - N) A4EG3920
      ETINR(N) = FTOUTR(MSTEPS - N) A4EG3930
      STROTR(N) = STRGTR(MSTEPS - N) A4EG3940
  390 STRINR(N) = STRGNR(MSTEPS - N) A4EG3950
      STEPL(MSTEPS) = STEP(MSTEPS) A4EG3960
      THCK0(MSTEPS) = 0. A4EG3970
      THICK1(MSTEPS) = THCKNO(1) A4EG3980
      FTOTR(MSTEPS) = 0. A4EG3990
      ETINR(MSTEPS) = ETINR(1) A4EG4000
      STROTR(MSTEPS) = 0. A4EG4010
      STRINP(MSTEPS) = STRGTP(1) A4EG4020
      V = ALPHAN A4EG4030
      ALPHAN = ALPHAT A4EG4040
      ALPHAT = V A4EG4050
      JFLAG = 2 A4EG4060
      NPVRS = 0 A4EG4070
      GO TO 60 A4EG4080
      GO TO 60 A4EG4090
C BYPASS ELASTIC-PLASTIC COMPUTATIONS IF ADHERENDS ARE MORE CRITICAL A4EG4100
C 1 THAN ADHESIVE A4EG4110
  400 IF (PSCALE .GE. RTAUMX1) GO TO 820 A4EG4120
C RECORD ELASTIC JOINT STRENGTH A4EG4130
      ELSTR = TOUTER(1) A4EG4140
      ELSTR = TOUTER(1) A4EG4150
C START ELASTIC-PLASTIC SOLUTION A4EG4160
C ELASTIC SOLUTION HAS IDENTIFIED CRITICAL END OF JOINT, AND REVERSED A4EG4170
C 1 ORDER OF DATA IF NECESSARY, SO THERE IS NO NEED FOR SUCH A4EG4180
C 2 CAPABILITY IN THE ELASTIC-PLASTIC SOLUTION A4EG4190
C ADD EXTRA LOCATIONS INSIDE STEPS TO ACCOUNT FOR POTENTIAL PLASTIC-TO- A4EG4200
C 1 PLASTIC AND ELASTIC-TO-PLASTIC TRANSITIONS IN ADHESIVE A4EG4210
  410 DO 420 N = 1, MSTEPS A4EG4220
      L = 3 * N A4EG4230
      THCKNO(L-2) = THICK0(N) A4EG4240
      THCKNO(L-1) = THICK0(N) A4EG4250
      THCKNO(L) = THICK0(N) A4EG4260
      THCKNI(L-2) = THICK1(N) A4EG4270
      THCKNI(L-1) = THICK1(N) A4EG4280
      THCKNI(L) = THICK1(N) A4EG4290
      FTOUTR(L-2) = FTOTR(N) A4EG4300
      FTOUTR(L-1) = FTOTR(N) A4EG4310
      FTOUTR(L) = FTOTR(N) A4EG4320
      ETINNR(L-2) = ETINR(N) A4EG4330
      ETINNR(L-1) = ETINR(N) A4EG4340
      ETINNR(L) = ETINR(N) A4EG4350
      STRGTP(L-2) = STROTR(N) A4EG4360
      STRGTP(L-1) = STROTR(N) A4EG4370
      STRGTP(L) = STROTR(N) A4EG4380
      STRGNP(L-2) = STRINR(N) A4EG4390
      STRGNP(L-1) = STRINR(N) A4EG4400

```

```

420 STRGNP(L) = STRINR(N)
THCKNO(MCHECK) = 0.
THCKNT(MCHECK) = THICKT(MSTEPS)
ETOUTR(MCHECK) = 0.
FTINNP(MCHECK) = FTINR(MSTEPS)
STRGTR(MCHECK) = 0.
STRGNR(MCHECK) = STRINP(MSTEPS)
A4EG4410
A4EG4420
A4EG4430
A4EG4440
A4EG4450
A4EG4460
A4EG4470
A4EG4480
A4EG4490
A4EG4500
A4EG4510
A4EG4520
A4EG4530
A4EG4540
A4EG4550
A4EG4560
A4EG4570
A4EG4580
A4EG4590
A4EG4600
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C USE OUTER LOOP TO ADJUST MAXIMUM ADHESIVE SHEAR STRAIN LEVEL
GAMUPR = GAMMAX
GAMLWR = GAMMAE
A4EG4490
A4EG4500
A4EG4510
A4EG4520
A4EG4530
A4EG4540
A4EG4550
A4EG4560
A4EG4570
A4EG4580
A4EG4590
A4EG4600
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C GAMUPR AND GAMLWR SERVE AS BOUNDS ON SHEAR STRAIN ACTUALLY DEVELOPED
C SET UPPER AND LOWER BOUNDS ON STRENGTH
TUPPER = TAUMAX * OLAP * 2.
A4EG4510
A4EG4520
A4EG4530
A4EG4540
A4EG4550
A4EG4560
A4EG4570
A4EG4580
A4EG4590
A4EG4600
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C NOTE THAT THE FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF
C 1 ADHERENDS. IF BONDED ON ONE SIDE ONLY, REDUCE TO 1.
TLOWER = FLSTR
A4EG4510
A4EG4520
A4EG4530
A4EG4540
A4EG4550
A4EG4560
A4EG4570
A4EG4580
A4EG4590
A4EG4600
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C TMAX * F0. FULLY-PLASTIC JOINT STRENGTH
C TMIN * F0. PERFECTLY-ELASTIC JOINT STRENGTH
ICHECK = 0
JCHECK = 10
A4EG4510
A4EG4520
A4EG4530
A4EG4540
A4EG4550
A4EG4560
A4EG4570
A4EG4580
A4EG4590
A4EG4600
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C NEED TO SET DIFFERENT VALUES FOR ICHECK AND JCHECK WITH NEITHER
C 1 EQUAL TO EITHER 1 OR 2
DO 760 I = 1, 50
IF ( (KADHSV .EQ. 1) .AND. (I .GT. 1) ) GO TO 770
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C THIS INSTRUCTION PRINTS OUT SOLUTION FOR POTENTIAL BOND SHEAR STRENGTH
TMAX = TUPPER
TMIN = TLOWER
GAMMA(1) = (GAMUUPR + GAMLWR) / 2.
IF ( (KADHSV .EQ. 1) .OR. (I .EQ. 1) ) GO TO 440
IF ( (ICHECK .NE. JCHECK) GO TO 450
IF ( (ICHECK .EQ. 1) GO TO 430
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C IF NOT, ICHECK = F0. 2 AND LOAD HAS BEEN TOO HIGH FOR TWO CONSECUTIVE
C 1 ITERATIONS
GAMMA(1) = (GAMMA(1) + GAMLWR) / 2.
ICHECK = 2
GO TO 450
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C IF4 IF CHECK = F0. 1, LOAD HAS BEEN TOO LOW FOR TWO CONSECUTIVE ITERATIONS
430 GAMMA(I) = (GAMMA (1) + GAMUUPR) / 2.
ICHECK = 1
GO TO 450
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C S4 IF CHECK = F0. 1, LOAD HAS BEEN TOO LOW FOR TWO CONSECUTIVE ITERATIONS
440 GAMMA(1) = GAMMAX
S4 SET INITIAL CONDITIONS
450 TOUTER(1) = (TUPPER + TLOWER) / 2.
TINNFR(1) = 0.
TAU(1) = TAUMAX
DELTAN(1) = 0.
DELTAI(1) = SGNL0 * GAMMA(1) * ETA
DO 700 IFLAG = 1, NSTEPS
J1 = 3 * IFLAG
J2 = J1 + 3
SCHECK = 1000000000000000.
DO 660 NCOUNT = 1, 50
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C USUALLY 20 CYCLES OF ITERATION WERE SUFFICIENT AT THIS POINT
C MIDDLE LOOP ADJUSTS LOAD LEVEL
C CHECK ON CONVERGENCE OF TOUTER(J1)
P = TOUTER(J1) / SCHECK
IF ( (1.000000001 .GT. P) .AND. (0.99999999 .LT. P) ) GO TO 670
V11 = TOUTER(1)
V10 = -1. * V11
DO 600 N = IFLAG, NSTEPS
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C INNERMOST LOOP COMPUTES JOINT STRENGTH
L = 3 * N - 2
STEP(L) = STEP1(N)
STEP(L+1) = 0.
STEP(L+2) = 0.
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C IF ADHESIVE IS NOT LOADED INTO PL4 PLASTIC ZONE IN LATER STEPS OF JOINT,
C BYPASS SUCH COMPUTATIONS AND PROCEED TO PERFECTLY-ELASTIC SOLUTION
VR = GAMMA(L)
IF ( (VR .LE. GAMMAE) .AND. (VR .GE. C7) ) GO TO 510
XP = 100000000000.
MFLAG = 0
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C SOLVE FOR MAXIMUM POSSIBLE EXTENT OF PLASTIC ADHESIVE ZONE
C 1 AND COMPARE WITH STEP LENGTH
V4 = ETOTR(N)
V5 = ETINR(N)
V6 = TOUTER(L)
V7 = TINNFR(L)
V9 = STEP(L)
V = (1. / V4 + 1. / V5) / ETA
A = TAUMAX * V
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C IF BONDED ON ONE SIDE OF JOINT ONLY, DIVIDE A BY 2.
B = (C5 * SGNL0 - V6 / V4 + V7 / V5) / ETA
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C NOTE THAT B SHOULD BE NEGATIVE AT AND NEAR MORE CRITICAL END OF JOINT
C IF NOT, SOLUTION IS PROCEEDING FROM WRONG END OF JOINT
C HENCE NEED FOR PRIOR ELASTIC SOLUTION TO IDENTIFY CRITICAL END
IF (V8 .LT. 0.) GO TO 460
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

C PROCEDURE FOR POSITIVE PLASTIC ADHESIVE SHEAR STRAINS
IF (B .GE. 0.) GO TO 470
A4EG4610
A4EG4620
A4EG4630
A4EG4640
A4EG4650
A4EG4660
A4EG4670
A4EG4680
A4EG4690
A4EG4700
A4EG4710
A4EG4720
A4EG4730
A4EG4740
A4EG4750
A4EG4760
A4EG4770
A4EG4780
A4EG4790
A4EG4800
A4EG4810
A4EG4820
A4EG4830
A4EG4840
A4EG4850
A4EG4860
A4EG4870
A4EG4880
A4EG4890
A4EG4900
A4EG4910
A4EG4920
A4EG4930
A4EG4940
A4EG4950
A4EG4960
A4EG4970
A4EG4980
A4EG4990
A4EG5000
A4EG5010
A4EG5020
A4EG5030
A4EG5040
A4EG5050
A4EG5060
A4EG5070
A4EG5080
A4EG5090
A4EG5100
A4EG5110
A4EG5120
A4EG5130
A4EG5140
A4EG5150
A4EG5160
A4EG5170
A4EG5180
A4EG5190
A4EG5200
A4EG5210
A4EG5220
A4EG5230
A4EG5240
A4EG5250
A4EG5260
A4EG5270
A4EG5280

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C = VR - GAMMAE          A4EG5290
D = R**2 - 4. * A * C   A4EG5300
IF (D .LT. 0.) GO TO 470  A4EG5310
C IF SO, PLASTIC ZONE IS UNBOUNDED, AS AT FAR END OF JOINT
XP = (-1. * B - DSORT(D)) / (2. * A)  A4EG5320
GO TO 470  A4EG5330
A4EG5340
A4EG5350
A4EG5360
A4EG5370
A4EG5380
A4EG5390
A4EG5400
A4EG5410
A4EG5420
A4EG5430
A4EG5440
A4EG5450
A4EG5460
A4EG5470
A4EG5480
A4EG5490
A4EG5500
A4EG5510
A4EG5520
A4EG5530
A4EG5540
A4EG5550
A4EG5560
A4EG5570
A4EG5580
A4EG5590
A4EG5600
A4EG5610
A4EG5620
A4EG5630
A4EG5640
A4EG5650
A4EG5660
A4EG5670
A4EG5680
A4EG5690
A4EG5700
A4EG5710
A4EG5720
A4EG5730
A4EG5740
A4EG5750
A4EG5760
A4EG5770
A4EG5780
A4EG5790
A4EG5800
A4EG5810
A4EG5820
A4EG5830
A4EG5840
A4EG5850
A4EG5860
A4EG5870
A4EG5880
A4EG5890
A4EG5900
A4EG5910
A4EG5920
A4EG5930
A4EG5940
A4EG5950
A4EG5960
A4EG5970
A4EG5980
A4EG5990
A4EG6000
A4EG6010
A4EG6020
A4EG6030
A4EG6040
A4EG6050
A4EG6060
A4EG6070
A4EG6080
A4EG6090
A4EG6100
A4EG6110
A4EG6120
A4EG6130
A4EG6140
A4EG6150
A4EG6160

C PROCEDURE FOR NEGATIVE PLASTIC ADHESIVE SHEAR STRAINS
460 IF (B .LE. 0.) GO TO 470
C = VR + GAMMAE
D = R**2 + 4. * A * C
IF (D .LT. 0.) GO TO 470
C IF SO, PLASTIC ZONE IS UNBOUNDED, AS AT FAR END OF JOINT
XP = (B - DSORT(D)) / (2. * A)
470 IF (XP .GE. V9) GO TO 480
C IF SO, ADHESIVE IS FULLY-PLASTIC THROUGHOUT THAT STEP
C IF NOT, BREAK UP STEP INTO PLASTIC AND ELASTIC PORTIONS
MFLAG = 1
STEP(1) = XP
STEP(L+1) = V9 - XP
V9 = XP
C MAY HAVE TO DECREASE STEP(3*N-1) AND ADD TO STEP(3*N) LATER
C PROCEDURE FOR FULLY-PLASTIC STEP OR STEP PORTION
C THIS SERIES OF EQUATIONS HOLDS REGARDLESS OF SIGN OF SHEAR STRESS
C 1. GRADIENT AT START OF STEP
480 DELT = 2. * TAU(L) * V9
C THE USE OF TAU(L) INSTEAD OF TAUMAX COVERS REVERSAL OF SIGN
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND.
C 1. IF BONDED ON ONE SIDE ONLY, REDUCE TO 1.
L = L + 1
TOUTER(L) = V6 - DELT
TINNER(L) = V7 + DELT
GAMMA(L) = GAMMA(L-1) + R * V9 + TAU(L-1) * V * (V9**2)
TAU(L) = TAU(L-1)
DELTAD(L) = DELTAD(L-1) + C3 * V9 + SGNLD * (V6 * V9 - TAU(L) *
1 (V9**2) / 2.) / V4
DELTAI(L) = DELTAI(L-1) + C4 * V9 + SGNLD * (V7 * V9 + TAU(L) *
1 (V9**2) / 2.) / V5
C NOTE THAT USE OF TAU(L) INSTEAD OF TAUMAX AUTOMATICALLY ACCOUNTS FOR
C 1. SIGN OF ADHESIVE SHEAR STRESS
IF (MFLAG .EQ. 1) GO TO 490
C IF NOT, STEP IS PLASTIC THROUGHOUT
L1 = L + 1
L2 = L + 2
TOUTER(L1) = TOUTER(L)
TOUTER(L2) = TOUTER(L)
TINNER(L1) = TINNER(L)
TINNER(L2) = TINNER(L)
TAU(L1) = TAU(L)
TAU(L2) = TAU(L)
GAMMA(L1) = GAMMA(L)
GAMMA(L2) = GAMMA(L)
DELTAD(L1) = DELTAD(L)
DELTAD(L2) = DELTAD(L)
DELTATI(L1) = DELTAI(L)
DELTATI(L2) = DELTAI(L)
490 IF (N .EQ. NSTEPS) GO TO 500
IF (TOUTER(L) .LT. V10) GO TO 620
IF (TINNER(L) .LT. V10) GO TO 610
C NOTE THAT THESE CONVERGENCE CHECKS ARE CRITICAL
500 IF (MFLAG .EQ. 1) GO TO 510
GO TO 590
C PROCEDURE FOR PERFECTLY-ELASTIC ZONE
C IDENTIFY WHETHER STEP IS ELASTIC-PLASTIC OR FULLY PLASTIC THROUGHOUT
510 K = L - 3 * N + 2
C K .EQ. 0 CORRESPONDS TO NO PLASTIC ZONE AT NEAR END OF JOINT
C SET INITIAL CONDITIONS AT START OF STEP
V4 = FTOTR(N)
V5 = FTINR(N)
V6 = TOUTER(L)
V7 = TINNER(L)
ALAMDA = DSQRT(C2 * (1. / V4 + 1. / V5))
LFLAG = 0
C COMPUTE VALUES AT FAR END OF ELASTIC ZONE
A = TAU(L)
B = (V7 / V5 - V6 / V4 + C5 * SGNLD) * C1 / ALAMDA
C NOTE THAT SGNLD SIGNIFIES WHETHER SHEAR LOAD IS TENSILE OR COMPRESSIVE
C STEP(L)
D = ALAMDA * C
E = DSINH(D)
F = DCOSH(D)
TAU(L+1) = A * F + B * E
IF ((TAU(L+1) .LE. TAUMAX) .AND. (TAU(L+1) .GE. C10) )
1 GO TO 540
C IF NOT, ELASTIC STEP SIZE IS EXCESSIVE. REDUCE BY ITERATION
ELMAX = C
ELMIN = 0.

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DO 520 LCOUNT = 1, 100
EL = (ELMAX + ELMIN) / 2.
D = ALAMDA * EL
F = DSINH(D)
E = DCOSH(D)
TCHECK = A * F + B * E
IF ((TCHECK .GT. TAUIMAX) .OR. (TCHECK .LT. C10)) ELMAX = EL
IF ((TCHECK .LT. TAUIMAX) .AND. (TCHECK .GT. C10)) ELMIN = EL
D = ELMIN / ELMAX
IF ((1.000000001 .GT. RI .AND. (0.999999999 .LT. RI)) GO TO 530
520 CONTINUE
530 STEP(L) = EL
STEP(L+1) = STEP(L+1) + C - EL
LFLAG = 1
IF (TCHECK .GT. 0.) TAU(L+1) = TAUIMAX
IF (TCHECK .LT. 0.) TAU(L+1) = C10
540 DELT = (2. / ALAMDA) * (A * F + B * (F - 1.))
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND.
C 1. IF BONDED ON ONE SIDE ONLY, REDUCE TO 1.
L = L + 1
TOUTER(L) = V6 - DELT
TINNER(L) = V7 + DELT
DELDLT = (2. / (ALAMDA**2)) * (A * (F - 1.) + B * (F - D))
C FACTOR OF 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND.
C 1. IF BONDED ON ONE SIDE ONLY, REDUCE TO 1.
DELTAD(L) = DELTAD(L-1) + C3 * EL + SGNLD * (V6 * EL - DELDLTI) / V4
DELTAT(L) = DELTAT(L-1) + C4 * EL + SGNLD * (V7 * EL + DELDLTI) / V5
GAMMA(L) = TAU(L) / G
IF (LFLAG .EQ. 1) GO TO 550
C IF NOT, THERE IS NO (SECOND) PLASTIC ZONE AT FAR END OF STEP
L1 = L + 1
TOUTER(L1) = TOUTER(L)
TINNER(L1) = TINNER(L)
TAU(L1) = TAU(L)
GAMMA(L1) = GAMMA(L)
DELTAD(L1) = DELTAD(L)
DELTAT(L1) = DELTAT(L)
IF (K .EQ. 1) GO TO 550
C IF NOT, THERE WAS NO (FIRST) PLASTIC ZONE AT NEAR END OF JOINT
L2 = L + 2
TOUTER(L2) = TOUTER(L)
TINNER(L2) = TINNER(L)
TAU(L2) = TAU(L)
GAMMA(L2) = GAMMA(L)
DELTAD(L2) = DELTAD(L)
DELTAT(L2) = DELTAT(L)
550 IF (N .EQ. NSTEPS) GO TO 560
IF ((TOUTER(L) .LT. V10) GO TO 620
IF ((TINNER(L) .LT. V10) GO TO 610
C NOTE THAT THESE CONVERGENCE CHECKS ARE CRITICAL
C IF V10 IS EITHER TOO LARGE OR TOO SMALL, CONVERGENCE FAILS
560 IF (LFLAG .EQ. 1) GO TO 570
GO TO 590
C PROCEDURE FOR (SECOND) PLASTIC ZONE AT FAR END OF STEP
570 V9 = STEP(L)
DELT = 2. * TAU(L) * V9
C FACTOR 2. ACCOUNTS FOR BONDING ON BOTH SIDES OF INNER ADHEREND.
C 1. IF BONDED ON ONE SIDE ONLY, REDUCE TO 1.
C SET INITIAL CONDITIONS AT START OF STEP
V4 = ETOTR(N)
V5 = ETINP(N)
V6 = TOUTER(L)
V7 = TINNER(L)
L = L + 1
TOUTER(L) = V6 - DELT
TINNER(L) = V7 + DELT
A = (TAU(L-1) / ETA) * (1./V4 + 1./V5)
C NOTE THE USE OF TAU(L-1) INSTEAD OF TAUIMAX IN ORDER TO ACCOUNT
C AUTOMATICALLY FOR THE SIGN OF THE SHEAR STRESS
C IF BONDED ON ONE SIDE OF JOINT ONLY, DIVIDE A BY 2.
B = (C5 * SGNLD - V6 / V4 + V7 / V5) / ETA
GAMMA(L) = GAMMA(L-1) + B * V9 + A * (V9**2)
TAU(L) = TAU(L-1)
DELTAD(L) = DELTAD(L-1) + C3 * V9 + SGNLD * (V6 * V9 - TAU(L) *
1. (V9**2) / 2.) / V4
DELTAT(L) = DELTAT(L-1) + C4 * V9 + SGNLD * (V7 * V9 + TAU(L) *
1. (V9**2) / 2.) / V5
C IF THERE HAS BEEN NO PLASTIC ZONE AT START OF STEP, TRANSFER VALUES
C 1. JUST COMPUTED ACROSS TO LAST SUBDIVISION IN STEP
C THIS IS NECESSARY TO PROVIDE INPUT DATA FOR START OF NEXT STEP
IF (K .EQ. 1) GO TO 580
L1 = L + 1
TOUTER(L1) = TOUTER(L)
TINNER(L1) = TINNER(L)
TAU(L1) = TAU(L)
GAMMA(L1) = GAMMA(L)
DELTAD(L1) = DELTAD(L)

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      DELTAT(L1) = DELTAT(L)
580  IF (N .EQ. NSTEPS) GO TO 630          A4EG7050
      IF (TOUTER(L) .LT. V101) GO TO 620      A4EG7060
      IF (TINNER(L) .LT. V101) GO TO 610      A4EG7070
C NOTE THAT THESE CONVERGENCE CHECKS ARE CRITICAL      A4EG7080
590  IF (N .EQ. NSTEPS) GO TO 630          A4EG7090
600  CONTINUE                                A4EG7100
C PROCEDURE FOR WHEN LOAD ESTIMATE IS TOO HIGH      A4EG7110
610  TMAX = TOUTER(J1)                      A4EG7120
      SCHECK = TOUTER(J1)                      A4EG7130
      TOUTER(J1) = (TMAX + TMIN) / 2.          A4EG7140
      GO TO 660                                A4EG7150
C PROCEDURE FOR WHEN LOAD ESTIMATE IS TOO LOW       A4EG7160
620  TMIN = TOUTER(J1)                      A4EG7170
      SCHECK = TOUTER(J1)                      A4EG7180
      TOUTER(J1) = (TMAX + TMIN) / 2.          A4EG7190
      GO TO 660                                A4EG7200
C
C CHECK WHETHER OR NOT PRECISELY 100 PERCENT OF LOAD HAS TRANSFERRED      A4EG7210
630  R1 = TOUTER(1) / TINNER(MCHECK)          A4EG7220
      P2 = TCHECK / TINNER(MCHECK)          A4EG7230
      IF ((1.000001 .GT. R1) .AND. (0.999999 .LT. R1)) .AND.      A4EG7240
      1 ((1.000001 .GT. P2) .AND. (0.999999 .LT. P2)) GO TO 710      A4EG7250
      IF (TOUTER(1) .LT. TINNER(MCHECK)) GO TO 640      A4EG7260
C NOTE THAT, HERE, LOAD IS TAKEN TO BE POSITIVE WHETHER TENSILE OR NOT      A4EG7270
C 1 FOR A NEGATIVE LOAD, PRECEDING INSTRUCTION SHOULD BE      A4EG7280
C 2 INTERCHANGED WITH THE FOLLOWING ONE      A4EG7290
C IF SO, LOAD ESTIMATE IS TOO LOW      A4EG7300
C IF REVERSE HOLDS, LOAD ESTIMATE IS TOO HIGH      A4EG7310
C
C PROCEDURE FOR WHEN LOAD ESTIMATE IS TOO LOW      A4EG7320
640  TMIN = TOUTER(J1)                      A4EG7330
      TOUTER(J1) = (TMAX + TMIN) / 2.          A4EG7340
      TCHECK = TINNER(MCHECK)          A4EG7350
      GO TO 660                                A4EG7360
C PROCEDURE FOR WHEN LOAD ESTIMATE IS TOO HIGH      A4EG7370
650  TMAX = TOUTER(J1)                      A4EG7380
      TOUTER(J1) = (TMAX + TMIN) / 2.          A4EG7390
      TCHECK = TINNER(MCHECK)          A4EG7400
660  CONTINUE                                A4EG7410
C CONVERGENCE WILL NOT PROCEED TO FAR END OF JOINT IN SINGLE PASS      A4EG7420
C 1 BECAUSE OF NUMERICAL ACCURACY PROBLEMS. REMEDY IS TO FREEZE      A4EG7430
C 2 EARLIER VALUES, WHICH HAVE CONVERGED, AND TO PERTURB SLIGHTLY      A4EG7440
C 3 INTERMEDIATE VALUES, AND TO CHECK FOR CONVERGENCE AT THE FAR END      A4EG7450
      TMAX = TUPPER
      TMIN = -1. * TMAX
      GO TO 700                                A4EG7460
670  IF (TOUTER(J2) .GT. 0.) GO TO 680
      IF (TOUTER(J2) .LT. 0.) GO TO 690
      TMAX = TOUTER(1) / 10.
      TMIN = -1. * TMAX
      GO TO 700                                A4EG7470
680  TMAX = 1.1 * TOUTER(J2)
      TMIN = 0.9 * TOUTER(J2)
      GO TO 700                                A4EG7480
690  TMAX = 0.9 * TOUTER(J2)
      TMIN = 1.1 * TOUTER(J2)
C THE BOUNDS ABOVE ARE CRITICAL IN ENSURING CONVERGENCE      A4EG7490
C THEY MUST BE NEITHER TOO LARGE NOR TOO SMALL      A4EG7500
700  CONTINUE                                A4EG7510
C ASCERTAIN WHETHER INTERNAL LOADS ARE CRITICAL FOR ELASTIC-PLASTIC      A4EG7520
C 1 ADHESIVE                                A4EG7530
710  PSCALE = TOUTER(1) / STRGTR(1)
      IF (PSCALE .LT. 0.) PSCALE = -1. * PSCALE
      PGAMAX = GAMMA(1) / GAMMAX
      IF (PGAMAX .LT. 0.) PGAMAX = -1. * PGAMAX
      DO 730 N = 2, MCHECK
      RINR = TINNER(N) / STRGNR(N-1)
C AT EACH STEP TRANSITION NEED THE THINNRP SECTION. HENCE THE (N-1)      A4EG7540
      IF (RINR .LT. 0.) RINR = -1. * RINR
      IF (RINR .GT. PSCALE) RSCALE = RINR
      IF (N .EQ. MCHECK) GO TO 720
      RDTR = TOUTER(N) / STRGTR(N)
      IF (RDTR .LT. 0.) RDTR = -1. * RDTR
      IF (RDTR .GT. PSCALE) PSCALE = RDTR
720  RGAMX = GAMMAINI / GAMMAX
      IF (RGAMX .LT. 0.) RGAMX = -1. * PGAMX
      IF (RGAMX .GT. RGAMAX) RGAMAX = RGAMX
      730 CONTINUE                                A4EG7550
C IF UPPER AND LOWER BOUNDS ON JOINT LOAD HAVE COALESCED,      A4EG7560
C 1 NO MORE CONVERGENCE IS POSSIBLE. PRINT OUT RESULTS.      A4EG7570
      P = TUPPER / TLLOWER
      IF ((1.000000001 .GT. P) .AND. (0.999999999 .LT. P)) GO TO 770      A4EG7580
C ADJUST MAXIMUM ADHESIVE SHEAR STRAIN IF ADHEREND STRENGTH GOVERNS OVER      A4EG7590
C 1 ADHESIVE STRENGTH CONSIDERATIONS      A4EG7600
      IF ((RSCALE .GT. 1.0001) .OR. (RGAMAX .GT. 1.0001))      A4EG7610

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1 GO TO 740 A4EG7930
1 IF (RSCALE .LT. 0.99991 .AND. (PGAMAX .LT. 0.9999) ) A4EG7940
1 GO TO 750 A4EG7950
C IF NEITHER OF THESE CHECKS IS MET, SOLUTION HAS CONVERGED. PRINT OUT. A4EG7960
1 GO TO 770 A4EG7970
C IF NOT, REITERATE A4EG7980
740 GAMUPP = GAMMA(1) A4EG7990
TUPPER = TOUTEP(1) A4EG8000
JCHECK = 2 A4EG8010
GO TO 760 A4EG8020
750 GAMLWP = GAMMA(1) A4EG8030
TLOWER = TOUTEP(1) A4EG8040
JCHECK = 1 A4EG8050
760 CONTINUE A4EG8060
C IF PROGRAM GOES PAST THIS CONTINUE STATEMENT, CONVERGENCE HAS FAILED A4EG8070
WPTTE (6,320) A4EG8080
C A4EG8090
C PRINT OUT RESULTS OF ELASTIC-PLASTIC COMPUTATIONS A4EG8100
770 WPTTE (6,780) TOUTEP(1), GAMMAX, SGNL0, DELTMD A4EG8110
780 FORMAT (1H1/, 5(1H0/), A4EG8120
1 47H ELASTIC-PLASTIC JOINT STRENGTH, PLOAD (LBS) = , F10.1/, A4EG8130
2 43H ALLOWABLE ADHESIVE SHEAR STRAIN, GAMMAX = , F6.3/, A4EG8140
3 1H , 8HSGNLD = , F4.1, 54H SGNLD = +1 FOR TENSILE SHEAR AND -1 A4EG8150
4 FOR COMPRESSIVE/, 36H TEMPERATURE DIFFERENTIAL (DEG F) = , F6.1//A4EG8160
5, 1H, 5X, 1HN, 2X, 5HSTEPL, 1X, 6HTHICK0, 1X, 6HTHICK1, 3X, A4EG8170
6 3HTAI, 4X, 5HGMMA, 1X, 6HDELTAD, 1X, 6HDELTAT, 5X, 6HTOUTER, A4EG8180
7 5X, 6HSTROTR, 5X, 6HTINNER, 5X, 6HSTRINP//) A4EG8190
DO 800 N = 1, MCHECK A4EG8200
WRITE (6,790) N, STEP(N), THCKND(N), THCKNT(N), TAU(N), GAMMA(N), A4EG8210
1 DELTAD(N), DELTAI(N), TOUTEP(N), STRGTP(N), TINNER(N), STRGNR(N) A4EG8220
21 A4EG8230
790 FORMAT (1H , 4X, I2, 1X, F6.4, 1X, F6.4, 1X, F6.4, 1X, F7.1, 1X, A4EG8240
1 F6.3, 1X, F6.4, 1X, F6.4, 1X, F10.1, 1X, F10.1, 1X, F10.1, 1X, A4EG8250
2 F10.1) A4EG8260
800 CONTINUE A4EG8270
IF (KADHSV .EQ. 1) GO TO 820 A4EG8280
IF (RSCALE .LT. PGAMAX) GO TO 820 A4EG8290
C IF NOT, COMPUTE POTENTIAL BOND STRENGTH OF ADHESIVE A4EG8300
KADHSV = 1 A4EG8310
DO 810 K = 1, MSTEP5 A4EG8320
STRNTR(K) = 10000000000000000000000000. A4EG8330
810 STRNTR(K) = 10000000000000000000000000. A4EG8340
GO TO 70 A4EG8350
820 CONTINUE A4EG8360
STOP A4EG8370
END A4EG8380

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## INPUT DATA

ALPHAO = 0.0000050 (PER DEG. F) ALPHAI = 0.0 (PER DEG. F) DELTMRP = -280.0 (DEG. F)

N	STEPL	THICKO	THICKI	STROTR	STRINR	ETOTR	ETINR
1	0.7500	0.2500	0.0440	32500.0	3036.0	4000000.0	352000.0
2	0.7500	0.2500	0.0880	26700.0	6072.0	3296000.0	704000.0
3	0.7500	0.1620	0.1320	21060.0	9108.0	2592000.0	1056000.0
4	0.7500	0.1180	0.1760	15340.0	12144.0	1888000.0	1408000.0
5	0.3750	0.0740	0.2200	9620.0	15180.0	1184000.0	1760000.0
6	0.3750	0.0300	0.2640	3900.0	18216.0	480000.0	2112000.0
7	*****	0.0	0.2640	0.0	18216.0	0.0	2112000.0

ELASTIC JOINT STRENGTH, PLOAD (LBS) = 10730.1  
 ALLOWABLE ADHESIVE SHEAR STRESS, TAUMAX (PSI) = 6000.0  
 N = -1.0 N = +1 FOR TENSILE SHEAR AND -1 FOR COMPRESSIVE  
 TEMPERATURE DIFFERENTIAL (DEG F) = -280.0

N	STEPL	THICKO	THICKI	TAU	GAMMA	DELTA0	DELTA1	TOUTR	STROTR	TINNER	STRINR
1	0.7500	0.2500	0.0440	5696.0	0.095	0.0	-0.0005	10730.1	32500.0	0.0	3076.0
2	0.7500	0.2500	0.0880	2568.5	0.043	-0.028	-0.0030	8817.0	26780.0	1913.1	6072.0
3	0.7500	0.1620	0.1320	2243.7	0.037	-0.037	-0.0059	7344.6	21060.0	3385.5	9108.0
4	0.7500	0.1180	0.1760	2239.0	0.037	-0.037	-0.0089	5804.3	15340.0	4925.8	12144.0
5	0.3750	0.0740	0.2200	2797.8	0.047	-0.017	-0.0119	4018.4	9620.0	6711.8	15180.0
6	0.3750	0.0300	0.2640	3722.2	0.062	-0.0132	-0.0135	2232.4	3900.0	8497.8	18216.0
7	*****	0.0	0.2640	6000.0	0.100	-0.0147	-0.0152	-0.0	0.0	10730.1	18216.0

## INPUT DATA

ALPHAO = 0.0 (PER DEG. F) ALPHAI = 0.0000050 (PER DEG. F) DELTMRP = -280.0 (DEG. F)

N	STEPL	THICKO	THICKI	STROTR	STRINR	ETOTR	ETINR
1	0.3750	0.2640	0.0300	19216.0	3900.0	2112000.0	480000.0
2	0.3750	0.2200	0.0740	15180.0	9620.0	1760000.0	1184000.0
3	0.7500	0.1760	0.1180	12144.0	15340.0	1408000.0	1888000.0
4	0.7500	0.1320	0.1620	9108.0	21060.0	1056000.0	2592000.0
5	0.7500	0.0880	0.2200	5072.0	26780.0	704000.0	3296000.0
6	0.7500	0.0440	0.2500	3036.0	32500.0	352000.0	4000000.0
7	*****	0.0	0.2500	0.0	32500.0	0.0	4000000.0

ELASTIC JOINT STRENGTH, PLOAD (LBS) = 10730.1  
 ALLOWABLE ADHESIVE SHEAR STRESS, TAUMAX (PSI) = 6000.0  
 N = -1.0 N = +1 FOR TENSILE SHEAR AND -1 FOR COMPRESSIVE  
 TEMPERATURE DIFFERENTIAL (DEG F) = -280.0

N	STEPL	THICKO	THICKI	TAU	GAMMA	DELTA0	DELTA1	TOUTR	STROTR	TINNER	STRINR
1	0.3750	0.2640	0.0300	6000.0	0.100	0.0	-0.0005	10730.1	19216.0	0.0	3900.0
2	0.3750	0.2200	0.0740	3722.2	0.062	-0.017	-0.0021	8497.8	15180.0	2232.4	9620.0
3	0.7500	0.1760	0.1180	2797.8	0.047	-0.037	-0.0035	6711.8	12144.0	4018.4	15340.0
4	0.7500	0.1320	0.1620	2239.0	0.037	-0.034	-0.0066	4925.9	9108.0	5804.3	12144.0
5	0.7500	0.0880	0.2200	2243.7	0.037	-0.039	-0.0095	3385.5	6072.0	7344.6	26780.0
6	0.7500	0.0440	0.2500	2568.5	0.043	-0.0122	-0.0124	1913.1	3036.0	9817.0	32500.0
7	*****	0.0	0.2500	5696.1	0.095	-0.0147	-0.0152	-0.0	0.0	10730.1	32500.0

ELASTIC-PLASTIC JOINT STRENGTH, PLOAD (LBS) = 16996.6  
 ALLOWABLE ADHESIVE SHEAR STRAIN, GAMMAX = 1.7  
 N = -1.0 N = +1 FOR TENSILE SHEAR AND -1 FOR COMPRESSIVE  
 TEMPERATURE DIFFERENTIAL (DEG F) = -280.0

N	STEPL	THICKO	THICKI	TAU	GAMMA	DELTA0	DELTA1	TOUTR	STROTR	TINNER	STRINR
1	0.1238	0.2640	0.0300	6000.0	0.218	0.0	-0.0011	16996.6	18216.0	0.0	3900.0
2	0.2357	0.2640	0.0300	6000.0	0.210	-0.0010	-0.0030	15510.9	19216.0	1485.7	3900.0
3	0.0155	0.2640	0.0300	6000.0	0.100	-0.0026	-0.0030	13282.4	18216.0	3714.2	3900.0
4	0.0185	0.2200	0.0740	6000.0	0.100	-0.0037	-0.0031	13026.3	15180.0	3900.3	9620.0
5	0.3565	0.2200	0.0740	6000.0	0.102	-0.0038	-0.0032	12874.5	15180.0	4122.1	9620.0
6	0.0	0.2200	0.0740	4616.9	0.077	-0.0077	-0.0071	10040.8	15180.0	6955.9	9620.0
7	0.7500	0.1760	0.1180	4616.9	0.077	-0.0077	-0.0071	10040.8	12144.0	6955.9	15340.0
8	0.0	0.1760	0.1180	3483.0	0.058	-0.0120	-0.0114	7168.7	12144.0	9827.9	15340.0
9	0.0	0.1760	0.1180	3483.0	0.058	-0.0120	-0.0114	7168.7	12144.0	9827.9	21160.0
10	0.7500	0.1320	0.1620	3483.0	0.058	-0.0120	-0.0114	7168.7	9108.0	9827.9	21160.0
11	0.0	0.1320	0.1620	3340.8	0.056	-0.0160	-0.0154	4824.1	9108.0	21172.6	21160.0
12	0.0	0.1320	0.1620	3340.8	0.056	-0.0160	-0.0154	4824.1	9108.0	21172.6	21160.0
13	0.7500	0.0880	0.2306	3340.8	0.056	-0.0160	-0.0154	4824.1	5072.0	21172.6	26780.0
14	0.0	0.0880	0.2306	3679.3	0.061	-0.0198	-0.0193	2676.1	6072.0	14320.5	26780.0
15	0.0	0.0880	0.2306	3679.3	0.061	-0.0198	-0.0193	2676.1	6072.0	14320.5	26780.0
16	0.7135	0.0440	0.2500	3679.3	0.061	-0.0198	-0.0193	2676.1	3036.0	14320.6	32500.0
17	0.0365	0.0440	0.2500	6000.0	0.100	-0.0233	-0.0232	438.1	3036.0	16558.6	32500.0
18	0.0	0.0440	0.2500	6000.0	0.136	-0.0233	-0.0232	-0.0	3036.0	16996.6	32500.0
19	0.0	0.0	0.2500	6000.0	0.136	-0.0233	-0.0232	-0.0	0.0	16996.6	32500.0

## INPUT DATA

ALPHAO = 0.0 (PER DEG. F) ALPHAI = 0.0000050 (PER DEG. F) DELTMR = -280.0 (DEG. F)

N STEPL THICKO THICKI STROTR STRINR ETOTR ETINR

1	0.3750	0.2640	0.0300	*****	*****	2112000.0	480000.0
2	0.3750	0.2200	0.0740	*****	*****	1760000.0	1184000.0
3	0.7500	0.1760	0.1180	*****	*****	1408000.0	1888000.0
4	0.7500	0.1320	0.1620	*****	*****	1056000.0	2592000.0
5	0.7500	0.0880	0.2160	*****	*****	704000.0	3296000.0
6	0.7500	0.0440	0.2500	*****	*****	352000.0	4000000.0
7	*****	0.0	0.2500	*****	*****	0.0	4000000.0

ELASTIC JOINT STRENGTH, PLOAD (LBS) = 10730.1  
 ALLOWABLE ADHESIVE SHEAR STRESS, TAUMAX (PSI) = 6000.0  
 $N = 1.0, N = +1$  FOR TENSILE SHEAR AND -1 FOR COMPRESSIVE  
 TEMPERATURE DIFFERENTIAL (DEG F) = -280.0

N	STEPL	THICKO	THICKI	TAU	GAMMA	DELTA0	DELTA1	TOUTR	STROTR	TINNER	STRINR
1	0.3750	0.2640	0.0300	6000.0	0.100	0.0	-0.005	10730.1	*****	0.0	*****
2	0.3750	0.2200	0.0740	3722.2	0.062	-0.017	-0.020	8497.8	*****	2232.4	*****
3	0.7500	0.1760	0.1180	2797.8	0.047	-0.033	-0.035	6712.8	*****	4618.4	*****
4	0.7500	0.1320	0.1620	2239.0	0.037	-0.064	-0.066	4923.8	*****	5804.3	*****
5	0.7500	0.0880	0.2060	2243.7	0.037	-0.093	-0.095	3385.5	*****	7344.6	*****
6	0.7500	0.0440	0.2500	2568.5	0.043	-0.122	-0.124	1913.1	*****	8817.0	*****
7	*****	0.0	0.2500	5696.1	0.095	-0.147	-0.152	-0.0	*****	10730.1	*****

ELASTIC-PLASTIC JOINT STRENGTH, PLOAD (LBS) = 30568.5  
 ALLOWABLE ADHESIVE SHEAR STRAIN, GAMMAX = 1.7  
 $N = -1.0, N = +1$  FOR TENSILE SHEAR AND -1 FOR COMPRESSIVE  
 TEMPERATURE DIFFERENTIAL (DEG F) = -280.0

N	STEPL	THICKO	THICKI	TAU	GAMMA	DELTA0	DELTA1	TOUTR	STROTR	TINNER	STRINR
1	0.3750	0.2640	0.0300	6000.0	1.700	0.0	-0.085	30568.5	*****	0.0	*****
2	0.0	0.2640	0.0300	6000.0	1.51	-0.052	-0.099	26068.5	*****	4500.0	*****
3	0.0	0.2640	0.0300	6000.0	1.51	-0.052	-0.099	26068.5	*****	4500.0	*****
4	0.3750	0.2200	0.0740	6000.0	1.51	-0.052	-0.099	26068.5	*****	4500.0	*****
5	0.0	0.2200	0.0740	6000.0	0.669	-0.0105	-0.0122	21568.5	*****	9000.0	*****
6	0.0	0.2200	0.0740	6000.0	0.669	-0.0105	-0.0122	21568.5	*****	9000.0	*****
7	0.7500	0.1760	0.1180	6000.0	0.669	-0.0105	-0.0122	21568.5	*****	9000.0	*****
8	0.0	0.1760	0.1180	6000.0	0.333	-0.0208	-0.0177	12568.5	*****	18000.0	*****
9	0.0	0.1760	0.1180	6000.0	0.333	-0.0208	-0.0177	12568.5	*****	18000.0	*****
10	0.520	0.1320	0.1620	6000.0	0.333	-0.0208	-0.0177	12568.5	*****	18000.0	*****
11	0.6980	0.1320	0.1620	6000.0	0.100	-0.0214	-0.182	11944.1	*****	18624.4	*****
12	0.0	0.1320	0.1620	5744.6	0.098	-0.0271	-0.240	7948.5	*****	22620.3	*****
13	0.0	0.7476	0.0880	5744.6	0.098	-0.0271	-0.240	7948.5	*****	22620.3	*****
14	0.0024	0.0880	0.2160	6000.0	0.100	-0.0337	-0.036	4355.5	*****	26213.1	*****
15	0.0	0.0880	0.2160	6000.0	0.102	-0.0337	-0.036	4326.7	*****	26241.9	*****
16	0.0018	0.0440	0.2500	6000.0	0.102	-0.0337	-0.036	4326.7	*****	26241.9	*****
17	0.6194	0.0440	0.2500	6000.0	0.100	-0.0337	-0.036	4305.3	*****	26263.3	*****
18	0.1288	0.0440	0.2500	6000.0	0.100	-0.0389	-0.398	1545.5	*****	29023.1	*****
19	0.0	0.0	0.2500	6000.0	0.271	-0.0393	-0.0369	0.0	0.0	30568.5	*****